

Fig 10.2. Variation of Capture Efficiencies with Velocity at $0.1 \mu\text{m}$

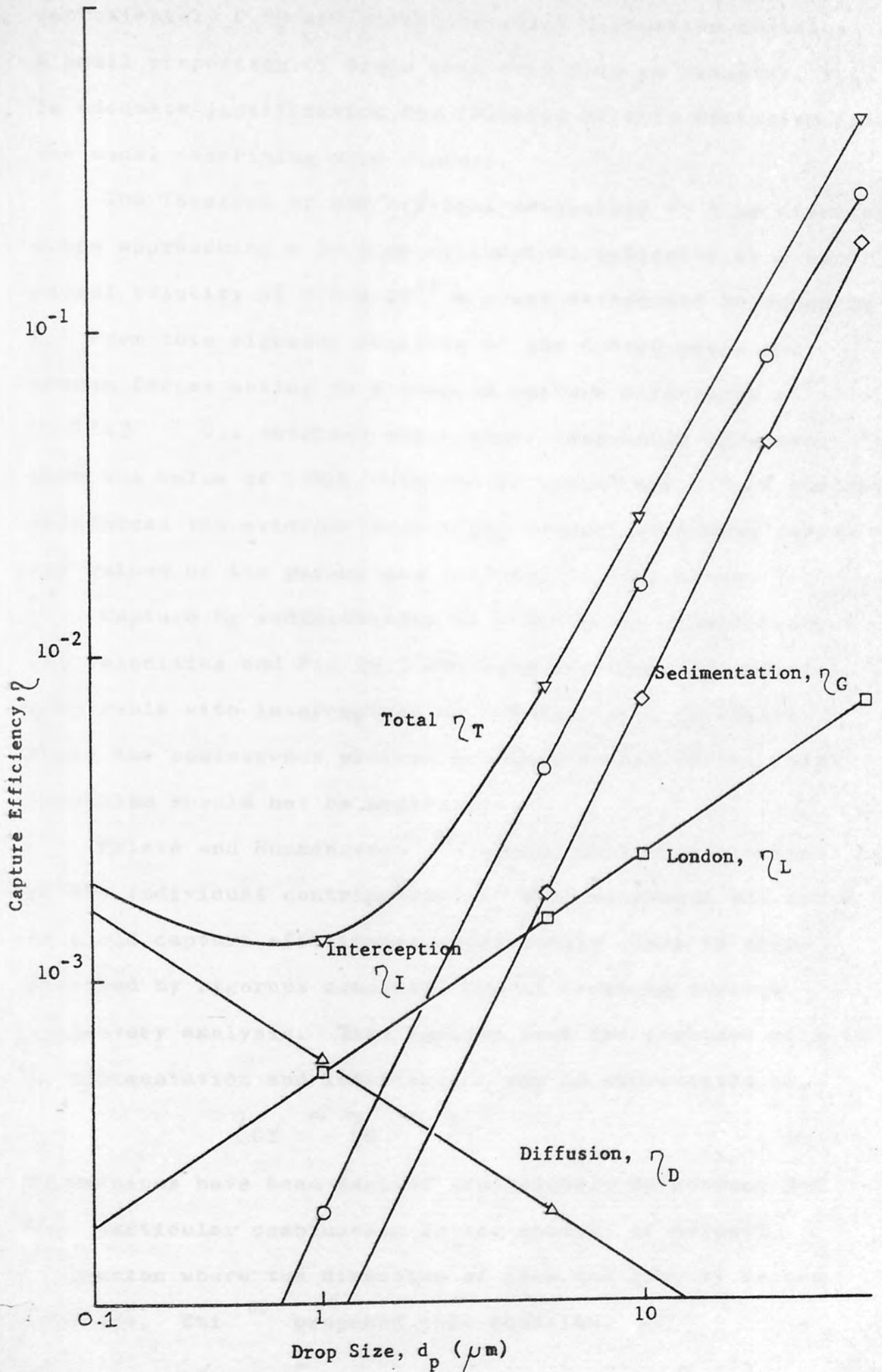


Fig 10.3. Variation of Capture Efficiencies with Drop Size at 0.1×10^{-2} m/s.

approximately 0.03 and since the inlet dispersion contains a small proportion of drops less than 5-10 μm diameter, this is adequate justification for omission of this mechanism from the model describing drop capture.

The location of the critical trajectory of 5 μm diameter drops approaching a 30.5 μm cylindrical collector at a superficial velocity of 0.5×10^{-2} m/s was determined in Appendix M. From this rigorous analysis of the hydrodynamic and London forces acting on a drop, a capture efficiency of 0.0263 was obtained which shows reasonable agreement with the value of 0.008 obtained in Appendix L. This further reinforces the evidence supporting neglect of London forces for values of the parameters relevant to this study.

Capture by sedimentation is shown to be significant at low velocities and Fig 10.3 demonstrates that its effect is comparable with interception as the drop size increases. Since the coalescence process produces larger drops, this mechanism should not be neglected.

Prieve and Ruckenstein⁷⁸ showed that superposition of the individual contributions of each mechanism was found to yield capture efficiencies sufficiently close to those obtained by rigorous consideration of coupling through trajectory analysis. This implies that the combined effects of sedimentation and interception may be represented by,

$$\eta_{GI} = \eta_G + \eta_I \quad 10.6$$

Expressions have been derived analytically to account for this particular combination in the context of aerosol filtration where the direction of flow and gravity forces coincide. Emi⁸⁰ proposed this equation,

$$\eta_{GI} = \frac{1+N_R}{1+N_G} \left\{ \frac{1}{2A} \left[\frac{1}{(1+N_R)^2} - 1 + \log_e (1+N_R)^2 \right] + N_G \right\} \quad 10.7$$

Pich ¹²⁵, who investigated the variation of capture by gravity forces with the angular position of a particle relative to a collector, suggested an alternative form,

$$\eta_{GI} = \eta_I + \eta_G + N_R N_G \quad 10.8$$

Within the range of variables relevant to this experimental study, sedimentation is predicted to be most significant when the velocity is 0.5×10^{-2} m/s and the drop diameter is 50 μ m.

Then from Table 10.2 $N_R = 1.639, N_{Re} = 0.1525, N_G = 0.0362$

from equation 5.1 $\eta_I = 0.3687$

from equation 5.5 $\eta_G = 0.0362$

For the combined effects of interception and sedimentation,

from the Additivity rule, equation 10.6, $\eta_{GI} = 0.4049$

from equation 10.7, $\eta_{GI} = 0.3677$

from equation 10.8, $\eta_{GI} = 0.4182$

From these results, the validity of the expression proposed by Emi must be jeopardised since it predicts a negative effect by coupling of the mechanisms which cannot be physically justified. As Pich's equation gives an efficiency which differs by only 3% from the result obtained using the additivity rule, the latter was selected as the equation for calculation of the drop capture efficiency.

10.3. RATE OF DROP CAPTURE.

Having established an equation to describe the drop capture efficiency of a single cylindrical collector, it is now possible to estimate the capture rate for a coalescer consisting of a series of meshes, each of which comprises an array of cylindrical fibres.

From Section 10.2, for a single cylindrical collector,

$$\eta_{GI} = \frac{1}{2A} \left[2(1+N_R) \log_e(1+N_R) + \frac{1}{(1+N_R)} - (1+N_R) \right] + N_G \quad 10.9$$

It was demonstrated in Chapter 9 that use of the hydrodynamic function derived by Keller predicted a Kozeny constant which showed excellent agreement with experimental values. Thus, it may be concluded that the Keller⁷³ equation adequately characterises flow through a bed of meshes that is initially free from the dispersed phase,

$$A = 1.257 (1-r)^{2.5} \quad 10.10$$

$$\text{where } r = \frac{d_c}{d_c + d_a}$$

The efficiency given by equation 10.9 is based upon the projected area of the collector and therefore the efficiency per layer, η_i is related to the fractional open area of the mesh,

$$\eta_i = \eta_{GL} \left[1 - \left\{ \frac{d_a}{d_a + d_c} \right\}^2 \right] \quad 10.11$$

The inlet drop size distribution curve (see Fig 7.3) was fitted to a distribution function by polynomial regression and the resulting expression employed to calculate the number of drops in each drop diameter interval ($2 \mu\text{m}$). The total capture rate for a mesh layer was then calculated by application of equations 10.9 and 10.11 to each size interval. Assuming no drop redispersion, the input of suspended drops to the second layer was determined as the difference between the drops entering the bed inlet and those captured in the first layer.

This calculation procedure was repeated, for a bed consisting of 120 mesh layers, using a computer program given in Appendix N. A graph, presented in Fig 10.4., was plotted from the program output, which shows the removal rate of suspended drops as a function of bed depth. Changing the value of velocity has a negligible effect on this curve because only the Gravity number in equation 10.9

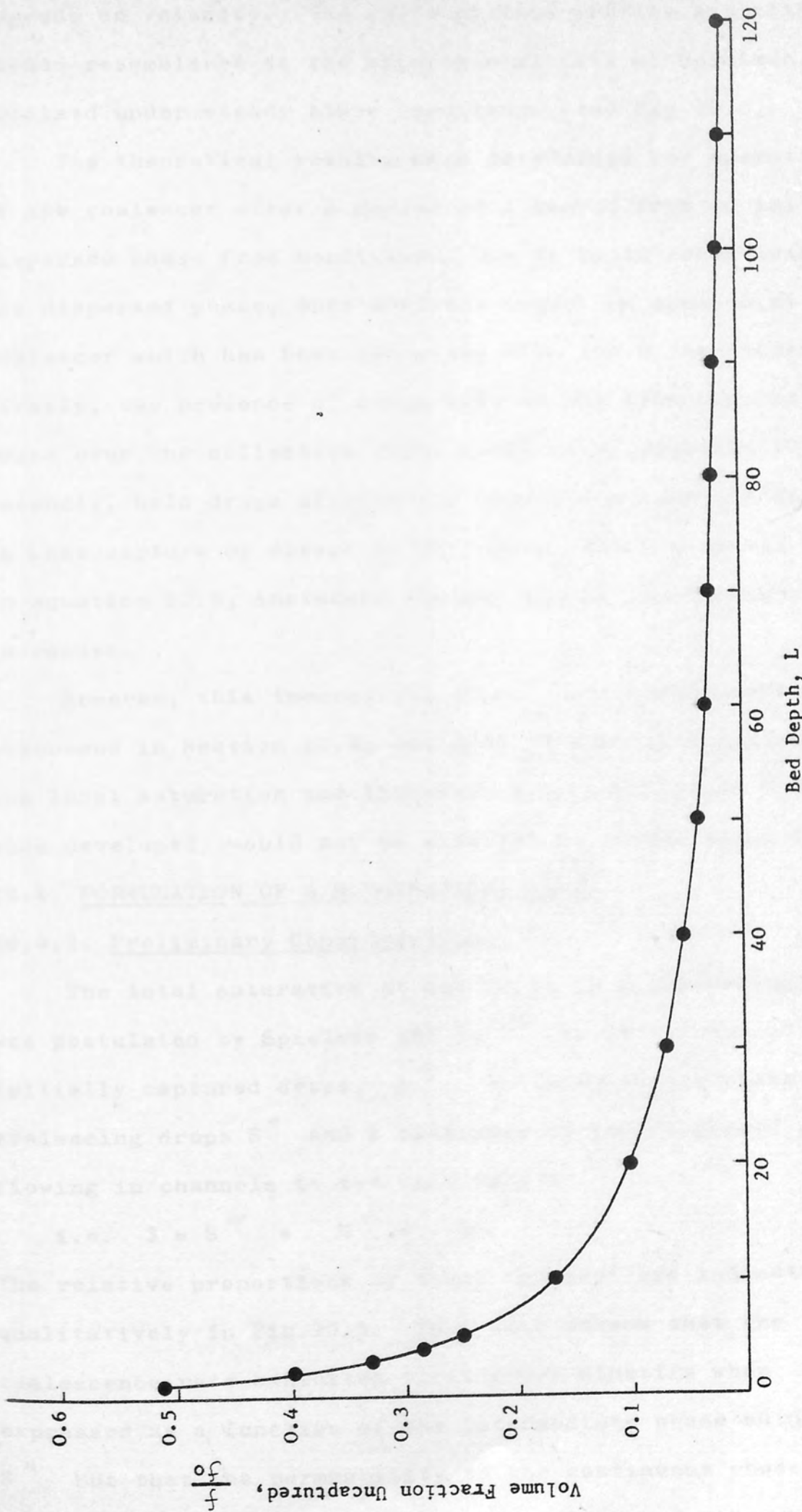


Fig.10.4 Variation of Volume Fraction of Dispersed Phase Uncaptured with Bed Depth.

depends on velocity. The shape of this profile exhibits a marked resemblance to the experimental data of Spielman⁶⁷ obtained under steady state conditions (see Fig 10.6).

The theoretical results were determined for operation of the coalescer after a period of 1 second from an initially dispersed phase free condition. Due to rapid accumulation of the dispersed phase, this analysis cannot be applied to a coalescer which has been operating even for a few seconds. Firstly, the presence of drops affects the flow of continuous phase over the collectors which invalidates equation 10.10. Secondly, held drops effectively reduce the aperture diameter so that capture by direct interception, which does not feature in equation 10.9, increases rapidly as the local saturation increases.

However, this theoretical approach is useful because, as discussed in Section 10.4, the drop capture rate depends on the local saturation and therefore the drop capture profile, once developed, would not be expected to change significantly.

10.4. FORMULATION OF A MATHEMATICAL MODEL.

10.4.1. Preliminary Considerations.

The total saturation at any point in a coalescing bed was postulated by Spielman and Su⁶⁷ to be the sum of the initially captured drops, S''' an intermediate phase of coalescing drops S'' and a continuum of the dispersed phase flowing in channels to the exit face S'

$$\text{i.e. } S = S''' + S'' + S' \quad 10.12$$

The relative proportions of these 'phases' are indicated qualitatively in Fig.10.5. They also showed that the coalescence rate exhibited first order kinetics when expressed as a function of the intermediate phase saturation, S'' but that the permeability to the continuous phase depended on the total saturation, S . It was further assumed

$$\begin{aligned} \text{Total Area} &= S \\ &= S' + S'' + S''' \end{aligned}$$

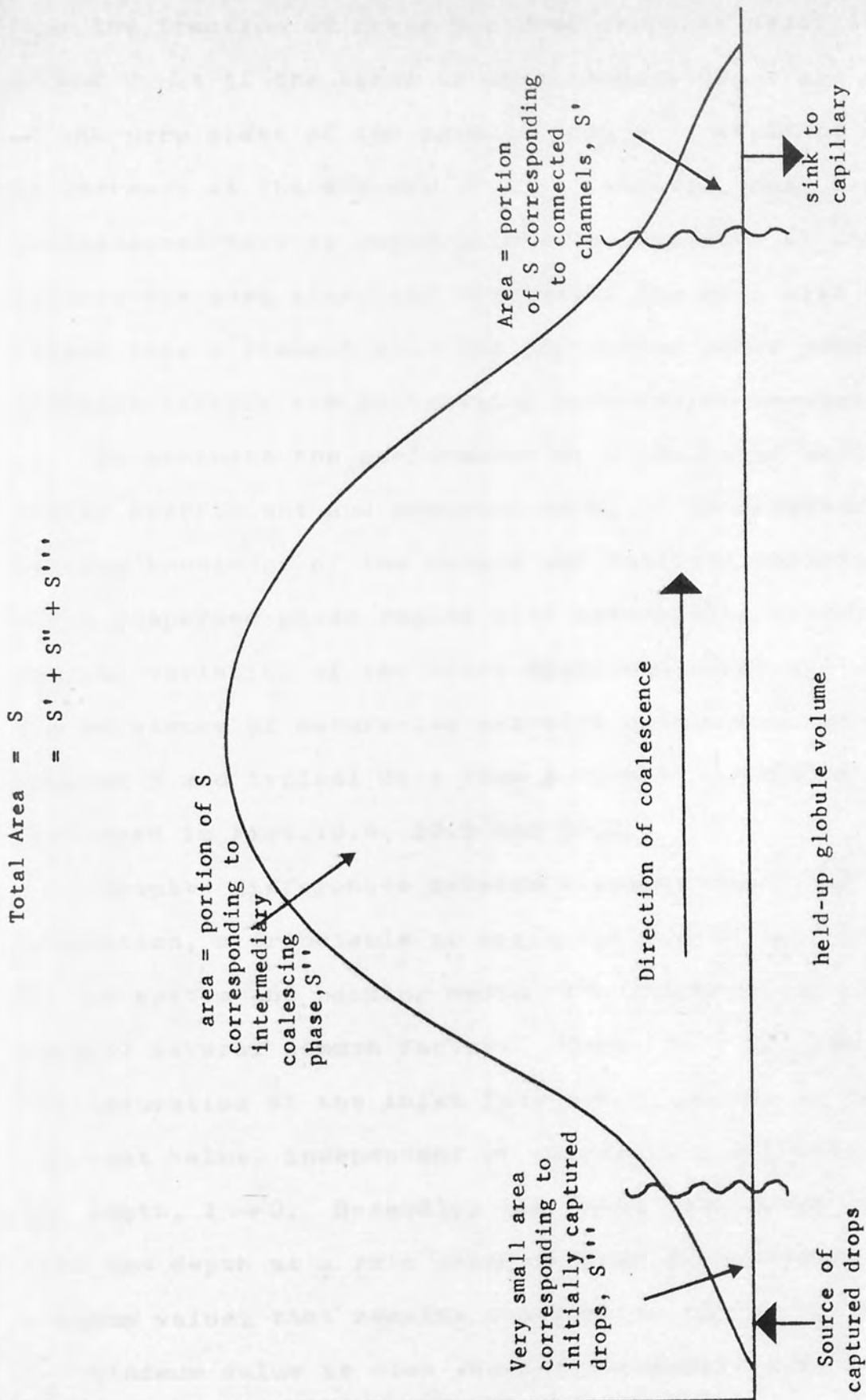


Fig.10.5 Qualitative distribution of the total volume of held-up oil over individual held-up globule volumes. Drops are captured at the left hand end of the spectrum. Dispersed phase is transferred to successively larger drops by coalescence eventually leaving by capillary conduction at the right.

Total Held-up Volume per unit increment of globule volume, per unit volume pore space

that the fraction of newly captured drops is negligible i.e. $S''' \rightarrow 0$ but if the sizes of these source drops are comparable to the pore sizes of the packing then S''' would be expected to increase at the expense of S'' . that is, only a few coalescences have to occur before the diameter of the drop exceeds the pore size, and eventually the drop will be forced into a channel when the continuous phase pressure gradient exceeds the restraining interfacial tension forces

To evaluate the performance of a coalescer in terms of filter coefficient and pressure drop, it is necessary to combine knowledge of the nature and relative amounts of the local dispersed phase regime with information concerning the spatial variation of the total dispersed phase saturation. The existence of saturation profiles has been discussed in Chapter 5 and typical data from a number of studies is presented in Figs.10.6, 10.7 and 10.8.

Despite differences between absolute values of local saturation, attributable to different operating parameters of the system and packing media, the shapes of the profiles exhibit several common factors. There is a maximum value of the saturation at the inlet face which appears to have a constant value, independent of superficial velocity, as the bed depth, $l \rightarrow 0$. Secondly, the local saturation decays with bed depth at a rate which depends on velocity; to a minimum value, that remains constant as far as the exit face. The minimum value is also shown experimentally to be a function of velocity. The profiles found by Shalhoub⁷⁰ and Bitten⁸⁵ indicate a slight increase in local saturation just upstream of the exit face for which no satisfactory explanation is available at present.

In this experimental study, it was not possible to measure the saturation profile without interfering with the

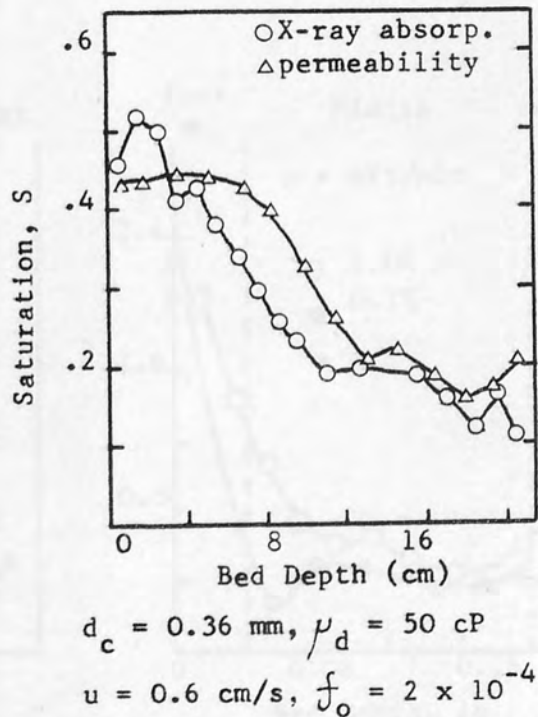
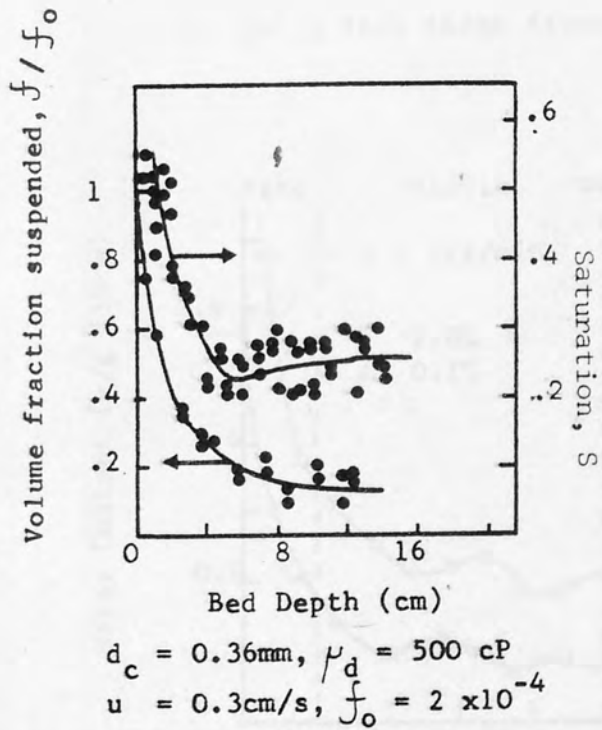
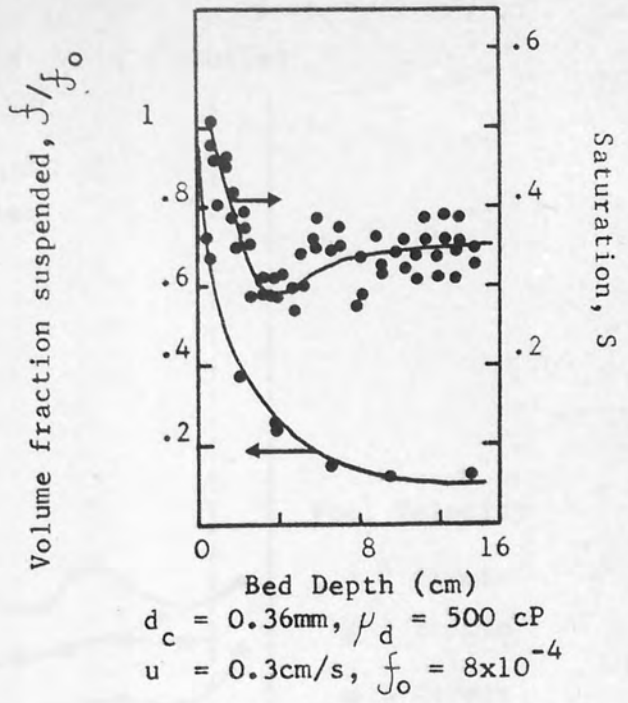
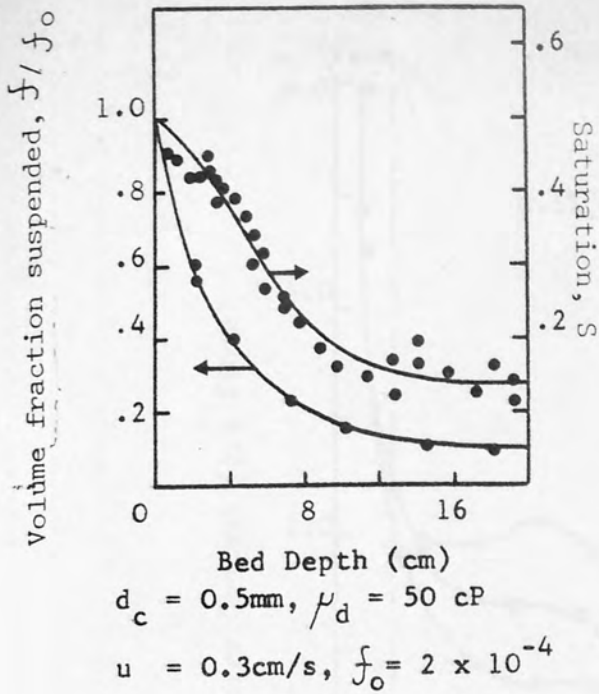
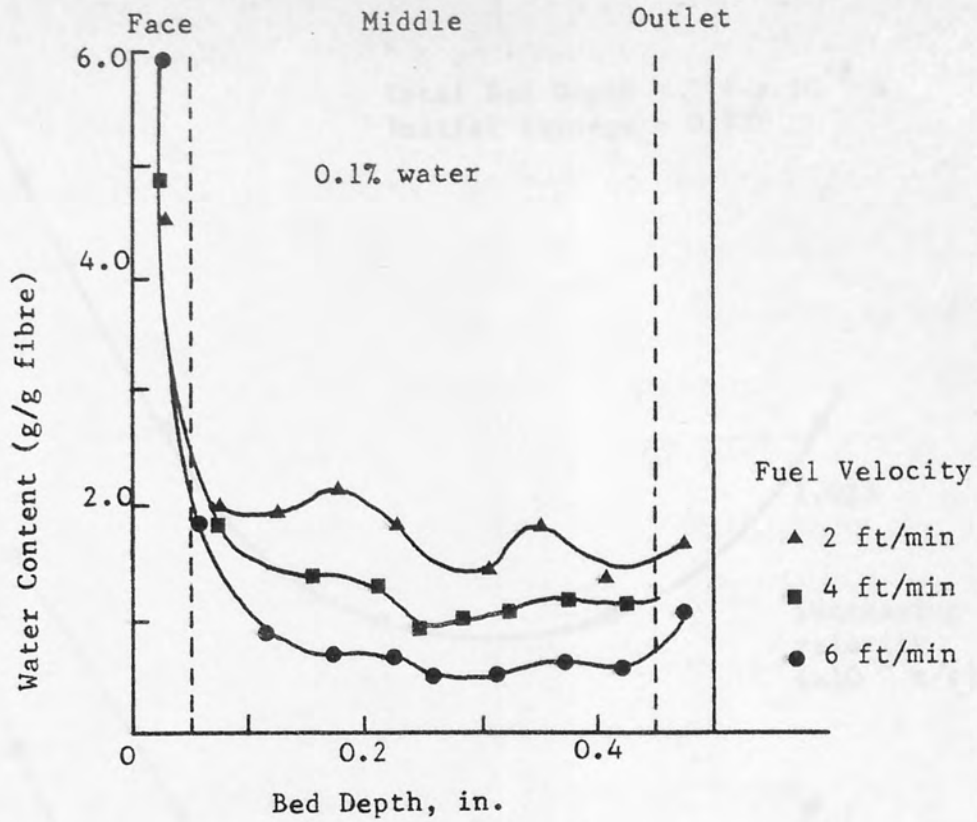
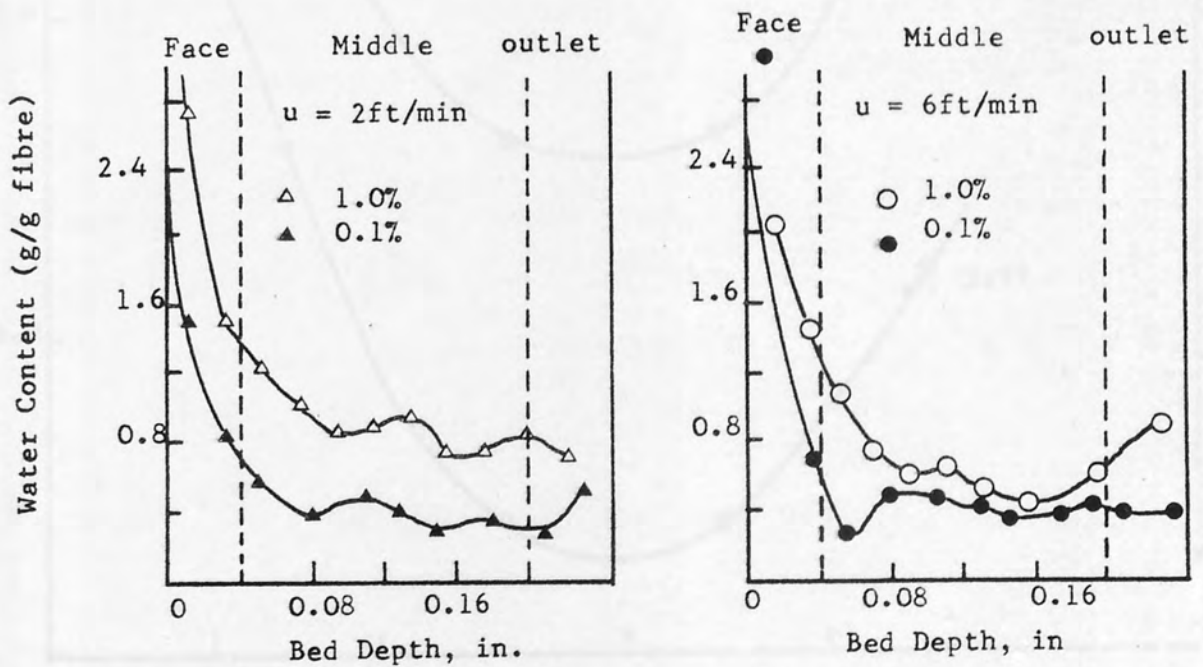


Fig 10.6 Variation of Volume Fraction of Suspended Drops and Dispersed Phase Saturation with Bed Depth.

(Data of Spielman and Su)



(a) $\frac{1}{2}$ inch thick fibreglass coalescer (6 lb/ft^3).



(b) $\frac{1}{4}$ inch thick fibreglass coalescer (14 lb/ft^3)

Fig 10.7 Variation of Hold-up with Bed Depth.

(Data of Bitten and Fochtman).

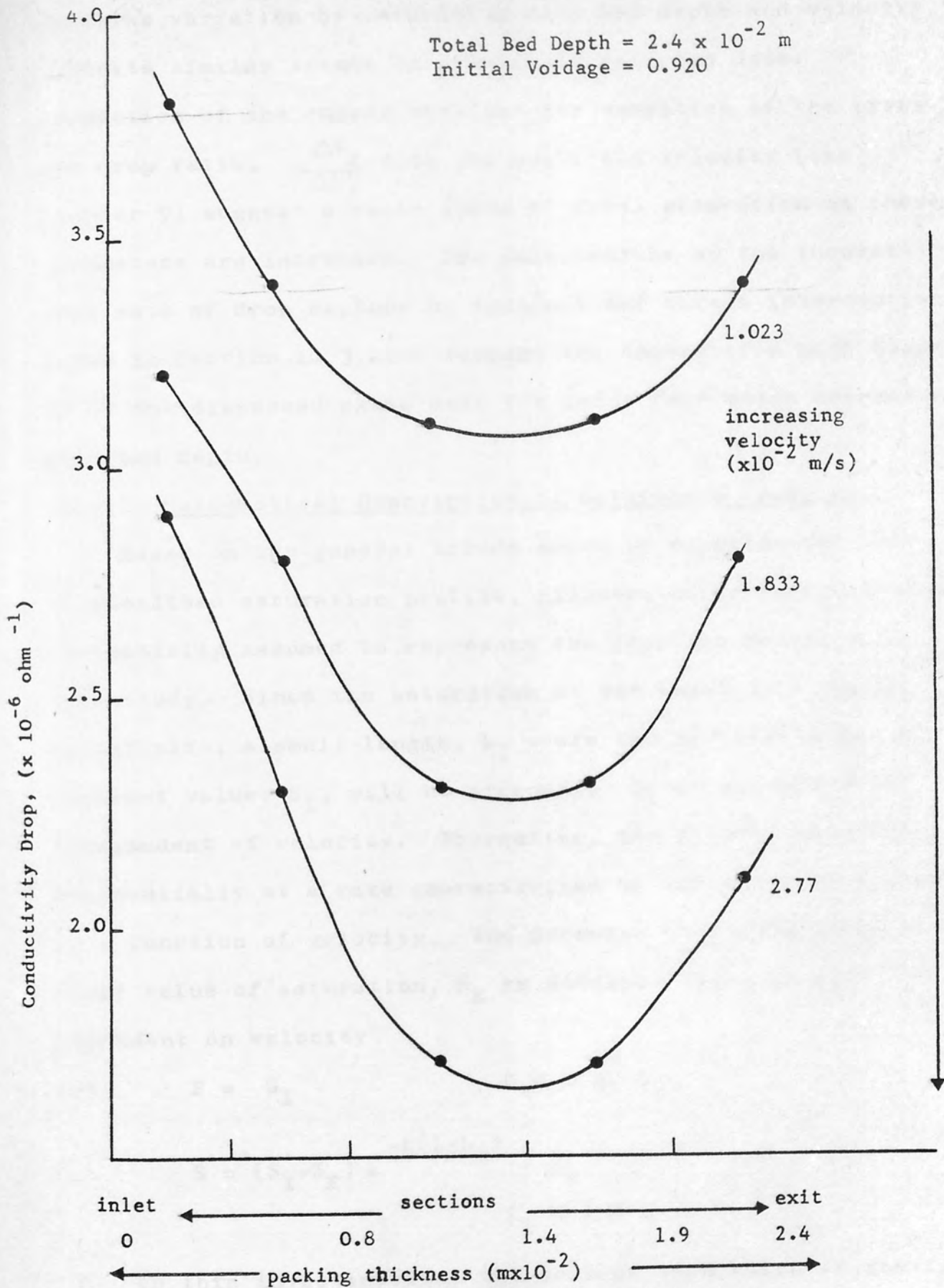


Fig 10.8 Variation of Hold-up with Superficial Velocity (Data of Shalhoub).

coalescence process and therefore it was necessary to assume that the variation of saturation with bed depth and velocity exhibits similar trends to previously reported data.

Inspection of the curves obtained for variation of the pressure drop ratio, $\frac{\Delta P_2}{\Delta P_1}$ with bed depth and velocity (see Chapter 9) suggest a rapid decay of local saturation as these parameters are increased. The calculations of the theoretical rate of drop capture by indirect and direct interception given in Section 10.3 also support the theory of a high hold-up of the dispersed phase near the inlet face which decreases with bed depth.

10.4.2. Mathematical Description of Saturation Profiles.

Based on the general trends shown by experimental data, an idealised saturation profile, illustrated in Fig.10.9 will be initially assumed to represent the profiles existing in this study. Since the saturation at the inlet face cannot be infinite, a small length, L_I where the saturation has a constant value, S_I , will be proposed. S_I is assumed to be independent of velocity. Thereafter, the saturation decays exponentially at a rate characterised by the value of k which is a function of velocity. The decrease continues until a final value of saturation, S_E is attained which is also dependent on velocity.

i.e. $S = S_I \quad 0 < l \leq L_I$

$$S = (S_I - S_E) e^{-k(1-L_I)} + S_E \quad 10.13$$

$$L_I \leq l \leq L$$

In this investigation, the average saturation at steady state, \bar{S} was determined on completion of an experiment and may be expressed mathematically by integration of the profile between 0 and L,

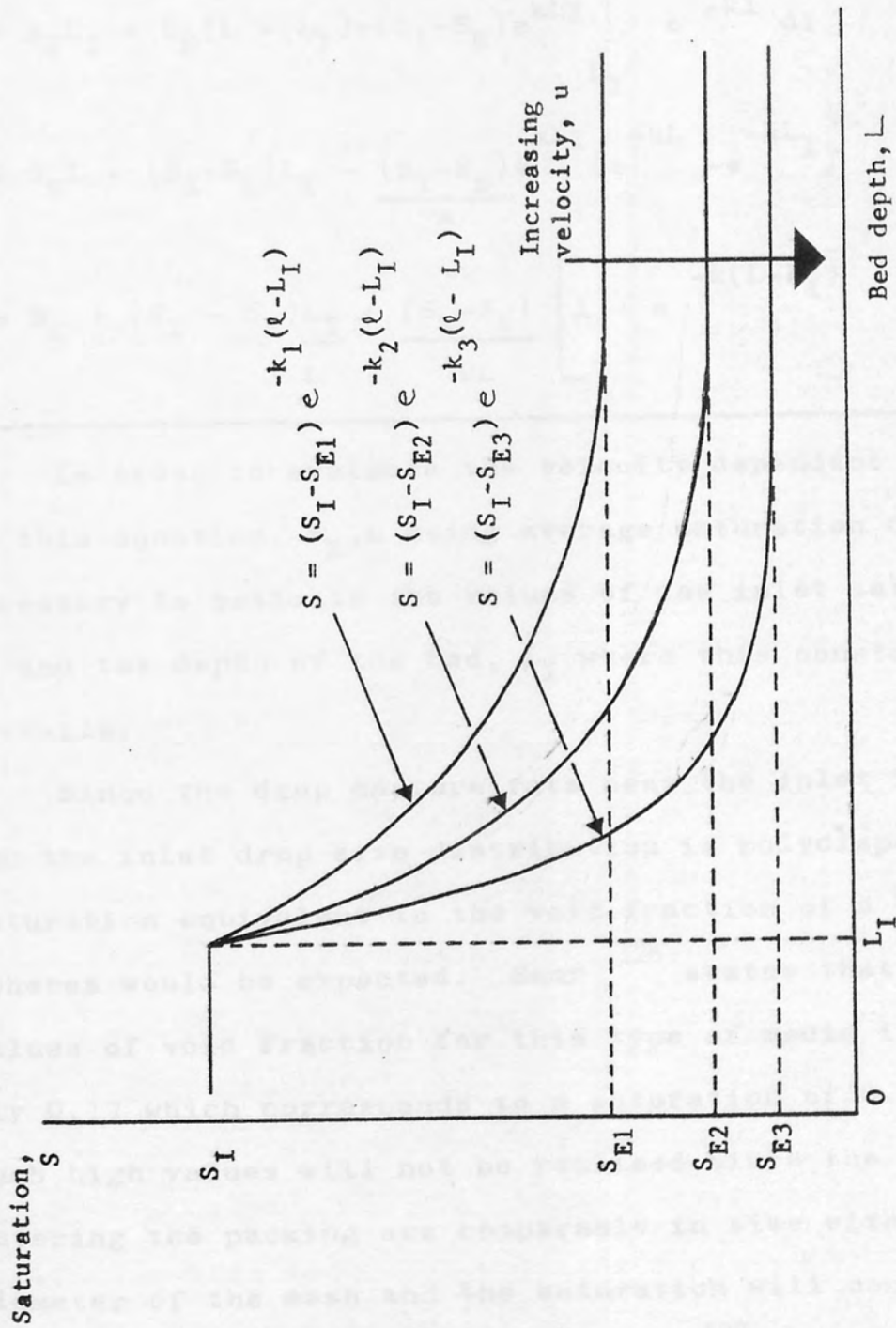


Fig. 10.9. Idealised Saturation Profiles.

$$\bar{S} = \frac{\int_0^L S dl}{\int_0^L dl}$$

$$\bar{S}L = \int_0^{L_I} S_I dl + \int_{L_I}^L (S_I - S_E) e^{-k(1-L_I)} + S_E dl$$

$$= S_I L_I + S_E (L - L_I) + (S_I - S_E) e^{kL_I} \int_{L_I}^L e^{-kl} dl$$

$$= S_E L + (S_I - S_E) L_I - \frac{(S_I - S_E)}{k} (e^{-kL} - e^{-kL_I})$$

$$\bar{S} = S_E + (S_I - S_E) \frac{L_I}{L} + \frac{(S_I - S_E)}{kL} \left[1 - e^{-k(L-L_I)} \right] \quad 10.14$$

In order to evaluate the velocity dependent parameters of this equation, S_E, k using average saturation data, it is necessary to estimate the values of the inlet saturation, S_I and the depth of the bed, L_I where this constant value prevails.

Since the drop capture rate near the inlet face is high and the inlet drop size distribution is polydisperse, a saturation equivalent to the void fraction of a bed of mixed spheres would be expected. Bear¹²⁶ states that typical values of void fraction for this type of media is approximately 0.17 which corresponds to a saturation of 0.83. However, such high values will not be realised since the drops entering the packing are comparable in size with the aperture diameter of the mesh and the saturation will consequently be reduced by this 'wall effect'. Brown¹²⁷ gives a correlation between void fraction and the diameter ratio, $\frac{d_p}{d_a}$ which for a mean drop size of $25 \mu\text{m}$ and a aperture diameter of $53 \mu\text{m}$, $\frac{d_p}{d_a} = 0.472$ gives $e_p = 0.45$ corresponding to a

saturation of 0.55. This value is in close agreement with the data of Spielman and will be used as an initial estimate of S_I .

The experimentally determined profiles all indicate that the decay of saturation with bed depth occurs very close to the inlet face implying that L_I is very small. However, since the saturation must be zero at the inlet face, in the absence of flooding, a small finite value will be assigned to L_I . Again, as an initial estimate, L_I will be assumed to be equal to the length of a single layer of mesh i.e. $2d_c$. Although the saturation will increase from 0 to S_I within a very small distance from the inlet face i.e. $\ll L_I$, omission of this section of the saturation profile from the integration of equation 10.13 will not incur any significant error.

Inspection of equation 10.14 shows that if the parameter, k is large, then the decay will approximate to a step change in saturation at L_I from S_I to S_E , and equation 10.14 becomes

$$\bar{S} \approx S_E + (S_I - S_E) \frac{L_I}{L} \quad \text{for large } k \quad 10.15$$

Figs.10.6, 10.7 and 10.8 indicate that this condition may be satisfied, especially for high velocities and bed depths.

10.5. TWO PHASE PRESSURE DROP PREDICTION.

10.5.1. Previous Models.

The prediction of pressure drop across a coalescing bed has been attempted only by those workers who advocated the travelling drop model which implies a constant saturation throughout the length of the bed. The equation proposed by Sherony⁶⁴ was based on the Blake-Kozeny equation which on rearrangement, may be expressed as,

$$\frac{\Delta P_2}{\Delta P_1} = \frac{[1 - e_1 (1 - \bar{S})]^2}{(1 - e_1)^2 (1 - \bar{S})^3} \quad 10.16$$

where the saturation, \bar{S} is determined from a curve giving the variation of $\frac{e_1 \bar{S}}{(1 - e_1)}$ with Reynolds Number, $N_{Re} = \frac{ud_c \rho_c}{\mu_c}$

Rosenfeld and Wasan suggest a similar, simplified equation,

$$\frac{\Delta P_2}{\Delta P_1} = \left[1 + \frac{e_1 \bar{S}}{1 - e_1} \right]^{2/3} \quad 10.17$$

where \bar{S} may be correlated with superficial velocity using

$$\bar{S} = C \frac{(1 - e_1) u}{e_1}^{-n}$$

Rosenfeld determined the values of the parameters, C and n to be 0.8 and 0.2 respectively when u is measured in ft/min. These models are inadequate representations of experimental data since they predict that the pressure drop ratio is independent of bed depth which is contrary to the variation shown in Figs. 9.6, 9.7, 9.8, 9.9, 9.10, 9.11. An entirely empirical approach was adopted by Euzen¹²⁴ who fitted their data to the following type equation,

$$\frac{\Delta P_2}{\Delta P_1} = k^1 (uL)^{-0.14} \quad 10.18$$

where the constant k^1 depends on the liquid system studied. Correlation of the data in the above form was attempted in Chapter 9 but the value of the exponent was found to vary considerably for different mesh types and a better fit was obtained by considering the exponent of u and L separately.

10.5.2. Derivation of Proposed Equation.

Therefore, generally, the ability of existing equations to predict two phase pressure drop for the operating conditions applied in this study was not acceptable. As the

Blake-Kozeny equation proved to be successful for correlating single phase pressure drop, it was used as the basis for this model. However, the Blake Kozeny equation must be modified since during two phase flow, the bed consists of a mixture of 'cylinders' and 'spheres' which possess different values of specific surface, a .

For single phase flow through an element of bed depth; δl ,

$$\frac{\delta P_1}{\delta l} = \frac{K \mu_c u (1-e_1)^2 a_1^2}{e_1^3} \quad 10.19$$

by writing equation 10.19 by analogy for two phase flow

$$\frac{\delta P_2}{\delta l} = \frac{K \mu_c u (1-e_2)^2 a_2^2}{e_2^3} \quad 10.20$$

where $e_2 = e_1 (1 - S)$

$$a_2 = \frac{\left[\frac{6e_1 S}{d_p} + \frac{4(1-e_1(1-S))}{d_c} \right]}{1 - e_1(1-S)} \quad \text{for } a$$

mixture of spheres and cylinders (see Appendix O)

substituting for e_2 and a_2 into 10.20 and taking the limit as $\delta l \rightarrow 0$

$$\frac{dP_2}{dl} = \frac{K \mu_c u}{e_1^3} \frac{\left[\frac{6e_1 S (1-e_1(1-S))}{d_p} + \frac{4(1-e_1(1-S))}{d_c} \right]^2}{(1-S)^3} \quad 10.21$$

The total two phase pressure drop may now be determined by integration of equation 10.21 between the limits of 0 and L

$$\therefore \Delta P_2 = \frac{K \mu_c u}{e_1^3} \int_0^L \frac{\left[\frac{6e_1 S [1-e_1(1-S)]}{d_p} + \frac{4 [1-e_1(1-S)]}{d_c} \right]^2}{(1-S)^3} dl \quad 10.22$$

Similarly from equation 10.19,

$$\Delta P_1 = \frac{16K \mu_c u (1-e_1)^2 L}{d_c^2 e_1^3} \quad 10.23$$

$$\text{then } \frac{\Delta P_2}{\Delta P_1} = \frac{d_c^2}{16(1-e_1)^2 L} \int_0^L f(s) dl \quad 10.24$$

$$\text{where } f(s) = \frac{\left\{ \frac{6}{d_p} \left[e_1 s (1-e_1(1-s)) \right] + \frac{4}{d_c} \left[1-e_1(1-s) \right] \right\}^2}{(1-s)^3} \quad 10.25$$

by letting $a' = \frac{6}{d_p}$ and $b' = \frac{4}{d_c}$, the numerator of the right hand side of equation 10.25 becomes,

$$\left\{ a' e_1 s \left[1-e_1 + e_1 s \right] + b' \left[1-e_1 + e_1 s \right] \right\}^2$$

further rearrangement gives,

$$\left\{ a' e_1^2 s^2 + e_1 \left[b' + a'(1-e_1) \right] s + b'(1-e_1) \right\}^2$$

introducing the following constants to simplify the coefficients,

$$\text{let } c' = a' e_1^2 ; d' = e_1 \left[b' + a'(1-e_1) \right]$$

$$e' = b' (1-e_1)$$

equation 10.25 is given by,

$$f(s) = \frac{(c' s^2 + d' s + e')^2}{(1-s)^3}$$

$$\text{or } f(s) = \frac{a s^4 + b s^3 + c s^2 + d s + f}{(1-s)^3}$$

$$\text{where } a = c'^2 ; b = 2 b' c' ; c = 2 c' e' + d'^2$$

$$d = 2 d' e' ; f = e'^2$$

considering equation 10.24, the integration of $f(s)$ cannot be performed in one stage since a discontinuity appears in the mathematical description of the saturation profile. From equation 10.13, when $0 < l \leq L_I$, $S = S_I$ and is independent

of 1, therefore,

$$\int_0^{L_I} f(S) dl = \left[\frac{a S_I^4 + b S_I^3 + c S_I^2 + d S_I + f}{(1-S_I)^3} \right]_{L_I} \quad 10.26$$

When $L_I \leq l \leq L$, $S = (S_I - S_E) e^{-k(1-L_I)} + S_E$

Letting $g = (S_I - S_E)$; $h = S_E$

and substituting $v = g e^{-k(1-L_I)} + h$

$$\frac{dv}{dl} = -k g e^{-k(1-L_I)}$$

$$= -k (v-h)$$

or $dl = - \frac{dv}{k(v-h)}$

also when $l = L_I$, $v = g + h$

$l = L$, $v = g e^{-k(L-L_I)} + h$

$$\therefore \int_{L_I}^L f(S) dl = \int_{g+h}^{g e^{-k(L-L_I)} + h} f(v) \frac{-dv}{k(v-h)}$$

$$= \frac{1}{k} \int_{g e^{-k(L-L_I)} + h}^{g+h} \frac{(av^4 + bv^3 + cv^2 + dv + f)}{(v-h)(1-v)^3} dv \quad 10.27$$

Evaluation of the integral given by equation 10.27 is fairly straightforward but tedious and so has been completed in Appendix P. The final result (equation P.9) is lengthy and expansion of the presented form would be complex and unnecessary but quite feasible since this is an analytical solution.

The total pressure drop across the bed will therefore be predicted using,

$$\frac{\Delta P_2}{\Delta P_1} = \frac{d_c^2}{16(1-e_1)L} \left[\int_0^{L_I} f(S) dl + \int_{L_I}^L f(S) dl \right] \quad 10.28$$

where $\int_0^{L_I} f(S) dl$ is determined using equation 10.26

and $\int_{L_I}^L f(S) dl$ is determined using equation P.9.

10.6. TESTING THE MODEL.

10.6.1. Evaluation of Parameters from Saturation Data.

As discussed in Section 10.4.2., using assumed values of S_I and L_I , it is possible to estimate the exit saturation, S_E and the decay factor, k . From equation 10.14,

$$\bar{S} = S_E + (S_I - S_E) \frac{L_I}{L} + \frac{(S_I - S_E)}{kL} \left[1 - e^{-k(L-L_I)} \right] \quad 10.14$$

When the rate of decay is large and L_I is small compared with L , then $e^{-k(L-L_I)} \rightarrow 0$ and equation 10.14 becomes,

$$S = S_E + (S_I - S_E) \left(L_I + \frac{1}{k} \right) \cdot \frac{1}{L} \quad 10.29$$

The average saturation results for the stainless steel meshes were plotted against the reciprocal of bed depth as shown in Fig 10.10. The linear relationship produced for all velocities confirms that the model is qualitatively valid.

The intercept, S_E and the slope, $(S_I - S_E) \left(L_I + \frac{1}{k} \right)$ were determined by linear regression analysis of the data in Fig 10.10. Then the four parameters characterising the proposed saturation profiles, S_I , S_E , L_I and k were used to predict the two phase pressure drop ratio from equation 10.28.

10.6.2. Comparison with Experimental Data.

A computer program was employed to calculate $\frac{\Delta P_2}{\Delta P_1}$ (see Appendix Q.1) from the parameters extracted from saturation data and the output, which expresses the ratio, $\frac{\Delta P_2}{\Delta P_1}$ in terms of total bed depth for each velocity, is given in Appendix Q.2. The predicted ratios were then plotted against experimental values from Appendix H. Although some

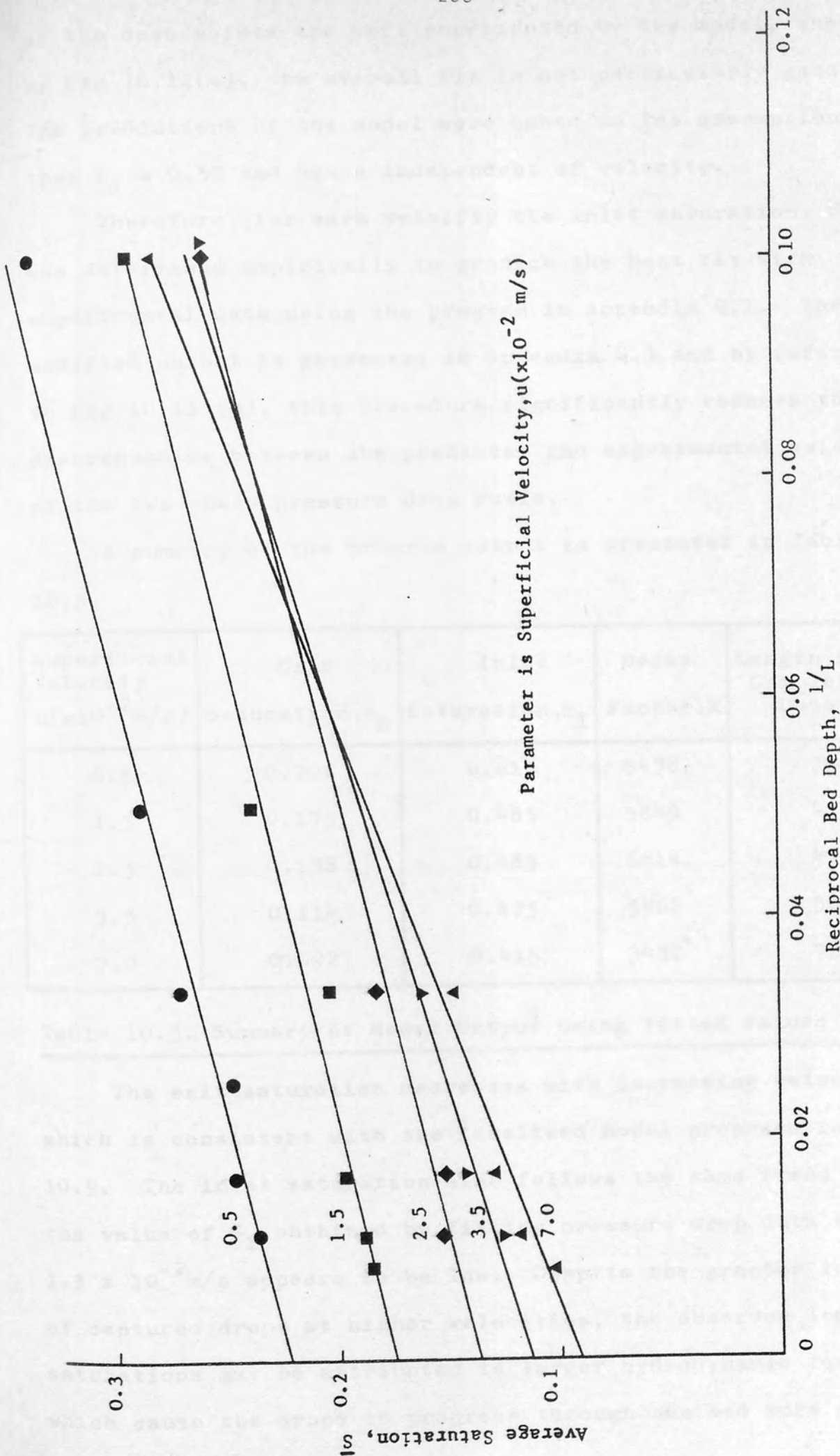


Fig. 10.10. Plot of average Saturation VS. Reciprocal Bed Depth for S300 Mesh at different velocities.

of the data points are well represented by the model, shown by Fig 10.11(a), the overall fit is not particularly good. The predictions of the model were based on the assumption that $S_I = 0.55$ and hence independent of velocity.

Therefore, for each velocity the inlet saturation, S_I was determined empirically to produce the best fit with experimental data using the program in Appendix Q.1. The modified output is presented in Appendix Q.3 and by reference to Fig 10.11 (b), this procedure significantly reduces the discrepancies between the predicted and experimental values of the two phase pressure drop ratio.

A summary of the program output is presented in Table 10.5.

Superficial Velocity $u(x10^{-2} \text{ m/s})$	Exit Saturation, S_E	Inlet Saturation, S_I	Decay Factor, k	Length for Complete Decay
0.5	0.226	0.615	8478	33
1.5	0.175	0.485	5849	47
2.5	0.138	0.485	6614	41
3.5	0.114	0.475	5462	50
7.0	0.092	0.415	3432	78

Table 10.5. Summary of Model Output using fitted values of S_I

The exit saturation decreases with increasing velocity which is consistent with the idealised model proposed in Fig 10.9. The inlet saturation also follows the same trend but the value of S_I obtained by fitting pressure drop data for $1.5 \times 10^{-2} \text{ m/s}$ appears to be low. Despite the greater influx of captured drops at higher velocities, the observed lower saturations may be attributed to larger hydrodynamic forces which cause the drops to progress through the bed more quickly.

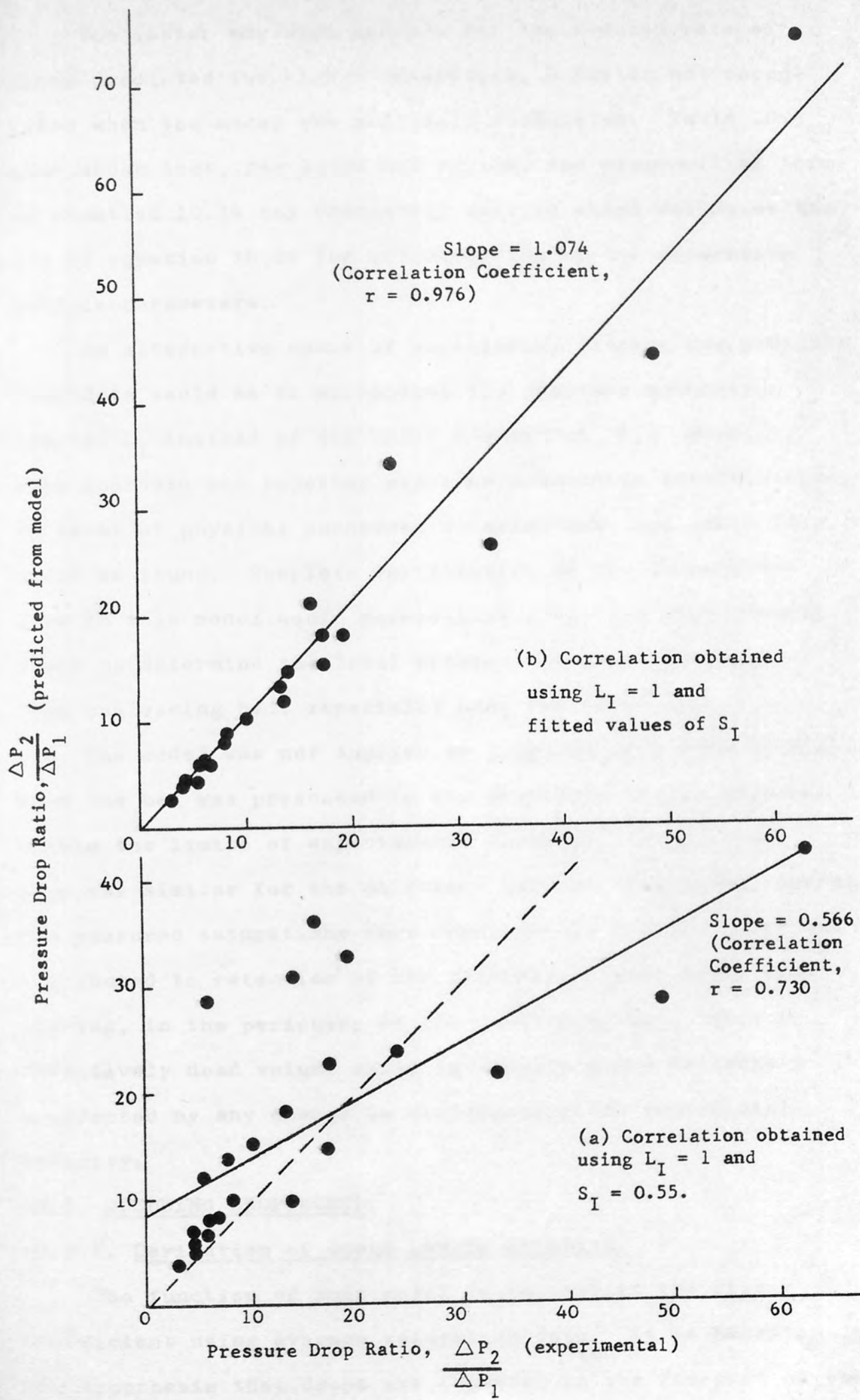


Fig.10.11. Comparison of Experimental and Predicted Pressure Drop.

The latter may also account for the reduced rate of decay predicted for higher velocities, a factor not recognised when the model was initially formulated. Table 10.5 also shows that, for large bed depths, the exponential term of equation 10.14 has completely decayed which validates the use of equation 10.29 for determination of the saturation profile parameters.

An alternative means of empirically fitting the pressure drop data would be to manipulate the constant saturation length, L_I instead of the inlet saturation, S_I . However, this approach was rejected since no reasonable justification, in terms of physical phenomena occurring near the inlet face, could be found. Complete verification of the assumptions made in this model would necessitate a further experimental study to determine the local saturations in a relatively deep coalescing bed, especially near the inlet face.

The model was not applied to pressure drop data obtained when the bed was presoaked in the dispersed phase, because, within the limits of experimental accuracy, the pressure drop was similar for the different initial conditions, whereas the measured saturations were considerably higher. This was attributed to retention of the dispersed phase, after presoaking, in the periphery of the coalescing bed. This is effectively dead volume whose saturation would be largely unaffected by any change in continuous phase superficial velocity.

10.7. QUEUEING DROP MODEL.

10.7.1. Derivation of Queue Length Equation.

The function of this model is to predict the filter coefficient using average saturation data. It is based on the hypothesis that drops are captured in the forepart of the bed where they accumulate until coalescence into a dispersed

phase continuum occurs. The dispersed phase is then conveyed in discrete channels to the release sites located on the exit face of the coalescer. As shown in Fig 10.12, the model recognises an analogy between drops waiting to coalesce into a dispersed phase continuum and 'customers' queueing at a service facility. A coalescer may therefore be considered to behave as a number of service channels in parallel, each of mean service rate, μ . The 'customer' arrival rate, due to capture by fibres and held drops, has a mean value, λ . Prediction of the mean queue length may then be employed to estimate the position of the service facility in the bed and hence determine the dispersed phase saturation.

This analogy may be pursued further by considering 'customer impatience' at a service facility which corresponds to redispersion of drops before coalescence into the continuum. The fraction of 'customers' which join the queue when there are n units in the system is given by,

$$f_n = e^{-\frac{\alpha n}{\mu}} \quad 10.30$$

This expression indicates that redispersion becomes more important as the saturation increases (n increases) or as coalescence time increases (θ or $1/\mu$ increases). The 'impatience factor', α which must be determined experimentally is analogous to the ratio describing the rate of coalescence to the rate of drop capture⁶⁴.

If there are n units in the queueing system, the probability that there are n units in the system at time $t+dt$ is $P_n(t+dt)$. If the processes which cause a transition from this state are independent of time which is true for steady state operation,

$$P_n(t+dt) = \sum_m T_{mn}(dt) P_m(t) \quad 128 \quad 10.31$$

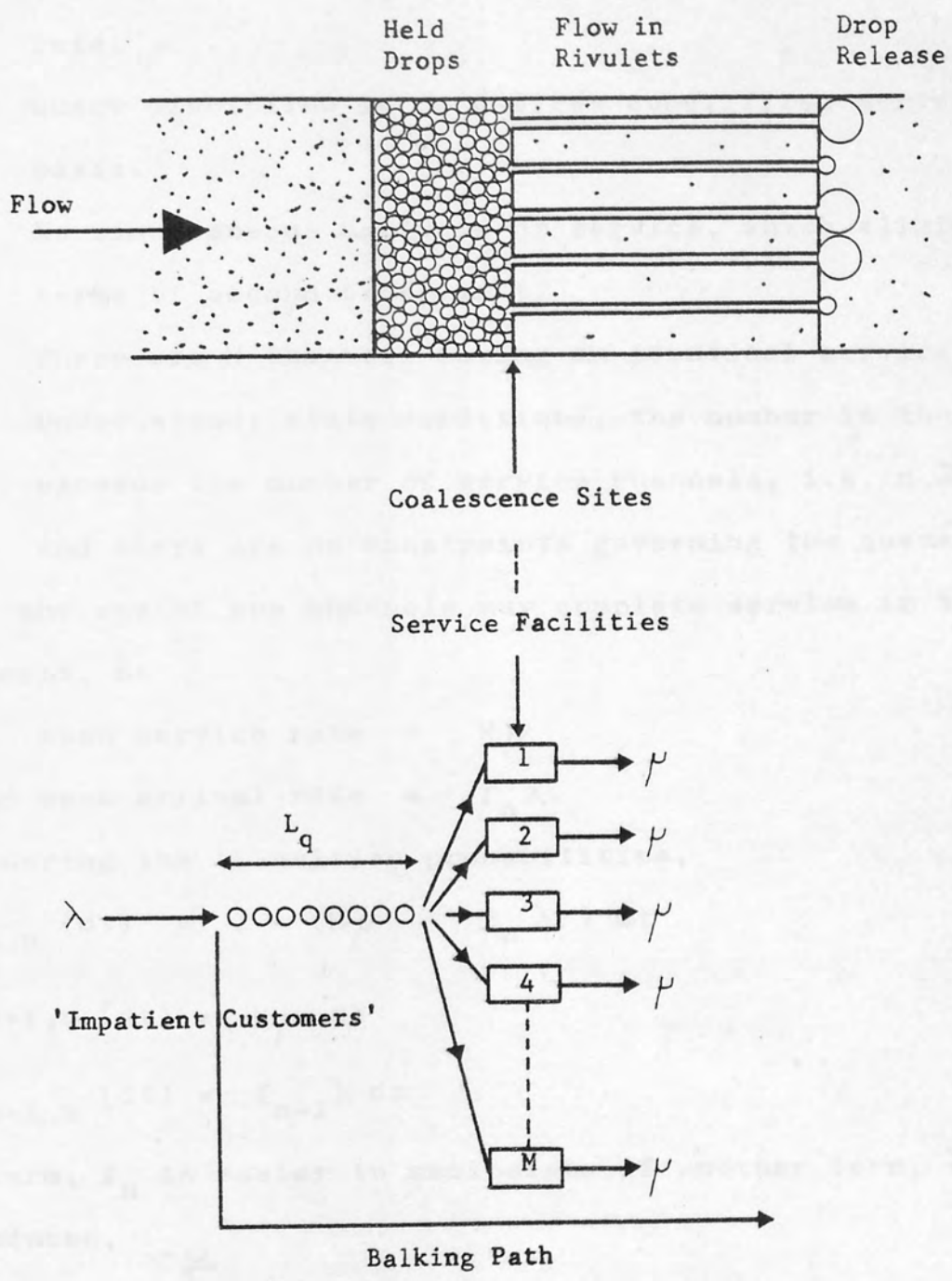


Fig 10.12 Diagrammatic representation for analogy between Coalescence Process and Multiple Channel Queuing System.

The following assumptions are now introduced,

- (i) Poisson arrival distribution of mean arrival rate, λ .
- (ii) Exponential service distribution of mean service rate, μ .
- (iii) Queue discipline is on a first come, first served basis.
- (iv) No simultaneous arrivals or service, which eliminates terms of second order in t .
- (v) There are M channels having an identical service rate.
- (vi) Under steady state conditions, the number in the system exceeds the number of service channels, i.e. $n \geq M$ and there are no constraints governing the queue length.

Since any one of the channels may complete service in the time increment, dt

$$\text{mean service rate} = M\mu$$

$$\text{and mean arrival rate} = f_n \lambda$$

Considering the transition probabilities,

$$T_{n,n}(dt) = 1 - (M\mu + f_n \lambda) dt \quad 10.32$$

$$T_{n+1,n}(dt) = M\mu dt \quad 10.33$$

$$T_{n-1,n}(dt) = f_{n-1} \lambda dt \quad 10.34$$

The term, f_n is easier to manipulate if another term, γ is introduced,

$$\gamma = e^{-\frac{\alpha}{2\mu}}$$

$$\text{then } f_n = \gamma^{2n} \quad ; \quad f_{n-1} = \gamma^{2n-2} \quad 10.35$$

substituting for the transition probabilities from equations 10.32, 10.33 and 10.34 into equation 10.31 and noting that,

$$P_n(t+dt) = P_n(t) dt + dP_n$$

gives the following,

$$P_n(t)dt + dP_n = \left[M\mu P_{n+1}(t) + \{1 - (M\mu + \gamma^{2n} \lambda)\} P_n(t) + \gamma^{2n-2} \lambda P_{n-1}(t) \right] dt \quad 10.36$$

dividing by dt and setting $\frac{dP_n}{dt} = 0$ for steady state conditions,

$$M\mu P_{n+1} + \gamma^{2n-2} \lambda P_{n-1} - (M\mu + \gamma^{2n} \lambda) P_n = 0 \quad 10.37$$

equation 10.37 takes a special form for $n = 0$ since P_{-1} does not exist and there is zero probability of moving from the state, $n = 0$ by completion of a service operation.

i.e. $P_{-1} = 0$; $n\mu P_n = 0$; $\gamma^{2n} = 1$

$$\therefore \mu P_1 - \lambda P_0 = 0 \quad 10.38$$

dividing equation 10.37 by μM and defining the utilisation factor, $\rho = \frac{\lambda}{\mu M}$,

$$P_{n+1} + \rho \gamma^{2n-2} P_{n-1} - (1 + \rho \gamma^{2n}) P_n = 0 \quad 10.39$$

for $n \geq M$.

If $n \leq M$, equation 10.39 becomes,

$$\left. \begin{aligned} (n+1)P_{n+1} + \rho \gamma^{2n-2} P_{n-1} - (n + \rho \gamma^{2n}) P_n &= 0 \\ P_1 &= \rho P_0 \quad \text{where } \rho = \frac{\lambda}{\mu} \end{aligned} \right\} \quad 10.40$$

The general solution of these equations is obtained by successively increasing the value of n , hence,

$$P_n = \frac{\rho^n \gamma^{n(n-1)}}{n!} P_0 \quad 1 \leq n \leq M \quad 10.41$$

$$P_n = \frac{\rho^n \gamma^{n(n-1)M}}{M!} P_0 \quad M \leq n \quad 10.42$$

The sum of the probabilities for all possible states of the system must be unity,

$$\therefore \sum_{n=1}^{\infty} P_n = 1$$

from equation 10.42,

$$\sum_{n=1}^{\infty} \frac{\rho^n \gamma^{n(n-1)M} P_0}{M!} = 1$$

$$\text{hence, } P_0 = \frac{M!}{M \sum_{n=1}^{\infty} \rho^n \gamma^{n(n-1)}} \quad 10.43$$

The length of the queue, L_q is given by,

$$L_q = \sum_{n=1}^{\infty} n P_n$$

$$\therefore L_q = \frac{\sum_{n=1}^{\infty} n \rho^n \gamma^{n(n-1)}}{\sum_{n=1}^{\infty} \rho^n \gamma^{n(n-1)}} \quad 10.44$$

10.7.2. Filter Coefficient.

Experimental data, as discussed in Chapter 8, indicates that secondary drops in the effluent arise because they are not captured rather than being produced by redispersion effects. The filter coefficient may therefore be related to the fraction of customers, f_n which undergo 'balking'. Writing equation 10.35 for steady state conditions when $n = L_q$,

$$f_{L_q} = \gamma^{2L_q} \quad 10.45$$

Now, from Section 10.1, the filter coefficient, λ_c is given by,

$$\lambda_c = \frac{-\log_e (1 - f_{L_q})}{L} \quad 10.46$$

Substituting for f_{L_q} from equation 10.45 and the bed depth, L ,

$$\lambda_c = \frac{-\log_e (1 - \gamma^{2L_q})}{2d_c N_L} \quad 10.47$$

The length of the 'queue', L_q , given by equation 10.44 is independent of the number of channels but is a function of γ . Assuming a utilisation factor of 1.0, values of 'queue' length were generated for different 'impatience' factors using a computer program and the results presented in Fig 10.13.

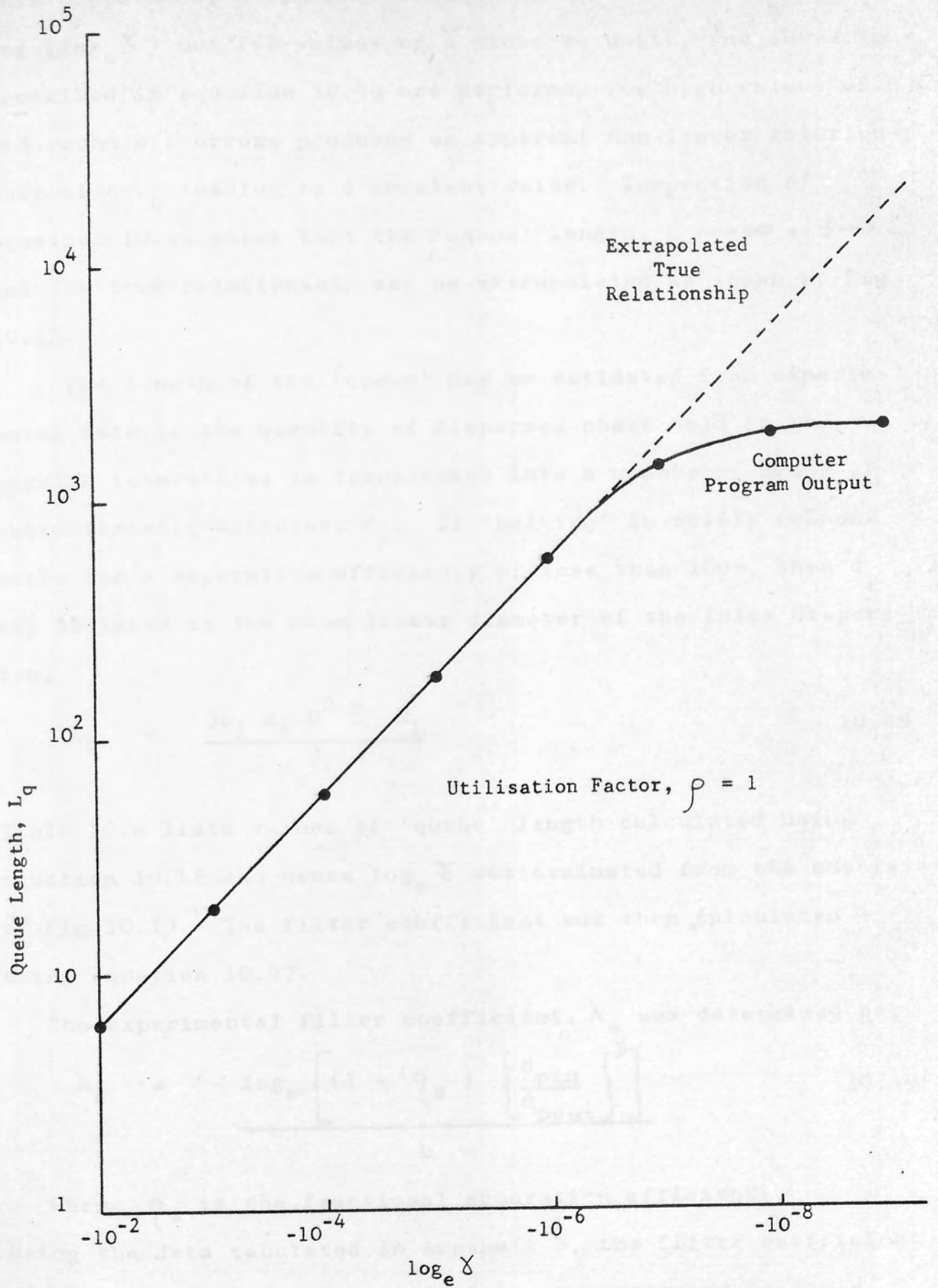


Fig 10.13. Variation of Steady State Queue length with degree of 'Balking'.

This graph shows a linear relationship between $\log(L_q)$ and $\log(\log_e \gamma)$ but for values of γ close to unity, the summations contained in equation 10.44 are performed for high values of n and round off errors produced an apparent non-linear relationship with L_q tending to a constant value. Inspection of equation 10.44 shows that the 'queue' length, $L_q \rightarrow \infty$ as $\gamma \rightarrow 1$ and the true relationship may be extrapolated as shown by Fig 10.13.

The length of the 'queue' may be estimated from experimental data if the quantity of dispersed phase held in the packing interstices is transformed into a number of drops of characteristic diameter, d_p . If 'balking' is solely responsible for a separation efficiency of less than 100%, then d_p may be taken as the mean linear diameter of the inlet dispersion,

$$L_q = \frac{3e_1 d_c D^2 \bar{S} N_L}{d_p^3} \quad 10.48$$

Table 10.6 lists values of 'queue' length calculated using equation 10.48 and hence $\log_e \gamma$ was evaluated from the abscissa of Fig 10.13. The filter coefficient was then calculated using equation 10.47.

The experimental filter coefficient, λ_e was determined as,

$$\lambda_e = \frac{-\log_e \left[(1 - \eta_s) \left\{ \frac{d_{pin}}{d_{pout}} \right\}^3 \right]}{L} \quad 10.49$$

where η_s is the fractional separation efficiency.

Using the data tabulated in Appendix H, the filter coefficient values obtained from equation 10.49 are presented in Table 10.6. Agreement between experimental and calculated filter coefficients within one order of magnitude was obtained for low velocities although experimental values were consistently lower. Above the critical velocity for this packing, the

Velocity $u(x10^{-2} \text{ m/s})$	Number Layers, N_L	Numberdrops in queue, L_q ($x10^6$)	$\log_e \delta$	Calculated Filter Coefficient, λ_c (m^{-1})	Experimental Filter Coefficient, λ_e (m^{-1})
0.5	10	22.43	$-1x10^{-15}$	27737	6118
"	20	38.10	$-4x10^{-16}$	14753	3418
"	30	53.76	$-1.8x10^{-16}$	10084	3003
"	40	65.84	$-1.45x10^{-16}$	7568	2470
"	60	98.37	$-5.7x10^{-17}$	5191	-
"	90	140.98	$-3.2x10^{-17}$	3500	702
1.5	10	9.29	$-6.5x10^{-15}$	27250	6139
"	20	15.20	$-2.3x10^{-15}$	14073	2623
"	30	19.56	$-1.6x10^{-15}$	9442	1836
"	60	37.59	$-4x10^{-16}$	4921	732
"	90	53.80	$-1.8x10^{-16}$	3361	512
"	120	70.59	$-1.2x10^{-16}$	2539	509
2.5	10	6.61	$-1.3x10^{-14}$	26671	972
"	30	14.07	$-2.3x10^{-15}$	9424	353
"	60	23.52	$-1x10^{-15}$	4799	458
"	90	35.28	$-4.2x10^{-16}$	3284	585

Table 10.6 (a) Comparison between Experimental and Predicted Filter Coefficients for S300 mesh.

Velocity $u(x10^{-2} \text{ m/s})$	Number Layers, N_L	Number drops in Queue, L_q ($x10^6$)	$\log_e \delta$	Calculated Filter Coefficient, λ_c (m^{-1})	Experimental Filter Coefficient, λ_e (m^{-1})
3.5	10	6.61	$-1.3x10^{-14}$	26671	160
"	30	12.45	$-3.2x10^{-15}$	9310	167
"	60	21.83	$-1x10^{-15}$	4820	113
"	90	29.05	$-6x10^{-16}$	3254	380
7.0	10	7.17	$-1x10^{-14}$	26968	-
"	30	11.38	$-4x10^{-15}$	9237	263
"	60	19.98	$-1.6x10^{-15}$	4715	242
"	90	27.21	$-7x10^{-16}$	3238	241
"	120	31.36	$-5.5x10^{-16}$	2442	191

Table 10.6(b) Comparison between Experimental and Predicted Filter Coefficients for S300 mesh.

agreement rapidly deteriorates due to low separation efficiencies.

The only factor in the queueing model which depends on superficial velocity is the average saturation term of equation 10.48 but this alone does not adequately account for the effect of velocity. Inclusion of the phenomenon of 'reneging' into the model which corresponds to drops which join the 'queue' but leave due to redispersion may improve the prediction of filter coefficients. In Appendix B, it was demonstrated that the critical drop diameter for release is very sensitive to velocity so that the d_p term of equation 10.48 could be replaced by d_{pc} given by equation B.1.

Alternatively a velocity dependant term could be introduced into the expression postulated for 'customer impatience' (equation 10.30).

Also, the model assumes a step change in saturation which is an oversimplification since the longitudinal variation of saturation would suggest that the analogy be better represented by a system of multiple channels in series. However, this statistical approach to determination of the filter coefficient yields results which are within the range of values predicted by previous models as shown in Section 10.1 and therefore refinement of the model by further work is justified.

... particles, containing ...
... was ...
... for investigation of the effects of operating ...
... had the ...
... secondary ...
... preferred to ...
... to ...

... measurements of ...
... a ...
... of the ...
... this ...
... 1.5×10^{-2} ...
... increased this ...
... 10^{-2} ...
... shorter ...
... operation ...

CONCLUSIONS

1. A theoretical comparison of ...
... demonstrated that direct and indirect ...
... significant under operating conditions ...
... study but diffusion and ...
... predominant for drops of ...
... forces cannot be neglected because they ...
... collapse ...
... efficiency ...
... may be estimated using

$$\eta = \frac{1}{1 + \frac{1}{2} \left(\frac{1}{\text{St}} \right)^2}$$

1. Packings, consisting of varying numbers of fine woven meshes in a series arrangement, provided suitable media for investigation of the effects of operating conditions and bed properties on the coalescence of secondary dispersions. Stainless steel meshes are preferred to nylon since the latter was found to be susceptible to swelling in aqueous environments.

2. Measurements of separation efficiency and coalesced drop sizes indicated the existence of a critical velocity above which breakthrough of the dispersion occurred. For beds initially free from dispersed phase, this velocity was found to be approximately 1.5×10^{-2} m/s. Presoaking the packings in dispersed phase increased this value to 3.5×10^{-2} m/s without increasing the two phase pressure drop. The shorter periods required to attain steady state operation emphasize the desirability of the presoaking operation.

3. A theoretical comparison of drop capture mechanisms demonstrated that direct and indirect interception are most significant under operating conditions encountered in this study but diffusion and London-Van der Waals forces may predominate for drops of less than $1\mu\text{m}$ diameter. Gravity forces cannot be neglected because they increase as coalescence produces larger drops. The drop capture efficiency, for beds initially free from dispersed phase, may be estimated using

$$\eta_{GI} = \frac{1}{2A} \left[2(1+N_R) \log_e(1+N_R) + \frac{1}{(1+N_R)} - (1+N_R) \right] + N_G$$

Application of this equation to typical operating conditions show that the drop capture rate is likely to be highest in the forepart of the bed.

4. The single phase pressure drop results were satisfactorily correlated by the Blake-Kozeny equation,

$$\left[\frac{\Delta P_1}{\mu_c} \right] \left[\frac{e_1^3}{(1-e_1)^2} \cdot \frac{d_c^2}{16N_L BT} \right] = Ku$$

Comparison of experimental values of the Kozeny constant, K, with various models describing flow through porous media showed that Keller's equation provided the best description for $0.4 < e_1 < 0.7$

$$K = \frac{e_1^2}{T(1-e_1)A}$$

where $A = 1.257 \left[1 - \left(\frac{d_c}{d_c + d_a} \right) \right]^{2.5}$

5. Two phase pressure drop results expressed as $\frac{\Delta P_2}{\Delta P_1} \left[\frac{\mu_c}{\mu_d} \right]$ to compensate for temperature and variations in single phase permeability, caused by the packing technique, were correlated with superficial velocity and bed depth,

$$\frac{\Delta P_2}{\Delta P_1} \left[\frac{\mu_c}{\mu_d} \right] = C u^a L^b$$

The values of the constant, C, and exponent, b, varied considerably with mesh type and saturation history but the data from all experiments indicated that ΔP_2 is proportional to \sqrt{u} .

6. An expression, based on the Blake-Kozeny equation, was derived to predict the two phase pressure drop from knowledge of the saturation profile,

$$\frac{\Delta P_2}{\Delta P_1} = \frac{d_c^2}{16(1-e_1)L} \int_0^L f(s)dl$$

A model was postulated from inspection of experimentally determined saturation profiles from other studies in which the local saturation decays exponentially from a constant value near the inlet face to a constant value in the downstream part of the bed,

$$S = S_I \quad 0 < l \leq L_I$$

$$S = (S_I - S_E)e^{-k(1-L_I)} + S_E \quad L_I \leq l \leq L$$

The parameters of this model were determined from average saturation data and used in the evaluation of $\int_0^L f(s)dl$. Specification of the inlet saturation, S_I , for a given superficial velocity produced good agreement between the predicted and experimental pressure drop. Saturation profiles inferred in this way were consistent with previous experimental results.

7. A preliminary study has been undertaken to investigate the feasibility of describing coalescence in a packed bed by analogy with a multiple channel queueing system. The saturation in the bed was interpreted as a queue of discrete drops waiting to coalesce into a continuum and uncaptured drops considered as 'impatient customers' at the service facility (coalescence site). An expression for the queue length was derived and through the use of average saturation data, the filter coefficient was predicted. Reasonable agreement for low superficial velocities was obtained but at higher velocities, predicted values were consistently higher than experimental values. Therefore, although refinement of the model is necessary to increase the

dependance on velocity, this novel approach is concluded to be valuable since the predicted values of filter coefficient lie within the range of values calculated from previous models.

8. A holographic technique has been successfully developed to measure drop sizes of dilute, secondary dispersions and hence to determine the filter coefficient. The method may be employed to characterise both O/W and W/O dispersions and in situ measurement eliminates sampling errors. The three-dimensional properties of images generated by reconstructed holograms permit storage of large quantities of data on a single photographic plate which may be examined at leisure. The technique also has considerable potential for automatic analysis of the reconstructed images by continuous scanning.

Measurements made using this technique showed that the diameter of secondary drops in the effluent remained fairly constant and similar to the mean diameter of the inlet dispersion. This suggests that some secondary drops pass through the bed without interception rather than being generated by redispersion phenomena. A quantitative analysis of the critical diameter for drop release supported this conclusion.

9. Based on this theoretical and experimental study, a mechanism of coalescence of secondary dispersions is proposed;

Drops are captured by direct and indirect interception, primarily in the forepart of the bed. The drops reside in the bed, either attached to fibres or intercepted by pores, where coalescence occurs until they attain such a size when

hydrodynamic forces exceed the restraining interfacial tension and adhesion forces. The drops are then squeezed through the packing interstices into a dispersed phase continuum. The dispersed phase is then conveyed in rivulets, within fixed channels, to the exit face where drops are formed and released by the mechanism of 'ballooning' (drip-point).

The study has been limited to the effects of superficial velocity and bed depth, filter diameter and pressure drop. It is recommended that the range of superficial velocity and aperture diameter be extended to determine the optimum conditions under which large drops are filtered. Since high drop velocities are obtained using fine filters and large drops are produced by media with wide apertures, it would be useful to perform experiments where the mass flow rate is varied through the bed.

The effects of other properties should also be investigated using other suitable dispersed phases.

8

RECOMMENDATIONS FOR FURTHER WORK

In experimental work, possible variations of liquid system properties should be considered, and it is desirable to elucidate the relationship between settling effects and the various parameters, pressure drop and filter coefficient, as well as particle size. It is necessary to determine the variability of the characteristics of the media and to check that the coalescence occurs satisfactorily. More specifically, this may be investigated by measuring the passage of a drop through a model aperture under the action of hydrodynamic forces, caused by flow of the surrounding continuous phase, for different settling situations.

Mathematical models developed in this work have been based largely on assumed relationships and it would be desirable to confirm the

1. This study has been restricted to investigation of the effects of superficial velocity and bed properties, viz., bed depth, fibre diameter and aperture diameter. It is recommended that the range of values employed for fibre and aperture diameters be extended to determine the optimum conditions where large drops are formed with a high separation efficiency. Since high drop capture efficiencies are realised using fine fibres and large drops are produced by meshes with wide apertures, it would be useful to perform experiments where the mesh size increases through the bed.

The effects of system properties should also be investigated using other suitable dispersed phases.

2. An experimental study, combining variation of liquid system properties for different mesh materials, would be desirable to elucidate the relationships existing between wetting effects and the coalescence parameters, pressure drop and filter coefficients. A more fundamental study is necessary to determine how wettability may be characterised for two phase flow in packed beds when coalescence occurs simultaneously. More specifically, this may be investigated by considering the passage of a drop through a model interstice under the action of hydrodynamic forces, caused by flow of the surrounding continuous phase, for different wetting situations.

3. Mathematical models developed in this work have been based largely on assumed saturation profiles and consequently it would be desirable to confirm the

assumptions by experiment. This could be achieved by permeability measurements over relatively deep beds of glass ballotini. Granular media may be easily characterised and it would be feasible to study the effects of particulate matter, suspended in the continuous phase, since the bed may be regenerated by fluidisation. Irrigation of the packing may also be incorporated to alleviate blocking of the pores.

4. Holography has been used successfully to determine the diameters of secondary droplets but the technique ultimately depends on analysis of photographs by manual counting. It is therefore recommended that a method be sought whereby the reconstructed images may be recorded on videotape and then automatically scanned and analysed to yield quantitative results.

An equation for the potential energy ψ in the region of the liquid is derived and is represented by Energy and Mass Conservation Equations

$$\nabla^2 \psi = -\frac{\rho}{\mu} \frac{d\mu}{dt} \quad (1)$$

where ψ is the irrotational velocity

$$\frac{d\mu}{dt} = \frac{d\mu}{dt} + \mathbf{v} \cdot \nabla \mu$$

is the surface tension of the interface between the solid phase, respectively

The term of Equation (1) is neglected in the case of a thin film

for water-hydrocarbon systems

APPENDICES

For oil

For water

For ethanol

$$1.5 \times 10^{-18}$$

Higher constants

APPENDIX A.

DETERMINATION OF THE HAMAKER CONSTANT.

An equation for calculating Hamaker's constant for a system of two immiscible liquids and a solid is presented by Sherony and Kinter¹²⁹ This equation is,

$$Q = 6 \pi r_{22}^2 (\sqrt{\sigma_d^d} - \sqrt{\sigma_c^d}) (\sqrt{\sigma_s^d} - \sqrt{\sigma_c^d}) \quad \text{A.1}$$

where r_{22} is the intemolecular distance

σ_c^d , σ_d^d and σ_s^d are the London-Van der Waal's component to the surface tensions of the continuous, dispersed, and solid phases, respectively.

The terms of Equation A.1 were obtained directly from the data of Fowkes¹³⁰,

for water-hydrocarbon systems, $6 \pi r_{22}^2 = 1.44 \times 10^{-18} \text{ m}^2$

for steel, $\sigma_s^d = 0.1080 \text{ N/m}$

for water, $\sigma_c^d = 0.0218 \text{ N/m}$

for toluene, $\sigma_d^d = \sigma_d = 0.0285 \text{ N/m}$

$$\therefore Q = 1.44 \times 10^{-18} (\sqrt{0.0285} - \sqrt{0.0218})(\sqrt{0.108} - \sqrt{0.0218})$$

Hamaker constant, $Q = 0.55 \times 10^{-20} \text{ J}$.

APPENDIX B.

CALCULATION OF CRITICAL DIAMETER FOR DROP RELEASE.

Rosenfeld⁷⁵ developed the following equation by equating the drag force to the sum of the adhesion and London Van der Waal's forces for an undeformed drop,

$$\frac{d_{pc}}{d_c} = \frac{Q^2}{144\pi^2 r_o^4} \left[\frac{3\mu_c u_I K(1 + 2\mu_c/3\mu_d)}{(1 + \mu_c/\mu_d)} - \sigma_{cd}(1 + \cos\theta_e) \right]^{-2} - 1 \quad \text{B.1.}$$

The critical drop diameter may be estimated by making the following assumptions for values of parameters in this equation. If complete non-wetting is assumed, corresponding to point contact between drop and collector, $\theta_e = 180^\circ$ and thus $\sigma_{cd}(1 + \cos\theta_e) = 0$. For a sphere-cylinder geometry, $r_o = 5 \times 10^{-10}$ m.⁷⁵ The factor, K which modifies the value of the drag force term, was introduced to account for fibre interaction.

Equation B.1 will now be applied to the data of Bitten⁶² who studied the coalescence of water drops on glass fibres. His experiments were conducted under the following conditions,

$$\mu_d = 1.0 \times 10^{-3} \text{ Ns/m}^2, \quad \mu_c = 2.5 \times 10^{-3} \text{ Ns/m}^2$$

$$d_c = 6 \mu\text{m}, \quad u_I = 1.52 \times 10^{-2} \text{ m/s}$$

$$Q = 0.2 \times 10^{-20} \text{ J (calculated using equation given in Appendix A).}$$

Insertion of these values into equation B.1 predicts a critical diameter of $30 \mu\text{m}$ which differs considerably from the observed value of $400 \mu\text{m}$. Application of this equation is therefore likely to produce an underestimation of actual critical diameters.

For packed beds, the velocity term, u_I must take account of the porosity of the packing and also the reduction of this porosity due to the dispersed phase saturation,

hence,

$$u_I = \frac{u}{e_1(1-s)}$$

For a stainless steel mesh of the type used in this experimental study,

$$d_c = 30.5 \mu\text{m}.$$

$$e_1 = 0.6995$$

A typical maximum value of saturation, S is 0.5 for low porosity beds and the maximum superficial velocity for this study was 7.0×10^{-2} m/s. The physical properties used are given in Appendices A and D. Substitution of these values into equation B.1 yields,

$$d_{pc} = 30.5 \times 10^6 \frac{(0.55 \times 10^{-20})^2}{144\pi^2 (5 \times 10^{-10})^4} \left[\frac{1.0 \times 3 \times 10^3 \times 7 \times 10^2 (1+1.12)}{0.6995 \times 0.5(1+1.68)} \right]^{-2} -1$$

hence,

Minimum critical drop diameter, $d_{pc} = 15.5 \mu\text{m}$.

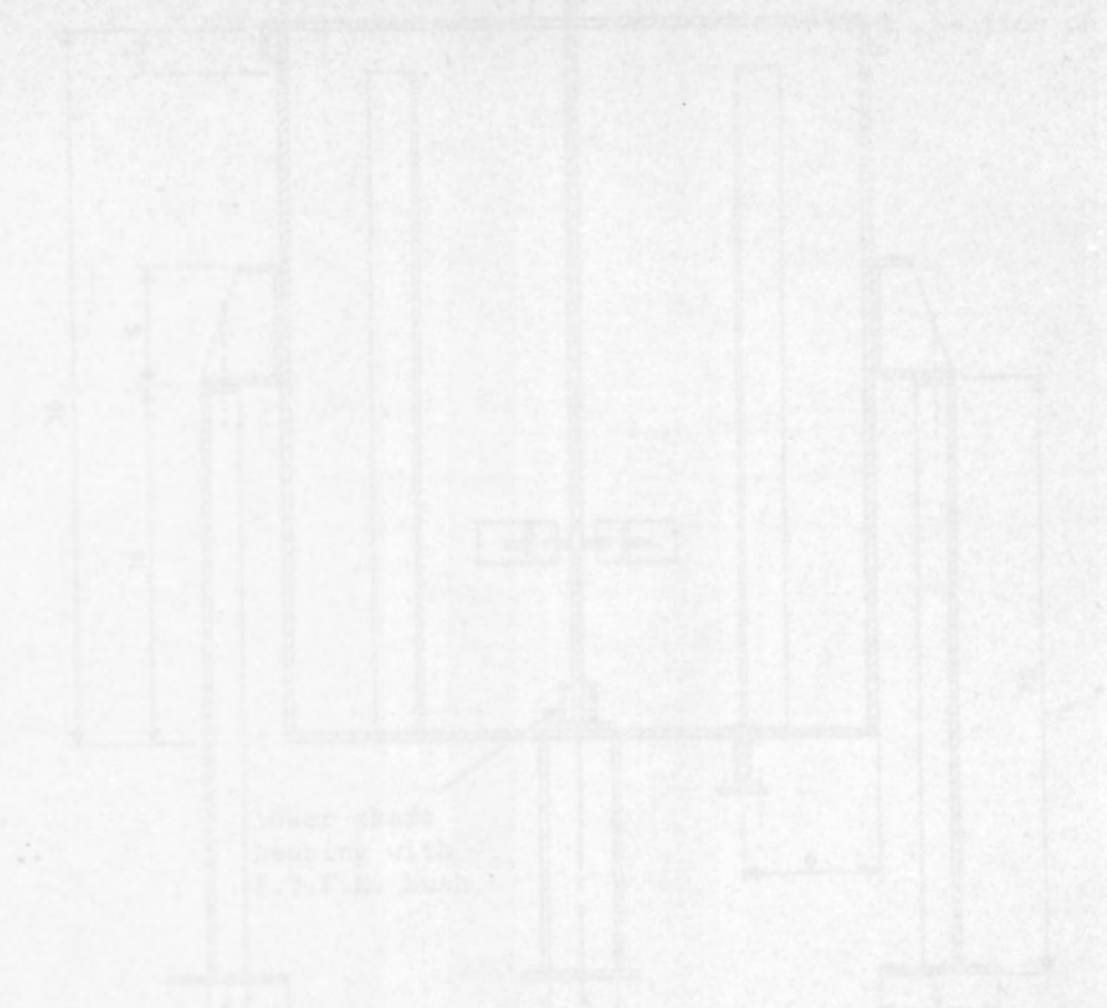
d_{pc} was calculated for superficial velocities of 0.5×10^{-2} and 5×10^{-2} m/s. to be 9.0 mm. and $59.7 \mu\text{m}$. respectively. Since the aperture diameter of this mesh is $53.0 \mu\text{m}$., these results indicate that redispersion of drops is likely for velocities greater than about 5×10^{-2} m/s. At lower velocities, drops which may become detached from a collector should be immediately recaptured by the pore catchment mechanism.

High porosity fibrous beds ($e > 0.9$) operate with a lower maximum saturation of approximately 0.2^{65} and the fibres used are typically between $1 \mu\text{m}$ and $13 \mu\text{m}$ diameter (see Chapter 4). The critical diameter, d_{pc} was estimated to be $80.8 \mu\text{m}$ using $d_c = 7 \mu\text{m}$ and $u = 5 \times 10^{-2}$ m/s. Pore size distribution data is not readily available for beds of this type and a rough estimate of the mean pore size was determined assuming an isotropic bed with the fibres

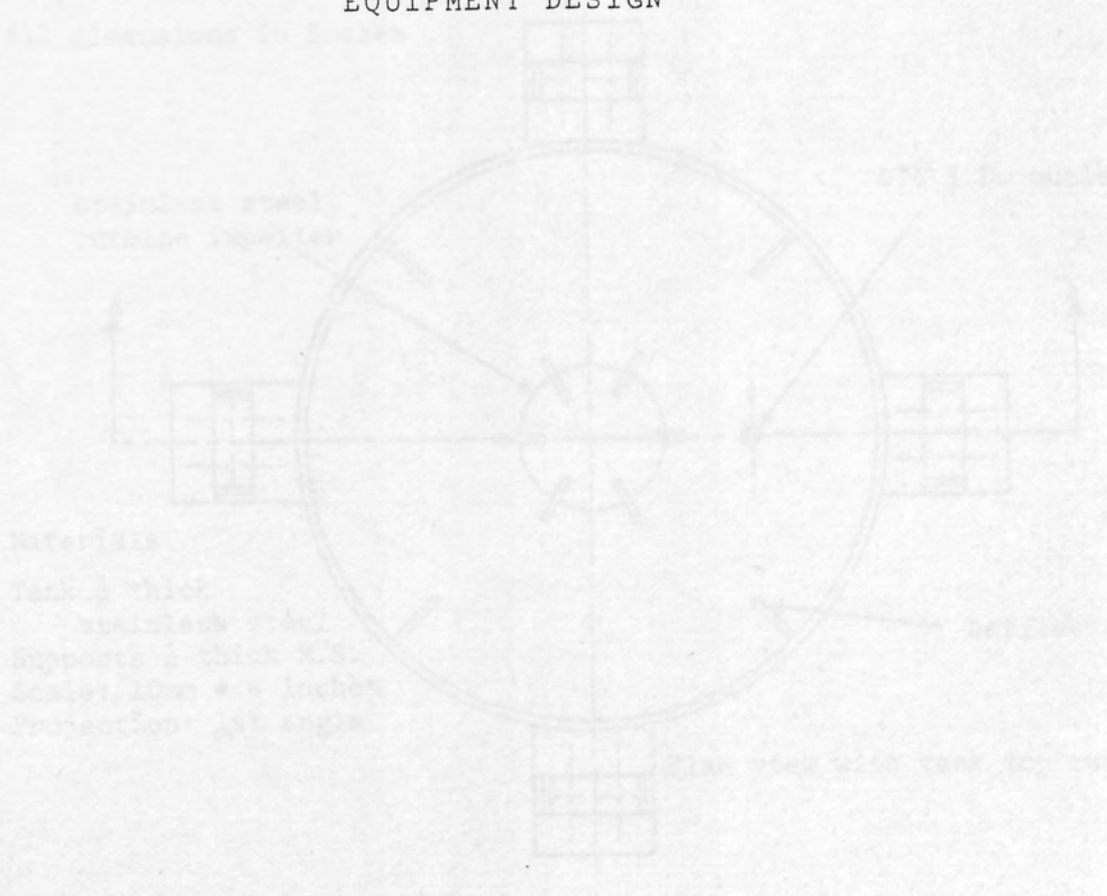
arranged in a cubic pattern. For a porosity of 0.9 and fibre diameter of $7 \mu\text{m}$, a value of $22.7 \mu\text{m}$ was obtained. When the latter is compared to a critical drop diameter of $80.8 \mu\text{m}$, it may be concluded that pore catchment is also significant in high porosity beds.

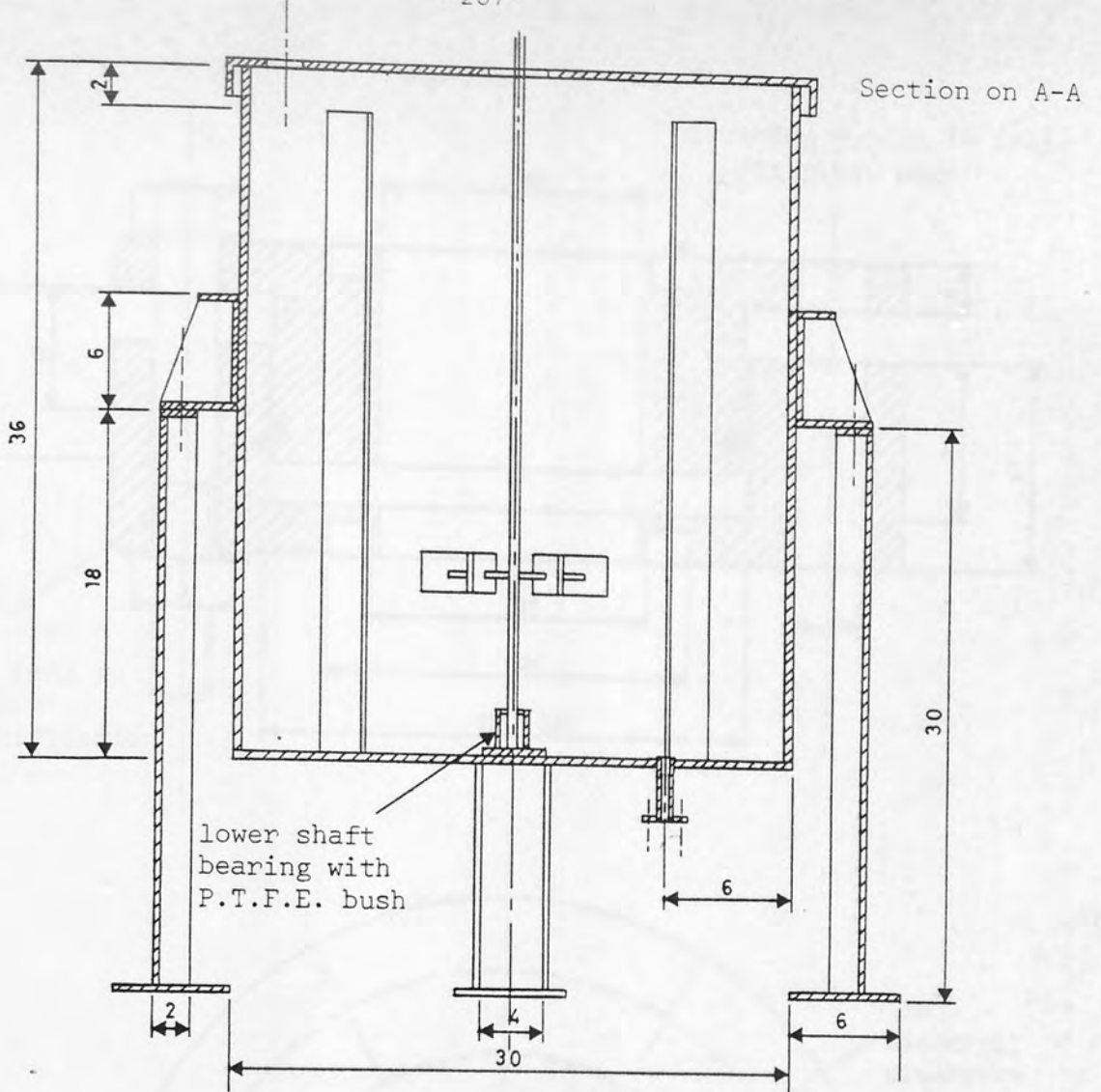
Maximum values of saturation were used for these calculations but if the variation of saturation with bed depth and superficial velocity were considered, the critical diameter would increase. Further, as drops grow by coalescence, the contact angle decreases⁹⁵ and the $\sigma_{cd}(1+\cos \theta_e)$ term of equation B.1 becomes significant which also increases the value of d_{pc} . Additional adhesion forces are created due to 'bridging' effects which have been observed¹³¹ when drops simultaneously contact two or more adjacent fibres.

The calculations completed in this section indicate that the hypothesis to justify the travelling drop model, where drops are redispersed into the flowing continuous phase, should be rejected for both low and high porosity beds.

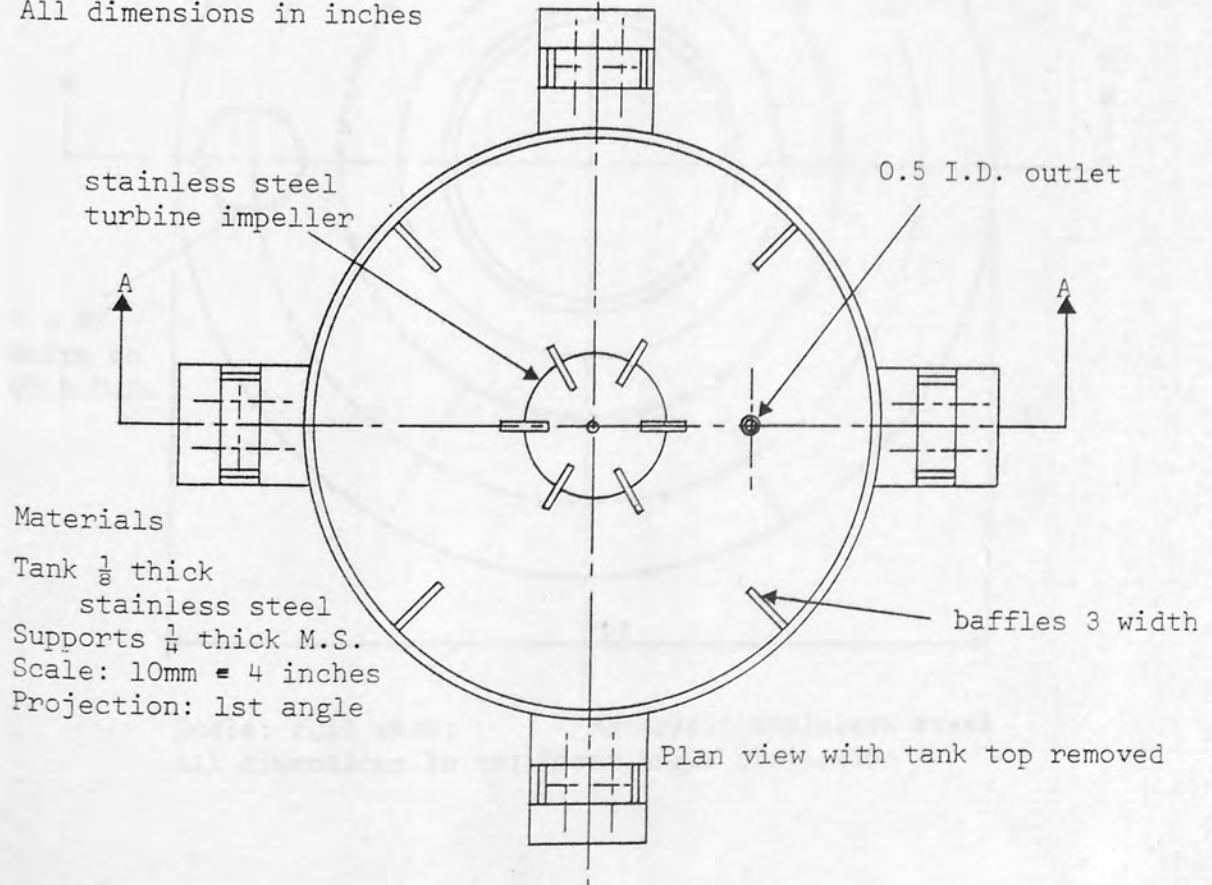


APPENDIX C
EQUIPMENT DESIGN



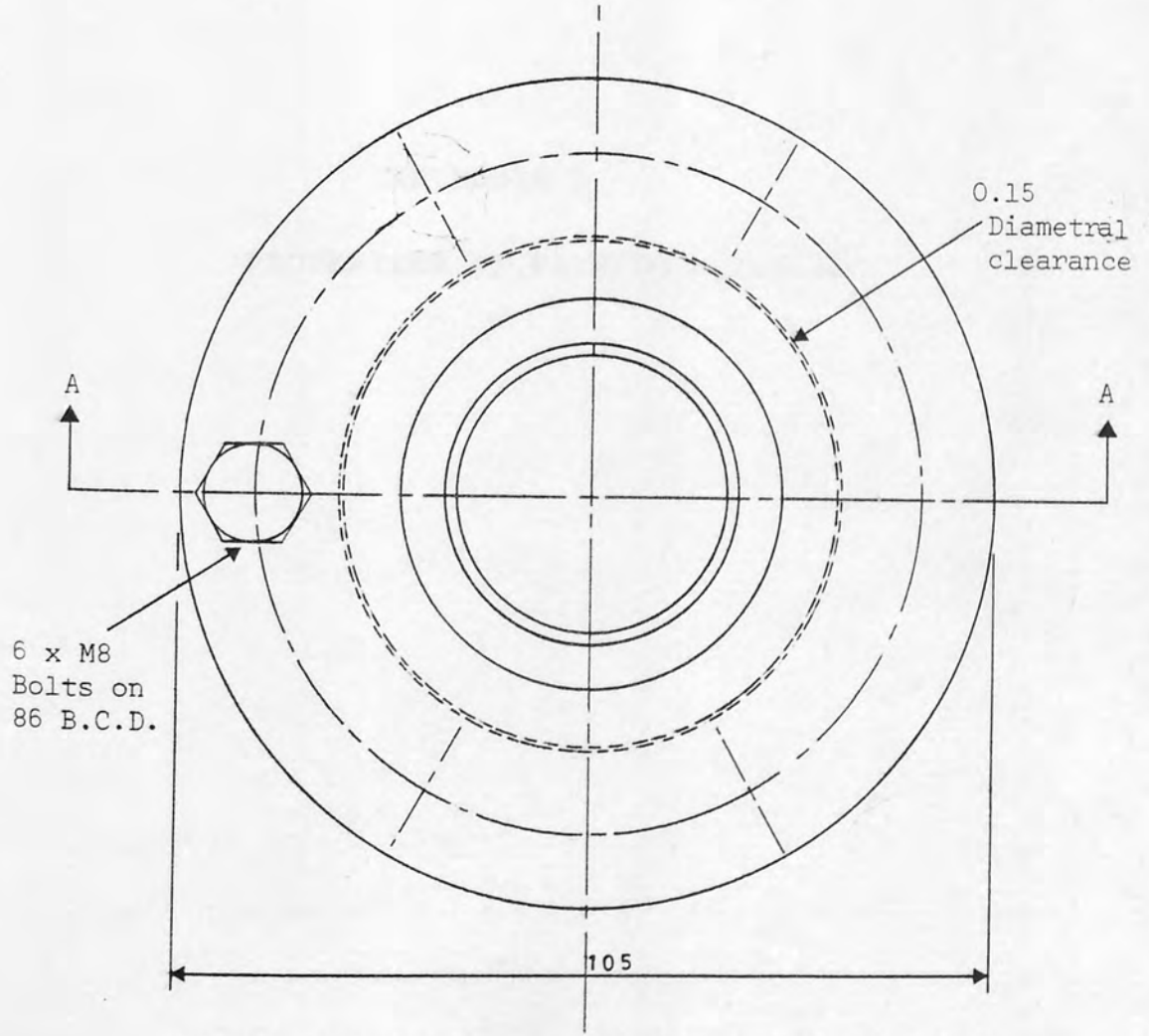
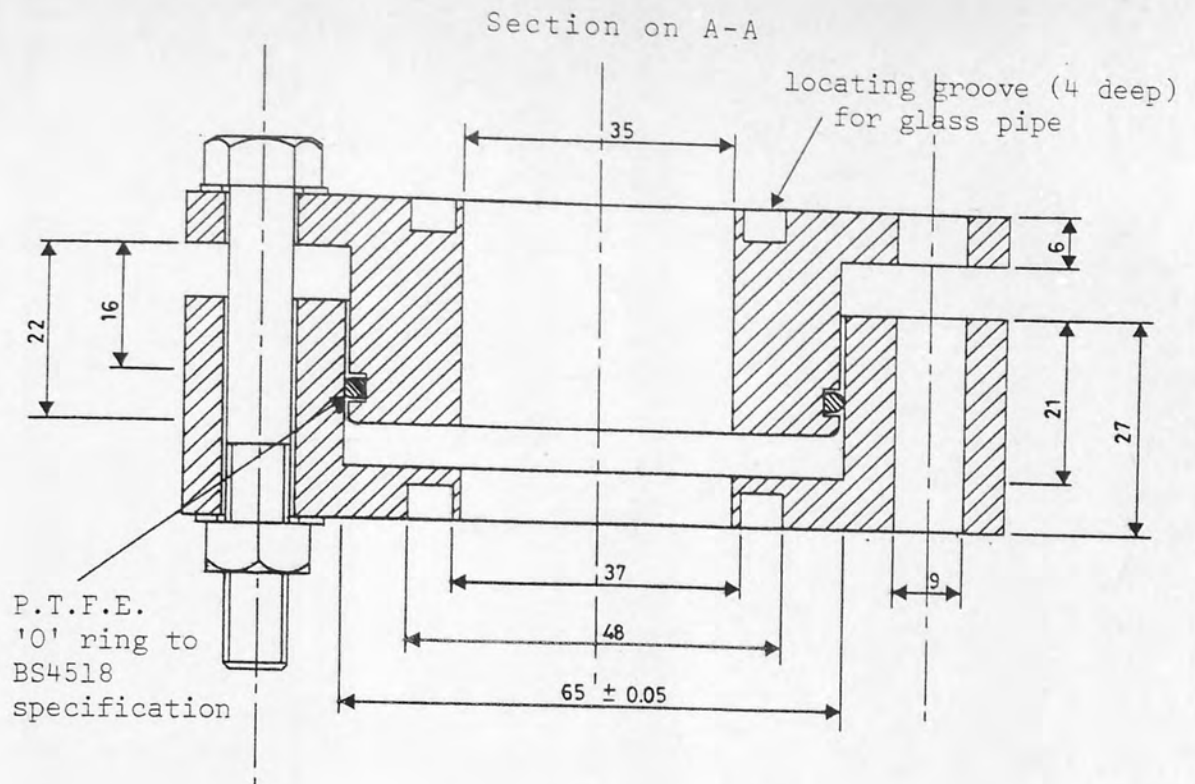


All dimensions in inches



Materials
 Tank $\frac{1}{8}$ thick stainless steel
 Supports $\frac{1}{4}$ thick M.S.
 Scale: 10mm = 4 inches
 Projection: 1st angle

Fig. C.1 Design of continuous phase reservoir



Scale: full size; Material: stainless steel
All dimensions in mm; First angle projection

Fig. C2 Design of coalescer assembly

Properties	
Specific Gravity	
Moisture Content (%)	
Compressive Strength (psi)	
Tensile Strength (psi)	
Relative Humidity	
Temperature at Breaking Point (°C)	
Moisture Absorption at 24 hrs	
Setting Point (°C)	
Following Point (°C)	
Volume Shrinkage (%)	
Resistance to acids	
Resistance to alkalis	
Resistance to organic solvents	

APPENDIX D

PROPERTIES OF PACKING MATERIALS

Table D-1. Properties of Packing Materials

This study was designed to determine the mechanical properties of various packing materials. The study was conducted under the following conditions:

The study was conducted on a variety of packing materials, including woven mesh fibers, non-woven fibers, and various types of gaskets. The results of the study are presented in Table D-1.

The study found that the mechanical properties of the packing materials varied significantly. The woven mesh fibers generally exhibited higher tensile strength and higher resistance to acids and alkalis. The non-woven fibers generally exhibited higher compressive strength and higher resistance to organic solvents.

The results of the study are summarized in Table D-1, which shows the properties of the various packing materials tested. The table includes data on specific gravity, moisture content, compressive strength, tensile strength, relative humidity, temperature at breaking point, moisture absorption, setting point, following point, volume shrinkage, and resistance to acids, alkalis, and organic solvents.

The physical and chemical properties of the materials used are listed and compared in Table D.1.

Property	Nylon (Type 66)	Polyester	Stainless Steel
Specific Gravity	1.14	1.38	7.83
Tenacity (dry)(grams/ denier)	4.0-6.5	4.0-6.5	-
Tensile Strength (dry) (kg/mm ²)	41-67	50-81	77
Relative wet strength (%)	90-95	100	100
Elongation at Breaking Point (wet) %	24-40	15-30	12
Moisture absorption at 65% RH (20°C)	3.8	0.4	0
Melting Point (°C)	240-250	250-260	1300
Softening Point (°C)	235	220-240	1300
Normal working temp. range (°C)	-40 to 115	-75 to 150	0 to 300
Recommended Max. operating temp. (°C)	100	150	300
Abrasion resistance	v.good	v.good	v.good
Resistance to acids	fair	v.good	fair
Resistance to alkalis	v.good	fair	fair
Resistance to organic solvents	good	good	v.good

Table D.1. Properties of Packing Materials. ¹³²

This study was designed to investigate the effects of mesh geometry on coalescence efficiency and pressure drop so that it was necessary to discover the extent of swelling of the nylon mesh fibres caused by water absorption. It was found that the single phase permeability of a full twill nylon mesh decreased as the immersion time in water increased as shown by Fig D.1. The initial permeability was then used to calculate the Kozeny constant as described in Section 9.3.

20 layers N20/16

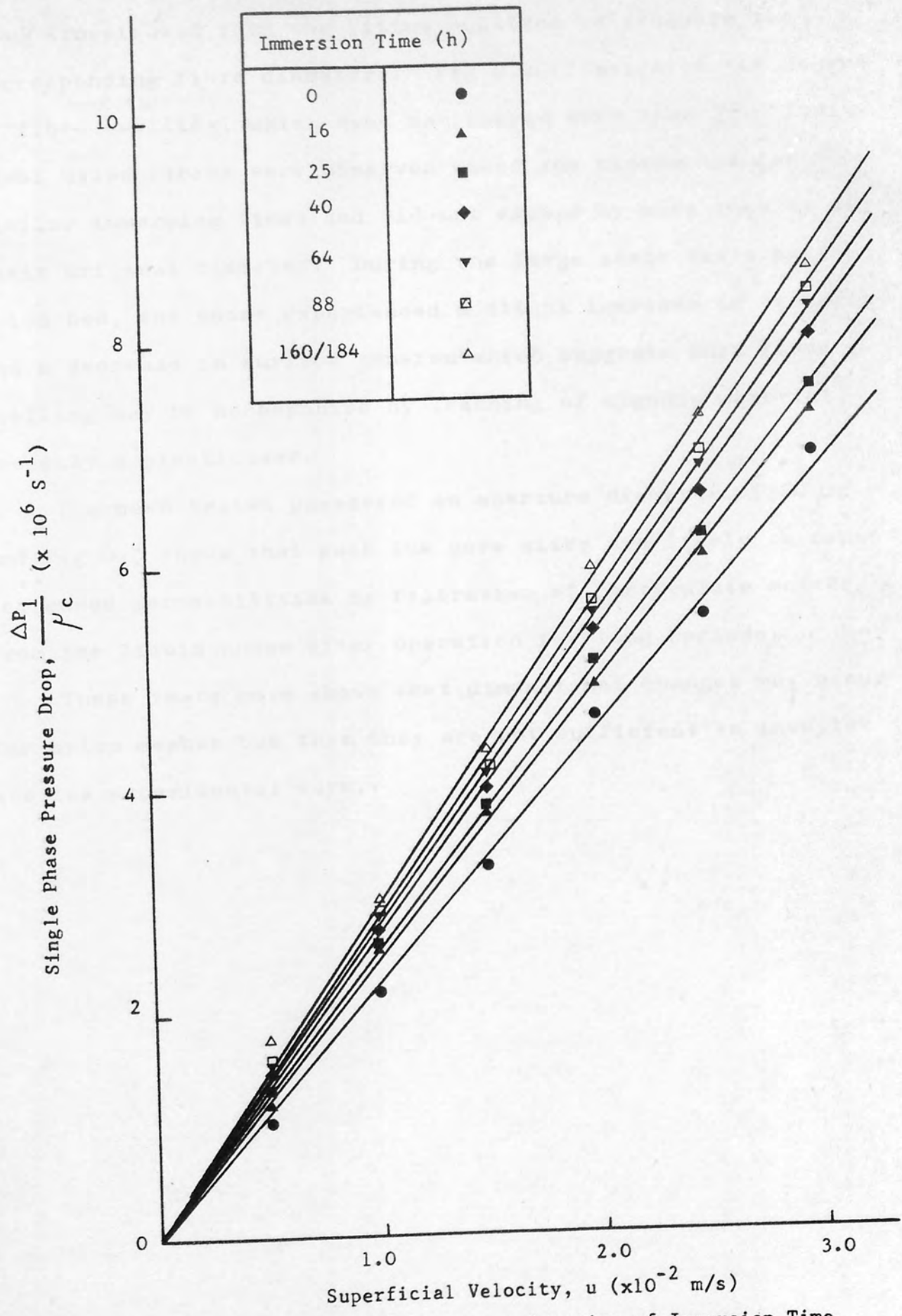


Fig D.1 Nylon mesh permeability as a function of Immersion Time.

The pressure drop data for subsequent immersion times was then substituted into the fitted equation to evaluate the corresponding fibre diameters. Fig D.2 illustrates the degree of fibre swelling, which does not exceed more than 7%. Individual nylon fibres were observed under the microscope for similar immersion times and did not expand by more than 4% of their original diameter. During the large scale tests on the nylon bed, the water experienced a slight increase in viscosity and a decrease in surface tension which suggests that fibre swelling may be accompanied by leaching of organic material, possibly a plasticiser.

The mesh tested possessed an aperture diameter of $20 \mu\text{m}$ and Fig D.3 shows that such low pore sizes are likely to cause decreased permeabilities by filtration of particulate matter from the liquid phase after operation for long periods.

These tests have shown that dimensional changes may occur for nylon meshes but that they are not sufficient to invalidate the experimental work.



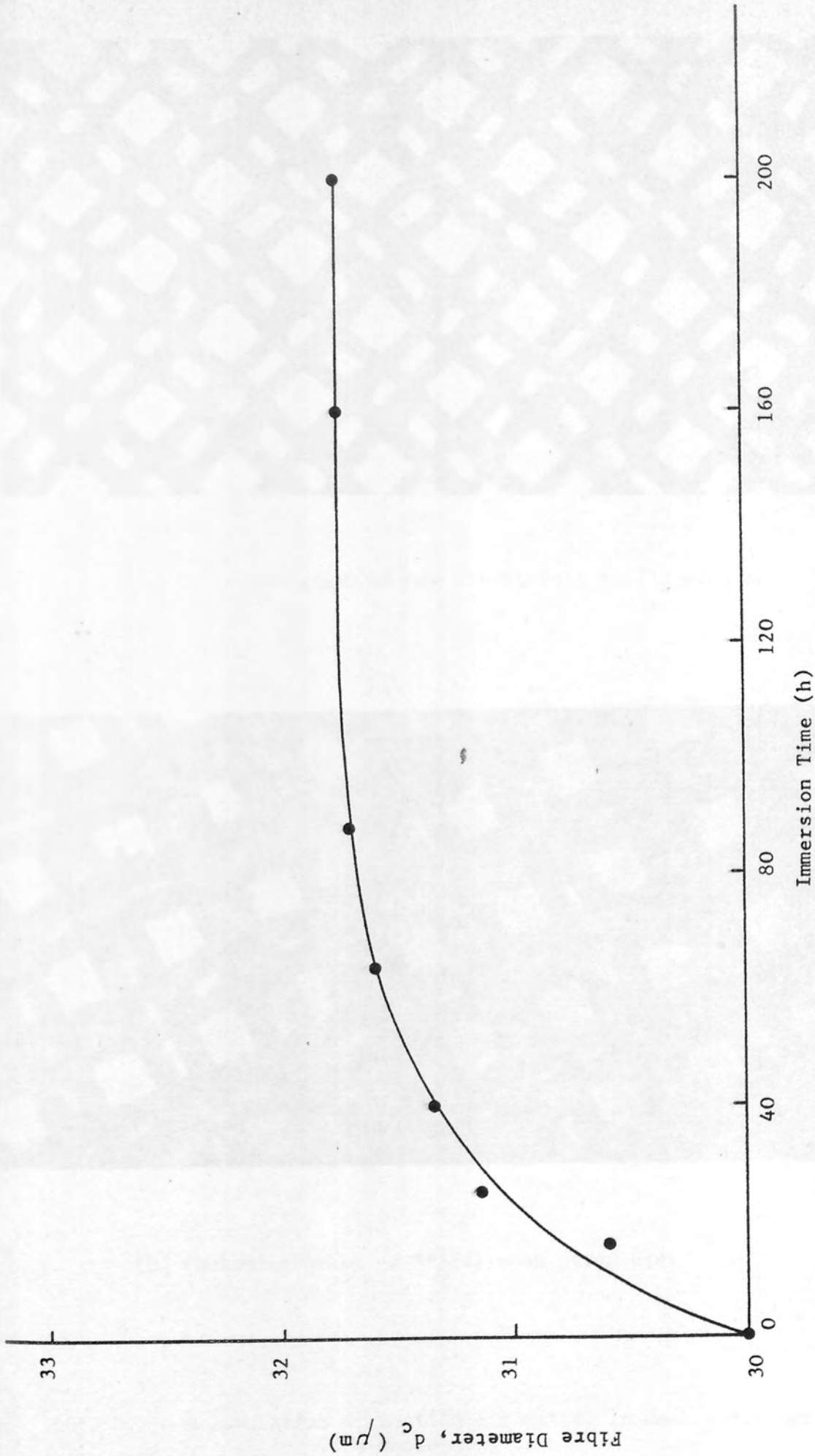
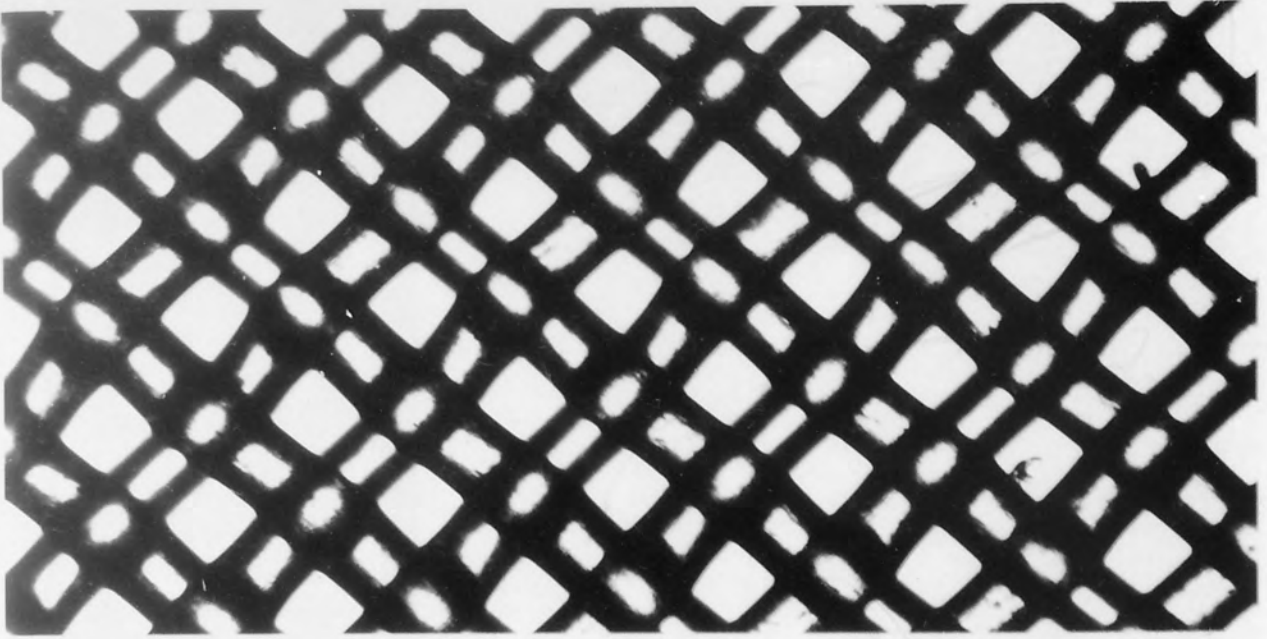
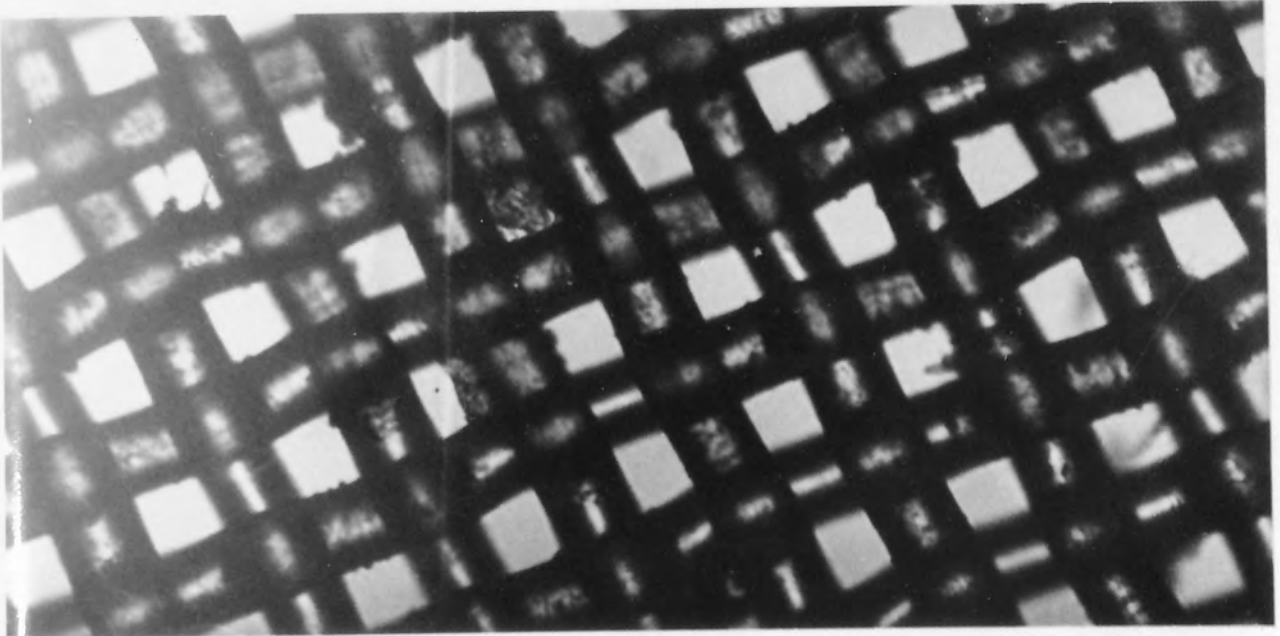


Fig D.2 Swelling of Nylon Fibres determined by Permeability Testing.



(a) Photomicrograph of new N25/21 full twill nylon mesh



(b) Photomicrograph of N25/21 mesh after use.

Fig. D.3 Accumulation of particulate matter in mesh after use
in a coalescing bed

temperature range of 20°C to 40°C. The physical properties of the liquid phases were determined as a function of temperature. The temperature coefficient of density for both components and dispersed phase is low and in the temperature range between 20°C and 40°C it is sufficient to take one value.

Density of Water, $\rho_w = 999.8 \text{ kg/m}^3$

Density of Polymer, $\rho_p = 807.2 \text{ kg/m}^3$

The remaining relevant properties are presented as a function of temperature in Figs E.1, E.2 and E.3. Viscosities and critical solubility data were extracted from the literature. Interfacial tension is sensitive to the purity of the two phases, this was measured with a Du Noüy Tensiometer. Interfacial tension is found to remain constant over the operational temperature range, as shown by Fig E.2.

APPENDIX E

PHYSICAL PROPERTIES OF LIQUID SYSTEM

To ensure equilibrium was reached, samples of the effluent were analysed using the method of extraction as described in Appendix F and the data presented in Fig E.3. Proximity of these data points to the equilibrium solubility curve shows that negligible mass transfer by dissolution or precipitation is likely, thus ensuring operation at the stated phase ratio.

As temperature changes were inevitable during operation of the coalescer, the physical properties of the liquid system were determined as a function of temperature. The temperature coefficient of density for both continuous and dispersed phases is low and in the temperature range between 10°C and 30°C, it is sufficient to take one value,

$$\text{Density of Water, } \rho_c = 998 \pm 2 \text{ kg/m}^3 \quad 133$$

$$\text{Density of Toluene, } \rho_d = 867 \pm 3 \text{ kg/m}^3 \quad 133$$

The remaining relevant properties are presented as a function of temperature in Figs E.1, E.2 and E.3. Viscosity and mutual solubility data were extracted from the literature¹³⁴ but since interfacial tension is sensitive to the purity of the two phases, this was measured with a Du Nouy Tensiometer. Interfacial tension was found to remain constant over the operational temperature range as shown by Fig E.2.

To ensure that the phases were mutually saturated, analysis of the effluent samples was completed using the method of extraction as described in Appendix F and the data presented on Fig E.3. Proximity of these data points to the equilibrium solubility curve shows that negligible mass transfer by dissolution or precipitation is likely, thus ensuring operation at the desired phase ratio.

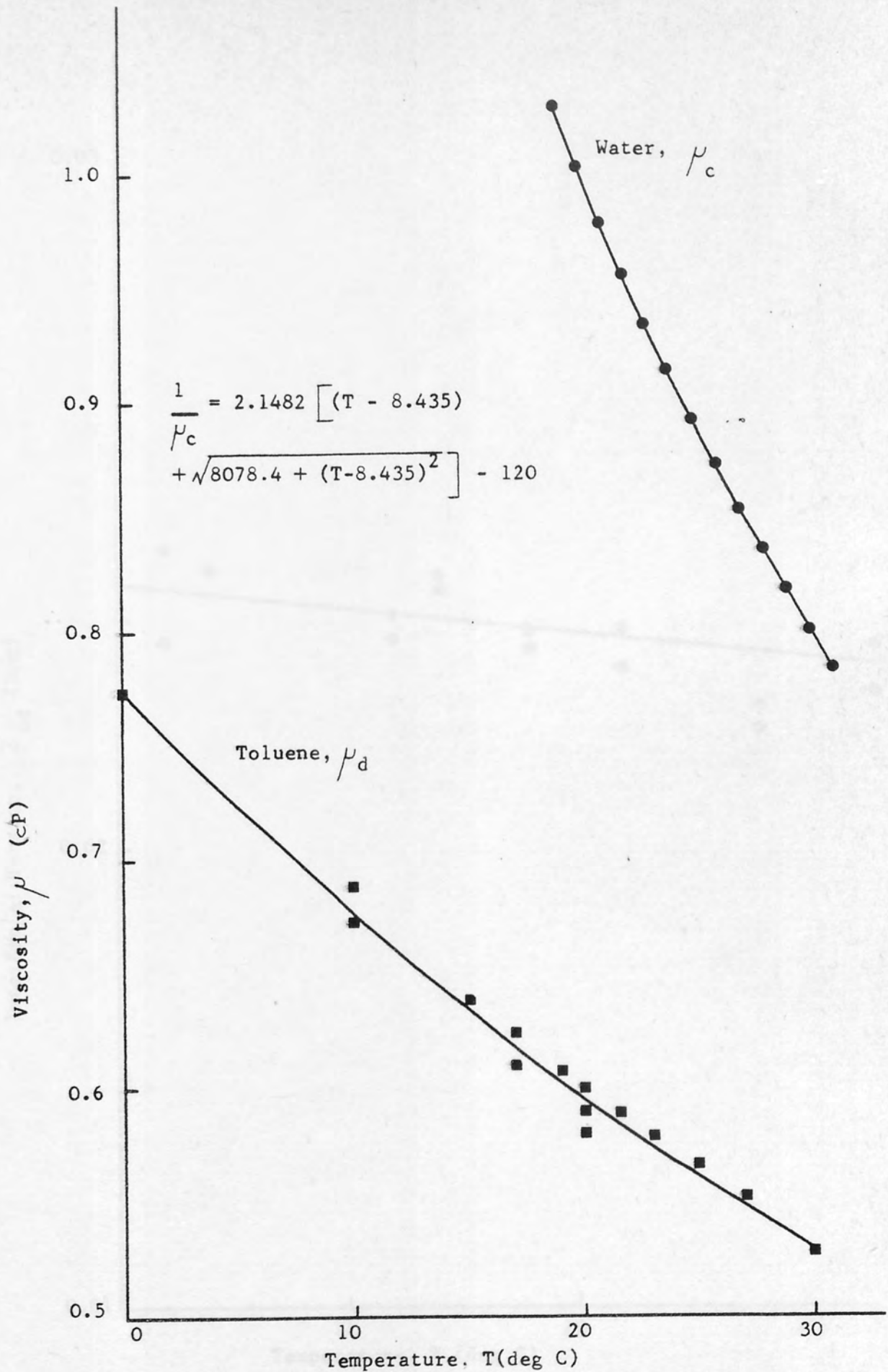


Fig E.1 Phase Viscosities as a function of Temperature.

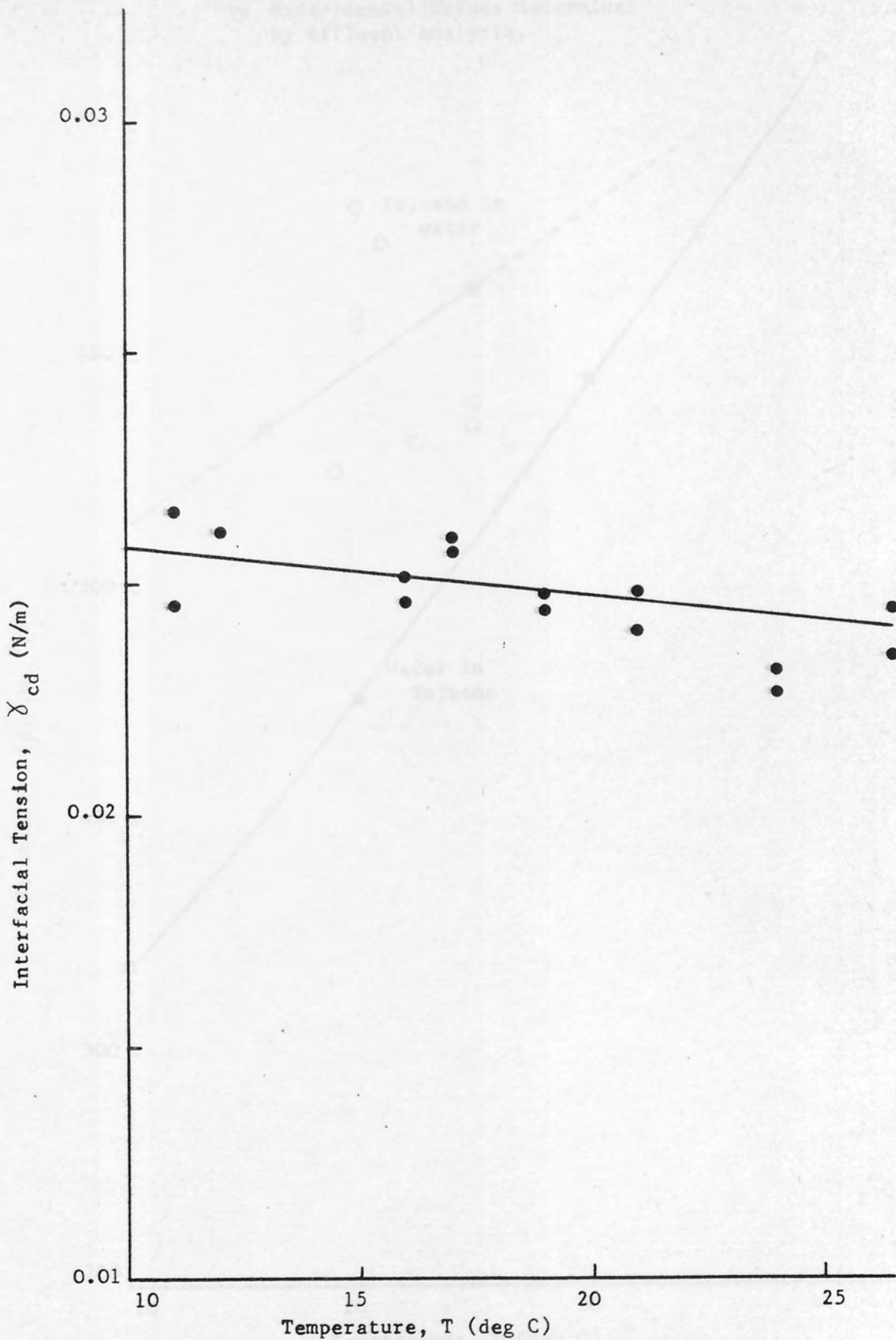


Fig E.2 Interfacial Tension for Toluene/Water system as a function of Temperature.

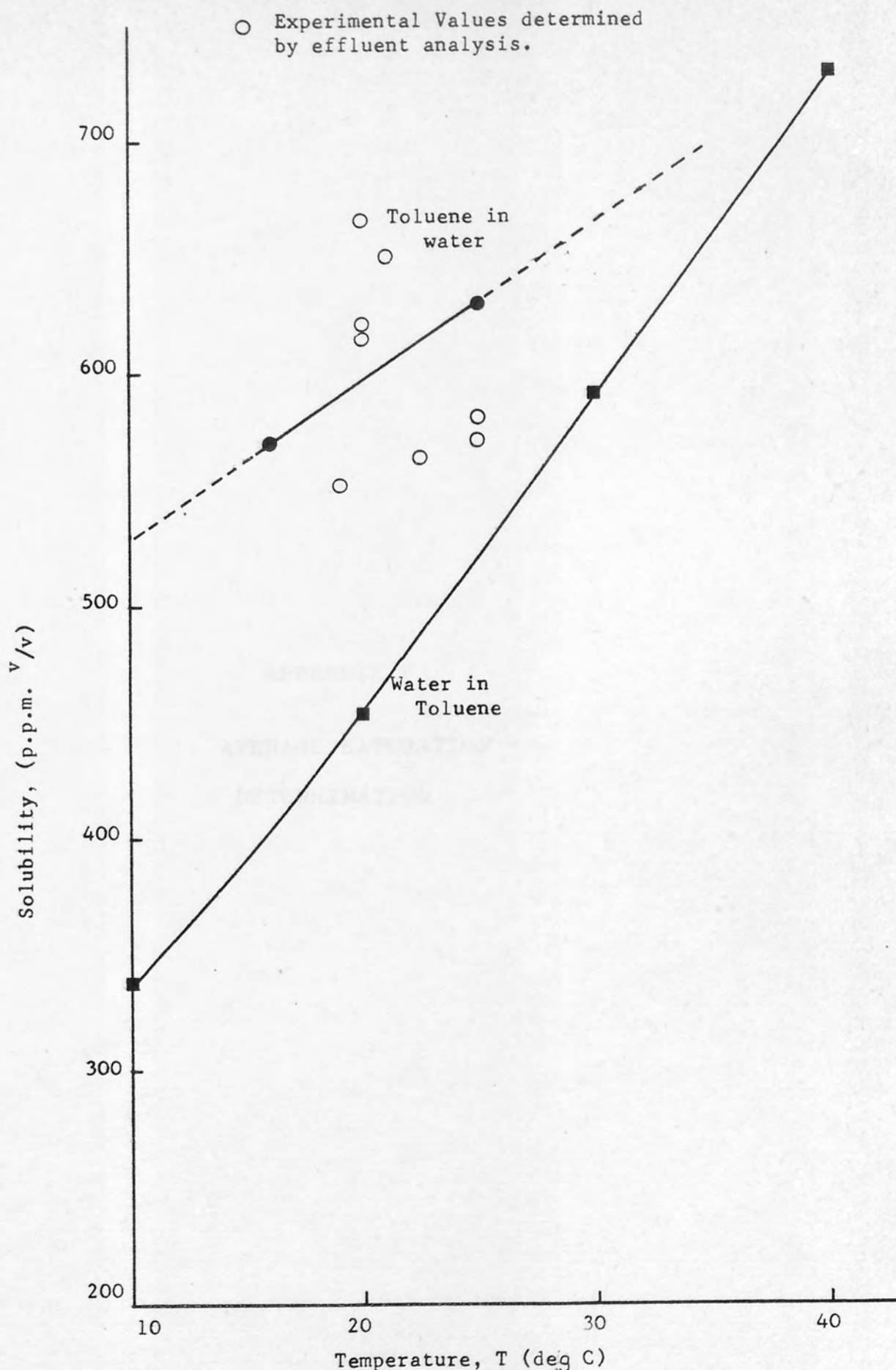


Fig E.3 Mutual Solubility of Toluene/Water System as function of Temperature.

APPENDIX F

AVERAGE SATURATION

DETERMINATION

The average saturation in the bed was determined on completion of an experimental run by measuring the quantity of toluene retained in the interstices of the packing. The coalescer assembly was dismantled and the bed rapidly immersed in 50 cm³ of ethyl acetate to extract the toluene. This procedure was repeated twice so that the dispersed phase was finally dissolved in 150 ml to produce a concentration of approximately 1% v/v. The latter was selected as a solvent since the quantity of toluene present could be determined using gas liquid chromatography under the following conditions:

Chromatograph : Pye Series 104

Column : PE G 400 on Chromosorb B

Oven temperature : 100°C

Detector : Flame Ionisation.

Carrier Gas : Nitrogen at flowrate of 40 cm³/min.

Sample volume : 0.1 μl

Repeated analysis demonstrated that the accuracy of the technique was such that average saturation values could be determined to ± 0.05 . The results are given in Appendix H.

APPENDIX G.

HOLOGRAM DEVELOPMENT PROCEDURE AND PROCESSING SOLUTIONS.

The following procedure, developed by Phillips¹¹³ was employed to produce phase holograms of the effluent dispersion. The quality of the holograms developed by the process were of a high standard thus permitting quantitative analysis of the drops by photography of the real images formed on reconstruction.

Processing Steps All at 20°C.	Time (min)	Solution Composition.
1. Wash and Harden	10	'Drysonal' ^a
2. Wash	3	-
3. Develop	8	'Neofin Blue' ^a (concentrated) +0.3mg/m ³ Benzotriazole +120mg/m ³ Sodium Metaborate.
4. Wash	5	-
5. Fix	3	Agfa Gaevert G334 solution
6. Wash	3	-
7. Bleach	10	15mg/m ³ Ferric Nitrate 3.3mg/m ³ Potassium Bromide. 0.3 mg/m ³ Phenosafranine 20 mg/m ³ Glycerol 50 mg/m ³ Propan-2-ol.
8. Wash	15	-
9. Rinse	1	'Drysonal' ^a

a - obtained from Tetrenal Photoproducts.

APPENDIX H
TABULATED EXPERIMENTAL
DATA

Mesh Type : S300		Initial Condition : Dispersed phase free						
Run No.	Temperature (deg.C)	Bed Depth	Velocity ($\times 10^{-2}$ m/s)	Average Saturation, \bar{S}	Pressure Drop $\frac{\Delta P_2}{\Delta P_1} \cdot \frac{\mu_c}{\mu_d}$	Separation Efficiency (%)	Effluent Drop Size (μ m)	Coalesced Drop Size (mm)
79	20.5	10	0.5	0.338	62.744	96.0	28.6	3.10
80	19.0	20	"	0.287	49.010	96.9	30.4	2.88
38	18.5	30	"	0.270	24.844	98.3	38.7	2.95
19	20.0	40	"	0.248	38.338	99.0	-	3.02
29	20.0	60	"	0.247	17.403	-	-	3.00
46	18.0	90	"	0.236	13.561	95.5	31.0	1.06
73	22.5	10	1.5	0.292	16.156	94.3	41.3	0.42
75	24.0	20	"	0.239	17.408	95.6	31.6	0.52
39	19.5	30	"	0.205	13.451	96.2	29.9	0.82
32	15.0	60	"	0.197	8.219	93.0	30.6	0.45
74	19.5	90	"	0.188	5.639	96.2	35.9	0.57
72	22.5	120	"	0.185	4.640	98.7	37.8	0.49

Mesh Type : S 300		Initial Condition : Dispersed phase free									
Run No.	Temperature (deg.C)	Bed Depth	Velocity ($\times 10^{-2}$ m/s)	Average Saturation, \bar{S}	Pressure Drop $\frac{\Delta P_2 - \Delta P_1}{\mu_c \mu_d}$	Separation Efficiency (%)	Effluent Drop Size (μ m)	Coalesced Drop Size (mm)			
81	21.0	10	2.5	0.258	19.166	68.3	27.5	0.69			
40	23.5	30	"	0.183	10.133	70.9	27.2	0.75			
33	15.0	60	"	0.153	6.453	79.4	34.2	0.51			
48	21.0	90	"	0.153	4.397	81.1	55.4	0.52			
82	19.5	10	3.5	0.258	13.990	44.5	28.1	0.58			
41	26.0	30	"	0.162	7.860	48.9	29.3	0.60			
34	26.0	60	"	0.142	6.022	47.3	30.7	0.60			
49	22.0	90	"	0.126	4.082	50.6	52.5	0.63			
77	28.5	10	7.0	0.280	6.065	28.0	-	0.62			
83	20.0	30	"	0.148	5.318	19.2	36.2	0.50			
71	24.5	60	"	0.130	4.061	22.2	40.9	0.36			
84	21.5	90	"	0.118	3.962	20.5	47.7	0.36			
76	27.0	120	"	0.102	2.850	23.2	48.3	0.48			

Initial Condition : Presoaked in dispersed phase.

Mesh Type : S 300

Run No.	Temperature (deg.C)	Bed Depth	Velocity ($\times 10^{-2}$ m/s)	Average Saturation, \bar{S}	Pressure Drop $\frac{\Delta P_2}{\Delta P_1} \cdot \frac{\mu_c}{\mu_d}$	Separation Efficiency (%)	Effluent Drop Size (μ m)	Coalesced Drop Size (mm)
42	19.5	30	0.5	0.405	22.312	99.8	27.7	4.00
31	17.0	60	"	0.274	12.750	101.6 ^a	42.9	3.72
50	18.0	90	0.5	0.326	27.253	99.0	78.1	3.07
43	21.5	30	1.5	0.345	12.487	96.8	31.0	3.61
35	20.0	60	"	0.243	11.136	97.1	34.1	3.50
51	18.0	90	"	0.282	5.319	97.6	-	2.87
44	26.0	30	2.5	0.331	10.367	94.4	33.5	2.18
36	16.0	60	"	0.200	7.627	93.9	36.2	2.44
52	21.0	90	"	0.230	3.800	95.8	-	2.27
45	27.0	30	3.5	0.330	8.479	49.4	31.0	0.82
37	15.5	60	"	0.198	7.794	53.6	33.3	1.73
53	27.0	90	"	0.195	3.873	82.4	33.3	1.32

a - value 100% attributable to measurement error

Mesh Type : S300										
Run No.	Temperature (deg.C)	Bed Depth	Velocity ($\times 10^{-2}$ m/s)	Average Saturation, \bar{S}	Pressure Drop $\frac{\Delta P_2}{\Delta P_1} \frac{\mu_c}{\mu_d}$	Separation Efficiency (%)	History ^a (Previous Velocity)	Coalesced Drop Size (mm)		
19	20.0	40	0.5	0.248	38.338	99.0	0 (free)	3.02		
	19.5	"	2.5	-	10.432	96.3	0.5	1.60		
	27.0	"	3.5	-	9.029	96.5	2.5	0.87		
	28.5	"	1.5	-	11.013	98.1	3.5	3.18		
	25.5	"	3.5	-	8.167	95.2	1.5	0.67		
	20.0	"	0.5	-	18.126	99.6	3.5	3.52		
	20.0	"	0.5	-	16.762	97.9	0.5	3.23		
	24.0	"	1.5	-	9.596	97.0	0.5	1.27		
	23.0	"	3.5	-	8.160	94.3	1.5	0.83		
	22.5	"	2.5	-	8.468	95.5	3.5	1.24		

a - Velocity changed during one run.

Mesh Type : Polyester/Nylon.									
Run No	Temperature (deg.C)	Bed Depth.	Velocity ($\times 10^{-2}$ m/s)	Average Saturation, \bar{S}	Drop Pressure $\frac{\Delta P_2}{\Delta P_1} \frac{\mu_c}{\mu_d}$	Separation Efficiency (%)	Initial Condition	Coalesced Drop Size (mm)	
PE 53/33									
17	19.0	40	0.5	-	25.209	97.2	free ^a	7.44	
	17.0	"	1.5	-	14.133	99.0	-	3.95	
	25.0	"	2.5	-	7.531	96.1	-	3.24	
	28.5	"	3.5	-	8.243	97.6	-	2.65	
N53/36									
18	18.5	40	0.5	-	23.268	99.4	free ^a	2.72	
	23.0	"	1.5	-	13.695	92.9	-	3.25	
	21.5	"	2.5	-	8.639	97.0	-	1.39	
	24.0	"	3.5	-	7.836	96.6	-	1.36	

a - velocity changed during one run.

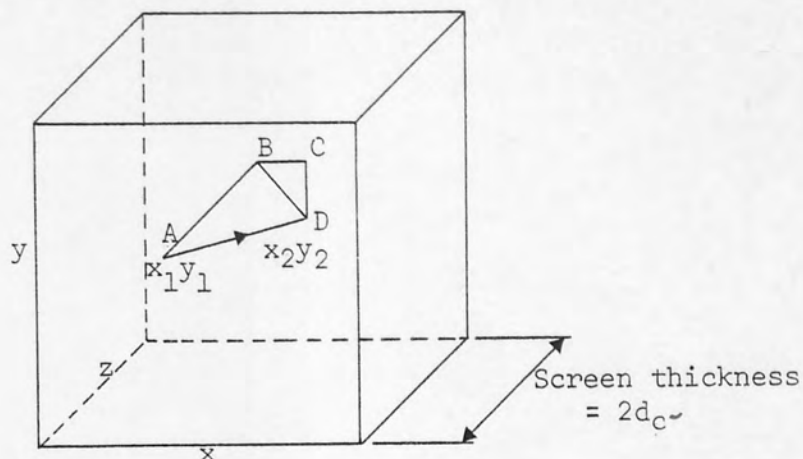
Mesh Type : N53/36

Run No.	Temperature (deg.C)	Bed Depth	Velocity ($\times 10^2$ m/s)	Average Saturation, \bar{S}	Pressure Drop $\frac{\Delta P_2}{\Delta P_1} \cdot \frac{\rho_c}{\rho_d}$	Separation Efficiency (%)	Initial Condition	Coalesced Drop Size (mm)
18	21.0	40	0.5	-	23.268	100.9	free	2.72
	"	"	1.5	-	13.695	97.4	-	3.25
	"	"	2.5	-	8.639	98.8	-	1.39
	"	"	3.5	-	7.837	95.0	-	1.36
64	20.0	60	0.5	0.101	10.077	98.9	free	3.93
54	19.5	"	1.5	0.076	7.189	99.1	free	2.73
66	20.0	"	2.5	0.060	4.463	97.5	free	1.18
68	25.0	"	3.5	0.054	4.580	93.6	free	0.75
65	20.0	"	0.5	0.125	10.587	96.3	presoaked	3.35
55	21.0	"	1.5	0.105	6.258	97.2	presoaked	2.13
67	22.0	"	2.5	0.078	4.903	98.7	presoaked	1.08
69	20.5	"	3.5	0.071	4.345	96.1	presoaked	0.77

Mesh Type : Nylon									
Run No.	Temperature (deg.C)	Bed Depth	Velocity ($\times 10^2$ m/s)	Average Saturation, \bar{S}	Pressure Drop $\frac{\Delta P_2 - \mu_c}{\Delta P_1 \mu_d}$	Separation Efficiency (%)	Initial Condition	Coalesced Drop Size (mm)	
N70/45									
60	20.0	60	1.5	0.074	7.764	89.6	free	2.08	
61	18.0	60	1.5	0.085	8.927	97.8	presoaked	3.10	
N69/38									
58	20.5	60	1.5	0.085	6.415	90.0	free	1.25	
59	21.5	60	1.5	0.119	6.025	98.1	presoaked	1.89	
N70/29									
62	23.0	60	1.5	0.057	7.481	95.3	free	3.54	
63	21.0	60	1.5	0.066	11.223	97.4	presoaked	3.34	
N25/18									
56	22.0	60	1.5	0.130	4.056	99.2	free	2.90	
57	21.0	60	1.5	0.218	4.220	100.6	presoaked	2.52	

APPENDIX I.

DERIVATION OF ALGORITHM FOR TORTUOSITY FACTOR.



Consider $\triangle BCD$ in xy plane

$$BD^2 = \left[(x_2 - x_1)^2 + (y_2 - y_1)^2 \right] d_a^2$$

Consider $\triangle ABD$

$$\begin{aligned} AD^2 &= AB^2 + BD^2 \\ &= (2d_c)^2 + \left[(x_2 - x_1)^2 + (y_2 - y_1)^2 \right] d_a^2 \end{aligned}$$

$$\therefore AD = d_a \left[4F^2 + (x_2 - x_1)^2 + (y_2 - y_1)^2 \right]^{0.5}$$

$$\text{where } F = \frac{d_c}{d_a}$$

Then, defining a Tortuosity factor, T as,

$$T = \left(\frac{AD}{2d_c} + 1 \right)$$

$$T = 0.5 \left\{ \frac{\left[4F^2 + (x_2 - x_1)^2 + (y_2 - y_1)^2 \right]^{0.5} + 1}{2F} \right\}$$

The Tortuosity factor, T was calculated by insertion of the co-ordinates (x_1, y_1) and (x_2, y_2) from a random number generator (see Appendix J). This procedure was repeated a number of times, corresponding to the number of layers of mesh used in the coalescer. This was an attempt

to simulate the geometry of the bed which was prepared by laying single meshes, one upon another, so that their relative orientations and positions were fairly random.

APPENDIX 4

COMPUTER PROGRAM FOR ANALYSIS
OF SINGLE PHASE PRESSURE DROP
DATA

APPENDIX J

COMPUTER PROGRAM FOR ANALYSIS
OF SINGLE PHASE PRESSURE DROP
DATA

```

20 REM CORRELATION OF PRESSURE DROP DATA BY LINEAR REGRESSION
30 DIM A(16),D(250),N(20),V(50),X(600),Y(600),Z(250)
40 READ K
45 REM READ FLUID VELOCITIES
50 FOR I=1,K: READ V(I): NEXT I
60 INPUT K1
70 M=1:K3=0
100 REM INPUT MESH PARAMETERS
110 INPUT W,D1,D6,N3,K2
120 REM EVALUATION OF PACKING PROPERTIES
130 IF W=2 THEN 150
140 A3=12*3.14159*N3*SQR((N3*D6)+2+1): GOTO 160
150 A3=6*3.14159*N3*(SQR((N3*D6)+2+1)+1)
160 E=1-A3*D6/48:D2=(1/N3-D6)/12:E=D6/6
170 C1=100/(12*2.54)
180 A3=A3*C1:D2=D2/C1:B=B/C1
190 REM LOOP FOR DIFFERENT BED DEPTHS
200 FOR J=1,K2
210 REM INPUT NUMBER OF MESH LAYERS
220 INPUT N1
224 PRINT " SINGLE PHASE PRESSURE DROP DATA": PRINT
225 PRINT TAB(15);"MESH: NO. OF LAYERS:";N1: PRINT
230 REM CALCULATION OF TORTUOSITY FACTOR BY RANDOM NO. GENERATION
240 Q=0:F=D6/D1
250 A1=8*J+3:A2=8*J-3
260 X(0)=RND(A1)
270 FOR I=1,N1:X(I)=RND(0): NEXT I
280 Y(0)=RND(A2)
290 FOR I=1,N1:Y(I)=RND(0): NEXT I
300 FOR N2=2,N1
310 Q1=D1*SQR(4*F+2+(X(N2)-X(N2-1))+2+(Y(N2)-Y(N2-1))+2)
320 Q=Q+Q1: NEXT N2
330 Q=Q/(2*D6*(N1-1))
340 Q=(Q+1)/2
341 PRINT Q
345 PRINT TAB(15);" VELOCITY TEMPERATURE PRESSURE DROP"
346 PRINT TAB(15);" M/SEC. DEG.C N/M+2"
350 REM LOOP FOR CORRESPONDING PRESSURE DROPS & TEMPERATURES

```

```
360 FOR I=1,K
370 INPUT P,T
375 PRINT TAB(15);V(I),1,P
380 REM CALCULATION OF FLUID VISCOSITY
390 M1=.1/(2.1482*((1-8.435)+SQR(8078.4+(1-8.435)^2))-120)
400 Z(M)=(P/M1)*(E+3/(A3+2*N1*B*C))
410 M=M+1
420 NEXT I
425 FOR I=1,14: PRINT : NEXT I
430 REM COUNTER FOR NO. OF "Y" VALUES FOR EACH "X" VALUE
440 K3=K3+1
450 NEXT J
470 REM SEQUENCE FOR REARRANGEMENT OF DATA FOR REGRESSION
480 M=1:D(M)=V(1):J2=2
490 FOR I=1,K
500 N(I)=K3:J1=I*(1+N(I))+1
520 M=M+1
530 IF M=J1 THEN D(M)=V(I+1): GOTO 550
540 J=(M-J2)*K+I:D(M)=Z(J): GOTO 520
550 J2=J2+(1+N(I)): NEXT I
555 REM ENTER LINEAR REGRESSION ANALYSIS SUBROUTINE
560 FOR I=1,16:A(I)=0: NEXT I
560 CALL (1,K,N(1),D(1),A(1))
570 FOR I=1,16: PRINT A(I): NEXT I
571 R=SQR(1-A(9)/(A(6)-A(7))): PRINT "R=",R
575 GOTO 70
580 STOP
590 END
700 DATA 15
710 DATA .5E-02,.15E-01,.25E-01,.35E-01,.45E-01,.55E-01,.65E-01,.75E-01
720 DATA .7E-01,.6E-01,.5E-01,.4E-01,.3E-01,.2E-01,.1E-01
```

SINGLE PHASE PRESSURE DROP DATA

MESH: S300 NO. OF LAYERS: 10

VELOCITY M/SEC.	TEMPERATURE DEG. C	PRESSURE DROP N/M ²
.5E-02	18.5	78
.15E-01	18.5	195
.25E-01	18.5	313
.35E-01	18.5	438
.45E-01	18	595
.55E-01	18	736
.65E-01	18	869
.75E-01	18	1026
.7E-01	18	948
.6E-01	18	807
.5E-01	18	673
.4E-01	18	532
.3E-01	18	399
.2E-01	18	266
.1E-01	18	133

SINGLE PHASE PRESSURE DROP DATA

MESH: S300 NO. OF LAYERS: 20

VELOCITY M/SEC.	TEMPERATURE DEG. C	PRESSURE DROP N/M ²
.5E-02	19.5	141
.15E-01	19.5	368
.25E-01	20	587
.35E-01	20	814
.45E-01	20	1081
.55E-01	20	1316
.65E-01	20	1574
.75E-01	20	1833
.7E-01	20	1692
.6E-01	20	1449
.5E-01	20	1198
.4E-01	20	963
.3E-01	20	712
.2E-01	20	485
.1E-01	20	258

SINGLE PHASE PRESSURE DROP DATA

MESH: S300 NO. OF LAYERS: 60

VELOCITY M/SEC.	TEMPERATURE DEG. C	PRESSURE DROP N/M ²
.5E-02	23.5	313
.15E-01	23.5	791
.25E-01	21	1574
.35E-01	20	2225
.45E-01	19	3032
.55E-01	20	3666
.65E-01	20	4364
.75E-01	19.5	5124
.7E-01	19.5	4724
.6E-01	19.5	4050
.5E-01	19	3369
.4E-01	19	2695
.3E-01	19	1997
.2E-01	19	1331
.1E-01	19	665

SINGLE PHASE PRESSURE DROP DATA

MESH: S300 NO. OF LAYERS: 60

VELOCITY M/SEC.	TEMPERATURE DEG. C	PRESSURE DROP N/M ²
.5E-02	19.5	352
.15E-01	19.5	1042
.25E-01	19.5	1708
.35E-01	19	2421
.45E-01	19	3196
.55E-01	19	3925
.65E-01	19	4677
.75E-01	19	5508
.7E-01	19	5092
.6E-01	19	4356
.5E-01	19	3619
.4E-01	18.5	2875
.3E-01	18.5	2146
.2E-01	18.5	1449
.1E-01	18.5	705

SINGLE PHASE PRESSURE DROP DATA

MESH: S300 NO. OF LAYERS: 90

VELOCITY M/SEC.	TEMPERATURE DEG. C	PRESSURE DROP N/M ²
.5E-02	18	485
.15E-01	18.5	1457
.25E-01	18.5	2397
.35E-01	18	3361
.45E-01	17.5	4552
.55E-01	17.5	5515
.65E-01	17.5	6573
.75E-01	17.5	7623
.7E-01	18	7012
.6E-01	18	6001
.5E-01	18	4967
.4E-01	18	3925
.3E-01	18	2922
.2E-01	18	1958
.1E-01	18	987

SINGLE PHASE PRESSURE DROP DATA

MESH: S300 NO. OF LAYERS: 120

VELOCITY M/SEC.	TEMPERATURE DEG. C	PRESSURE DROP N/M ²
.5E-02	22	658
.15E-01	22	1935
.25E-01	21.5	3165
.35E-01	20.5	4458
.45E-01	20.5	5868
.55E-01	20.5	7169
.65E-01	20.5	8524
.75E-01	20.5	9903
.7E-01	20	9135
.6E-01	20	7819
.5E-01	20	6550
.4E-01	20	5194
.3E-01	19.5	3854
.2E-01	19.5	2601
.1E-01	19.5	1292


```

10  REM
20  REM
30  DIM S(45,10),H(45,10),P(45,10),R(45,10),N(10)
35  DEF FNACT=.1/(2.1482*((1-8.435)+SQK(8078.4+(1-8.435)*2))-120)
36  DEF FNBCT=.1E-02*(.771691-.963663E-02*1+.563896E-04*1*2)
37  INPUT J1
38  FOR J=1,J1
40  INPUT N(J)
50  FOR I=1,N(J)
60  INPUT S(I,J),H(I,J),P(I,J)
70  S(I,J)=S(I,J)*.1E-02
80  NEXT I
85  NEXT J
90  CALL (1)
100 CALL (2,-1,10,-1,16)
110 CALL (3,2,0,0,E)
120 CALL (3,1,9,0,E)
140 CALL (3,2,0,15,E)
150 CALL (3,1,0,0,E)
265 FOR J=1,J1
270 FOR I=1,N(J)
280 M2=FNB(H(I,J))
290 R(I,J)=(P(I,J)/M2)*.1E-05
295 IF S(I,J)>9 THEN 310
300 CALL (3,1,S(I,J),R(I,J),E)
310 NEXT I
311 CALL (3,2,0,0,E)
312 NEXT J
320 CALL (3,2,-.125,-.25,E): CALL (4,7): PRINT "0";
321 FOR I=2,14,2: I1=-.75
322 CALL (3,2,I1,E): CALL (4,7): PRINT I;
323 NEXT I
324 FOR I=1,9
325 I1=-.5
326 I2=I-.75
328 CALL (3,2,I2,I1,E): CALL (4,7): PRINT I
329 NEXT I
370 CALL (4,7)
380 CALL (5)

```

```
385 FOR J=1,J1
386 S7=P(1,J)/FNA(H(1,J))*1E-05
390 PRINT TAB(20)," MESH:
395 PRINT
400 PRINT TAB(20)," SYSTEM:
410 PRINT TAB(20)," VELOCITY:"
411 PRINT TAB(29)," TIME TEMPERATURE PRESSURE DROP"
420 PRINT TAB(29)," SEC. DEG.C. 1/SEC."
425 FOR I=1,N(J):S(I,J)=S(I,J)*1000
430 PRINT TAB(29),S(I,J),H(I,J),R(I,J): NEXT I
431 S4=0: S3=0
432 FOR I=N(J),1,-1
433 N1=N(J)+1-I
434 S4=S4+R(I,J)
435 S3=S3+R(I,J)*2
436 S5=S4/N1: IF (S3-S4*2/N1)<0 THEN 442
437 S2=.252E-01*S5
438 S1=SQR((S3-S4*2/N1)/N1)
439 IF I=1 THEN 442
440 IF S1<S2 THEN 446
442 PRINT : PRINT TAB(20);"TIME TO STEADY STATE=";S(I,J);"SEC.": PRINT
443 PRINT TAB(15),"STEADY STATE PRESSURE DROP=";S5,"1/SEC."
444 IF S(I,J)>0 THEN PRINT : PRINT : GOTO 447
445 S8=S5/S7: PRINT : PRINT TAB(20),"PRESSURE DROP RATIO=",S8: GOTO 447
446 NEXT I
447 N2=21-N(J): IF N(J)>21 THEN N2=35-(N(J)-21)
448 FOR I=1,N2: PRINT : NEXT I
449 NEXT J
450 STOP
460 END
```

MESH: S300 NO. OF LAYERS: 10
 SYSTEM: TOLUENE SATURATION HISTORY: FREE
 VELOCITY: 0.1×10^{-2} m/s

TIME SEC.	TEMPERATURE DEG. C.	PRESSURE DROP 1/SEC.
0	18.5	.244813E-01
100	20.5	.518535E-01
250	21.5	.914372E-01
500	22	.119255
1000	23.5	.202994
1500	24	.340378
2000	25	.344512
2500	25	.358645
3000	26	.432716
5000	26	.55967
5500	21.5	.54185
7500	24.5	.561954
8500	25	.565352

TIME TO STEADY STATE= 3000 SEC.

STEADY STATE PRESSURE DROP= .532309 1/SEC.

PRESSURE DROP RATIO= 37.0011

MESH: S300 NO. OF LAYERS: 60
 SYSTEM: TOLUENE SATURATION HISTORY: FREE
 VELOCITY: 1.0×10^{-2} m/s

TIME SEC.	TEMPERATURE DEG. C.	PRESSURE DROP 1/SEC.
0	19.5	1.07399
250	20	4.8062
500	20	7.26334
1100	20.5	7.74458
1500	20.5	7.38996
2500	20.5	8.07242
3000	20.5	8.33504
3500	20.5	8.72812
4000	20.5	8.92383
4500	20.5	8.91044
5000	20.5	9.1212
5500	20.5	9.85384
6000	20.5	9.52767
6500	21	10.1651
7000	21	10.2307
7500	21	10.4159
8000	21	10.6666

TIME TO STEADY STATE= 6000 SEC.

STEADY STATE PRESSURE DROP= 10.2012 1/SEC.

PRESSURE DROP RATIO= 15.9644

MESH: S300 NO. OF LAYERS: 10
SYSTEM: TOLUENE SATURATION HISTORY: FREE
VELOCITY: 1.5×10^{-2} m/s

TIME SEC.	TEMPERATURE DEG. C.	PRESSURE DROP 1/ SEC.
0	20	.272645
100	21	1.81928
250	21.5	2.5196
500	21.5	2.75835
1000	22	2.77523
1500	22	2.53502
2000	22	2.53502
2500	22.5	2.65849
3000	22.5	2.65849
3500	22.5	2.65849

TIME TO STEADY STATE= 1000 SEC.

STEADY STATE PRESSURE DROP= 2.63679 1/ SEC.

PRESSURE DROP RATIO= 16.1562

MESH: S300 NO. OF LAYERS: 20
SYSTEM: TOLUENE SATURATION HISTORY: FREE
VELOCITY: 1.5×10^{-2} m/s

TIME SEC.	TEMPERATURE DEG. C.	PRESSURE DROP 1/ SEC.
0	20	.59849
100	20	2.70817
250	20	4.5585
500	20.5	6.00163
1000	20.5	5.72563
2000	21.5	5.91632
2500	22.5	6.15001
3000	23	6.33581
3500	24	6.41307
4000	25	6.36728

TIME TO STEADY STATE= 2000 SEC.

STEADY STATE PRESSURE DROP= 6.2365 1/ SEC.

PRESSURE DROP RATIO= 17.4079

MESH: S300 NO. OF LAYERS: 90
 SYSTEM: TOLUENE SATURATION HISTORY: FREE
 VELOCITY: 1.5×10^{-2} m/s

TIME SEC.	TEMPERATURE DEG. C.	PRESSURE DROP 1/ SEC.
0	19	2.34009
100	19	5.35182
250	19	7.77073
750	19	7.95137
1000	19	8.05319
1500	19	7.95137
2000	19.5	8.00041
2500	19.5	7.55925
3000	19.5	8.00041
3500	19.5	8.00041
4000	20	7.60581

TIME TO STEADY STATE= 3500 SEC.

STEADY STATE PRESSURE DROP= 7.80311 1/ SEC.

PRESSURE DROP RATIO= 5.63911

MESH: S300 NO. OF LAYERS: 120
 SYSTEM: TOLUENE SATURATION HISTORY: FREE
 VELOCITY: 1.5×10^{-2} m/s

TIME SEC.	TEMPERATURE DEG. C.	PRESSURE DROP 1/ SEC.
0	19.5	3.19719
100	21	5.93245
250	21.5	7.85344
500	21.5	8.38343
1000	22	8.62213
1500	22	8.71583
2600	22	8.95605
3000	22.5	9.17187
3500	22.5	9.10503

TIME TO STEADY STATE= 500 SEC.

STEADY STATE PRESSURE DROP= 8.82572 1/ SEC.

PRESSURE DROP RATIO= 4.63962

MESH: S300

NO. OF LAYERS: 10

SYSTEM: TOLUENE

SATURATION HISTORY: FREE

VELOCITY: 7.0×10^{-2} m/s

TIME SEC.	TEMPERATURE DEG. C.	PRESSURE DROP 1/SEC.
0	23	1.49859
100	24	6.15299
350	25	6.22771
500	25	6.22771
1000	27	6.37882
1500	28	6.45516
1750	28.5	6.49351

TIME TO STEADY STATE= 0 SEC.

STEADY STATE PRESSURE DROP= 5.6335 1/SEC.

PRESSURE DROP RATIO= 6.06502

MESH: S300

NO. OF LAYERS: 60

SYSTEM: TOLUENE

SATURATION HISTORY: FREE

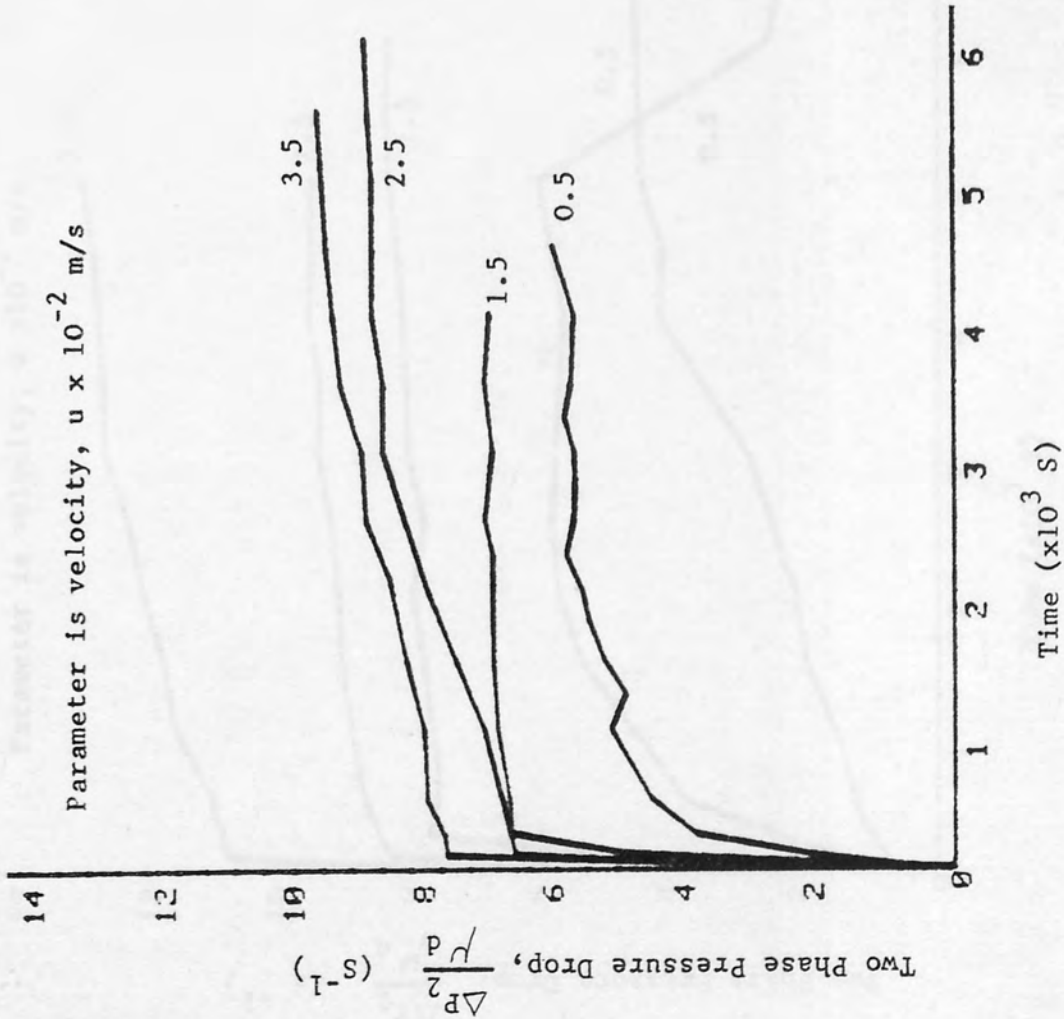
VELOCITY: 7.0×10^{-2} m/s

TIME SEC.	TEMPERATURE DEG. C.	PRESSURE DROP 1/SEC.
0	21	7.8443
100	24.5	18.6007
250	24.5	18.6007
500	25.5	18.8261
1000	27	19.3373
1500	28.5	19.6279
2000	30	20.0959

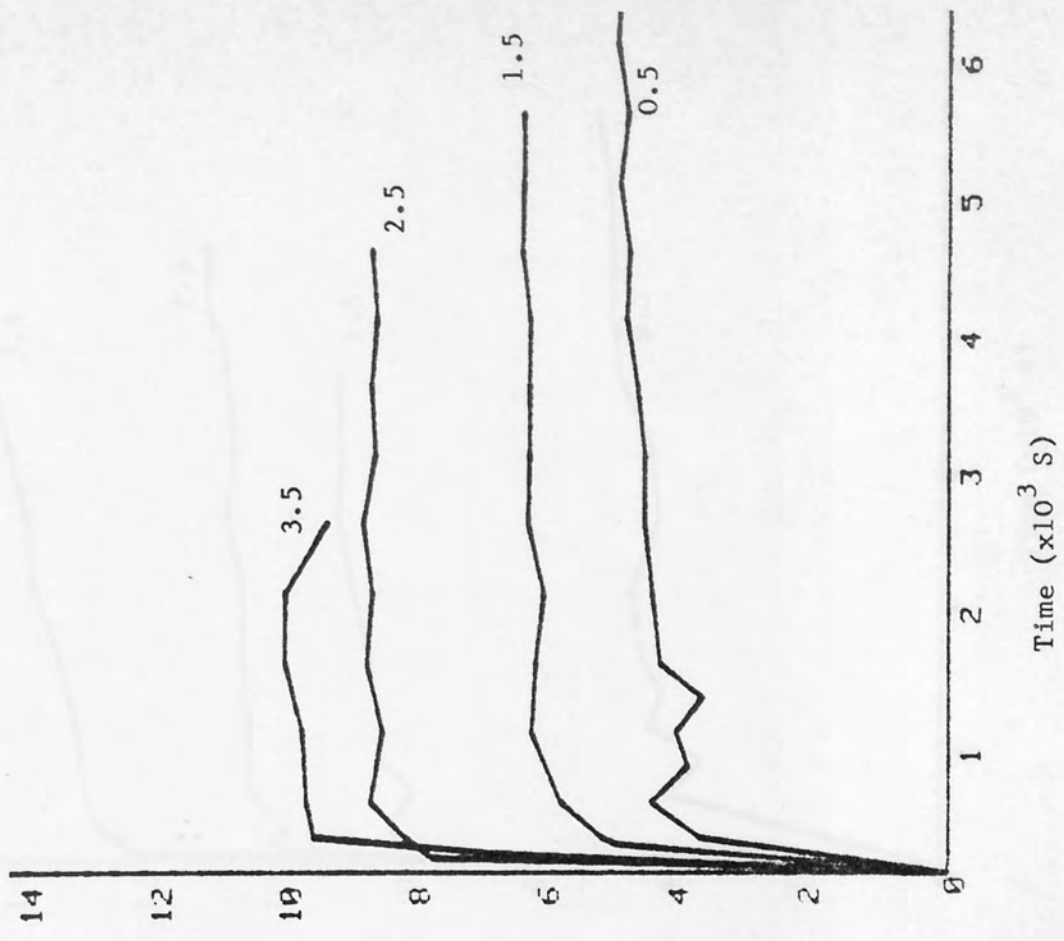
TIME TO STEADY STATE= 250 SEC.

STEADY STATE PRESSURE DROP= 19.2976 1/SEC.

PRESSURE DROP RATIO= 4.06104

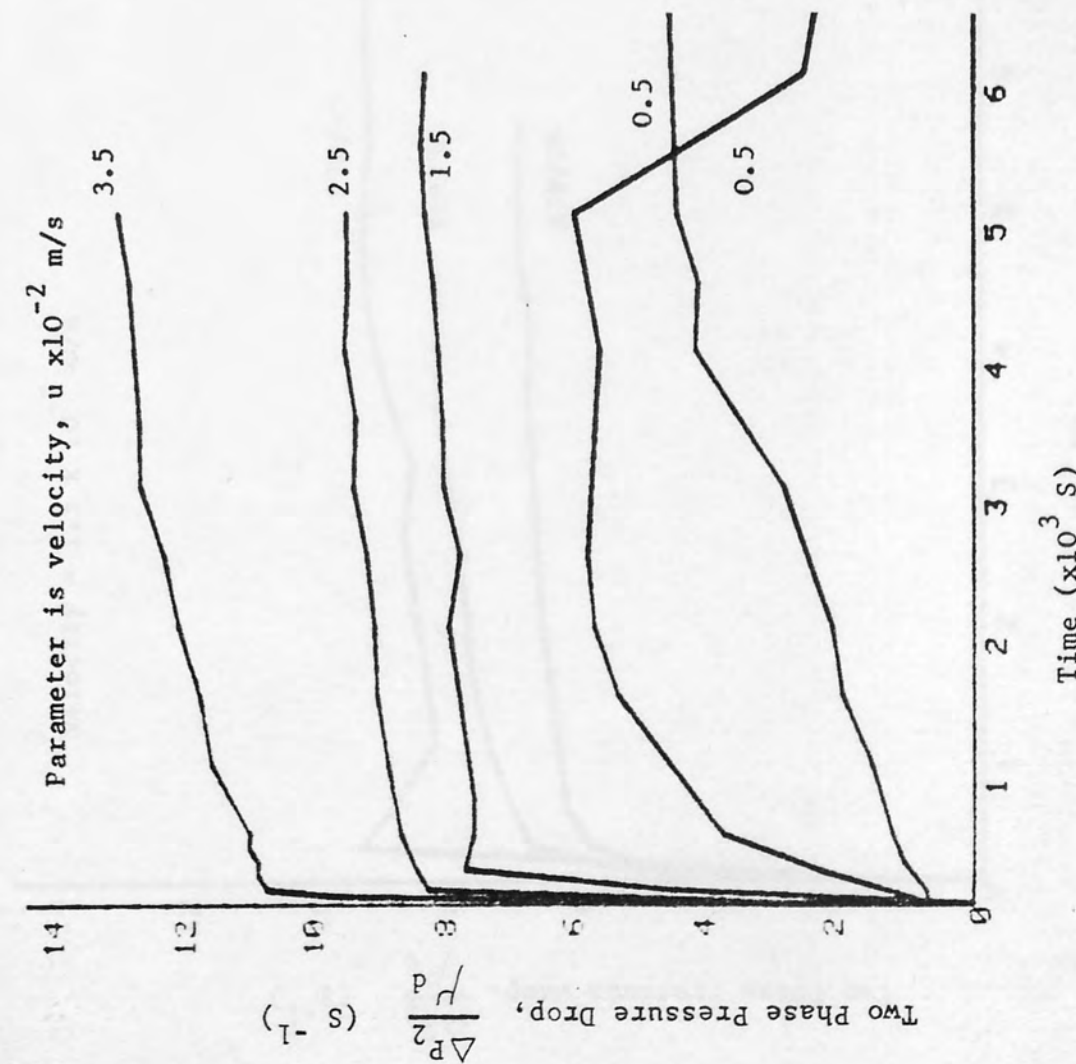


(a) Presoaked in Continuous Phase.

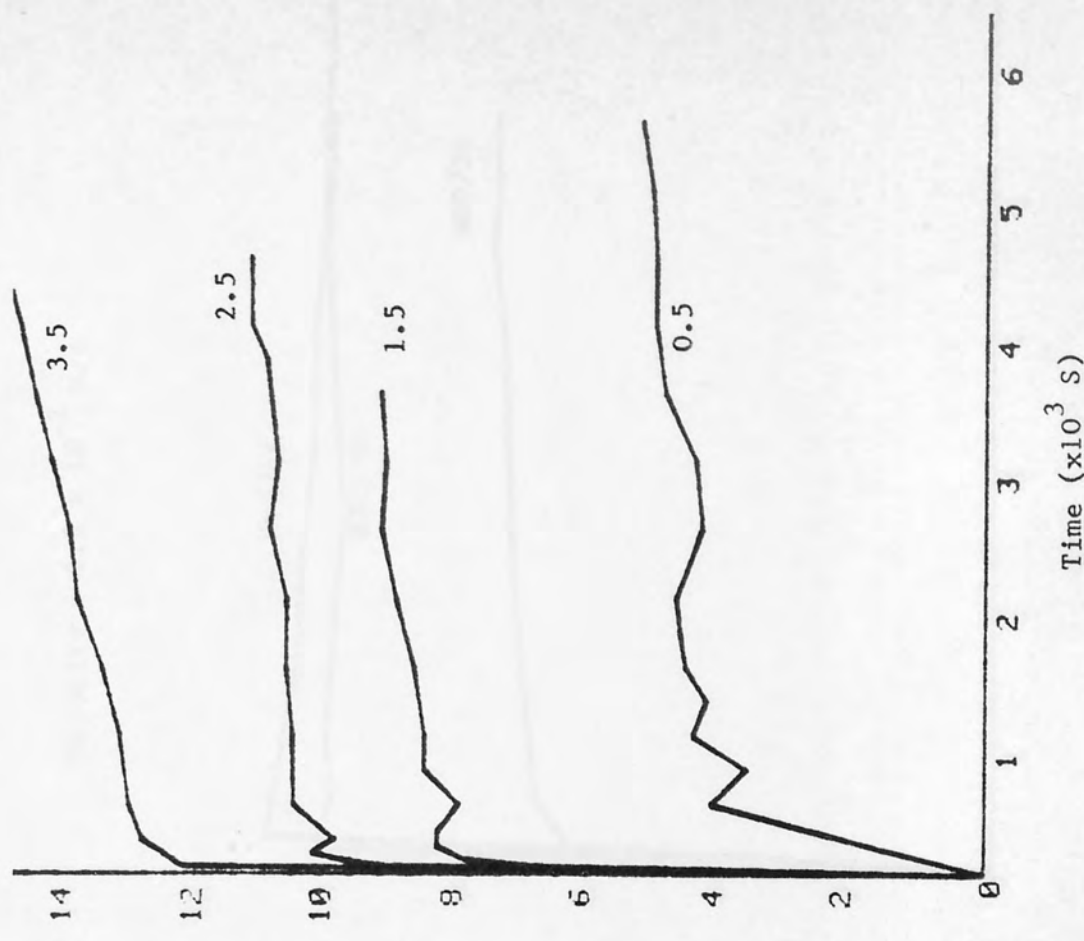


(b) Presoaked in Dispersed Phase.

Fig K.1. Transient Two Phase Pressure Drop for 30 layers of S300 Mesh.

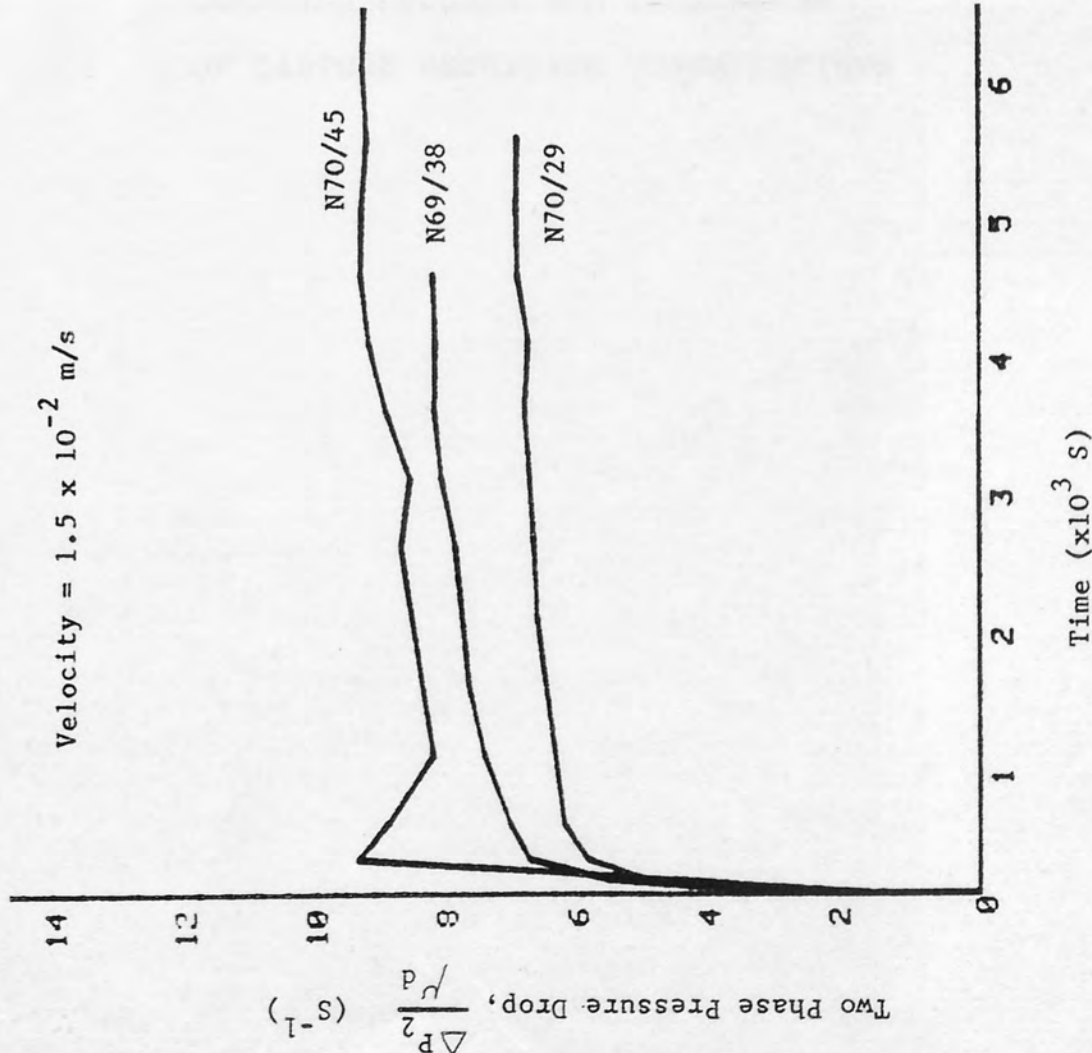


(a) Presoaked in Continuous Phase.

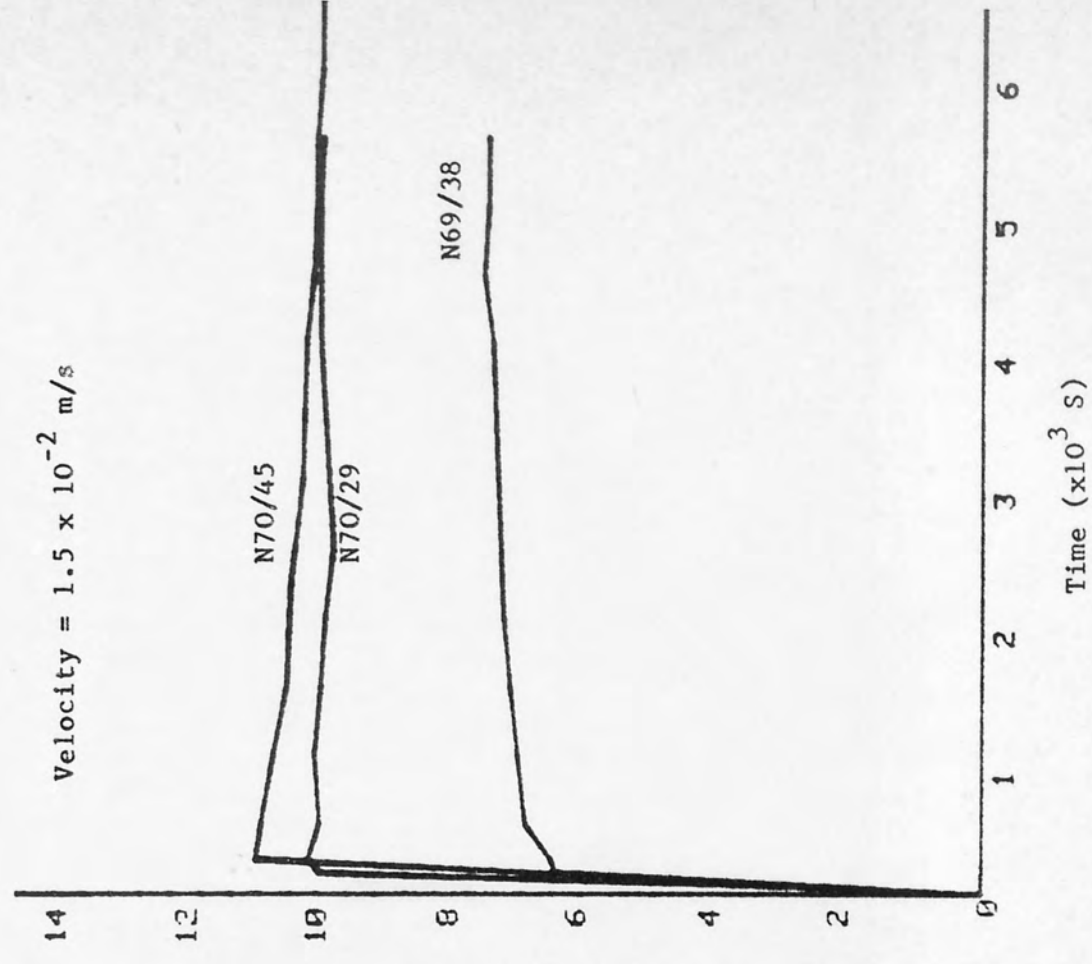


(b) Presoaked in Dispersed Phase.

Fig K.2. Transient Two Phase Pressure Drop for 60 layers of S300 Mesh.



(a) Presoaked in Continuous Phase.



(b) Presoaked in Dispersed Phase.

Fig. K.3 Transient Two Phase Pressure Drop for 60 layers of Nylon Mesh.


```
10 REM          EVALUATION OF CAPTURE MECHANISM CONTRIBUTIONS
11 REM          FOR S300 MESH ; TOLUENE-WATER SYSTEM
20 DIM F(7),N(7),P(7)
30 READ D1,D3,E
40 READ R3,M3,R4
50 READ K,T,G,G1
60 FOR L=1,7
70 READ F(L)
80 NEXT L
81 FOR M=1,7: READ P(M): NEXT M
90 FOR M=1,7
100 READ V
101 PRINT "VELOCITY=",V,"M/S": PRINT : V=V/E
102 PRINT "DROP SIZE          TOTAL          SIGNIFICANT          %REL"
103 PRINT "(MICRON)          EFFICIENCY          MECHANISM          CONTR"
110 FOR L=1,6
120 D2=F(L)*.1E-05
125 REM          CALCULATION OF DIMENSIONLESS GROUPS
130 R=D2/D3
140 R5=D3*V*R3/M3
150 H=2-LOG(R5)
160 S=(D2+2*R3*V)/(9*M3*D3)
170 G=(D2+2*(R4-R3)*G1)/(18*M3*V)
180 P5=(3*3.14159*D3*D2*V*M3)/(K*T)
190 N1=8*H*G/(9*3.14159*R+2*M3*V*D2+2)
195 REM          CALCULATION OF CAPTURE EFFICIENCIES
200 N(1)=(2*(1+R)*LOG(1+R)+1/(1+R)-(1+R))/(2*H)
210 N(2)=2.16/((2*H)+.33333*P5+.66667)
220 N(3)=-G
230 N(4)=R+2/H*(3*3.14159*N1/4)+.33333
240 N6=N(1)+N(2)+N(3)+N(4)
250 PRINT F(L),N6
260 FOR I=1,4
270 C=INT(N(1)*100/N6)
280 IF C<1 THEN 300
290 IF I=1 THEN PRINT TAB(30);"INTERCEPTION",TAB(46);C
291 IF I=2 THEN PRINT TAB(30);"DIFFUSION",TAB(46);C
292 IF I=3 THEN PRINT TAB(29);"SEDIMENTATION",TAB(46);C
293 IF I=4 THEN PRINT TAB(29);"LONDON FORCES",TAB(46);C
300 NEXT I
305 PRINT
310 NEXT L
315 FOR I=1,P(M): PRINT : NEXT I
320 NEXT M
330 STOP : END : FINISH
500 DATA .53E-04,.305E-04,.6995
510 DATA 1000,.1E-02,866.9
520 DATA .138048E-22,293,.55E-20,9.81
530 DATA .1,1,5,10,25,50,100
535 DATA 1,1,1,1,7,6,20
540 DATA .1E-03,.1E-02,.5E-02,.1E-01,.25E-01,.5E-01,.1
```

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VELOCITY=		M/S		
DROP SIZE (MICRON)	TOTAL EFFICIENCY		SIGNIFICANT MECHANISM	%RELATIVE CONTRIBUTION
• 1	• 1E-02			
	• 222527E-02		DIFFUSION	95
			LONDON FORCES	4
1	• 119493E-02			
			INTERCEPTION	17
			DIFFUSION	38
			SEDIMENTATION	4
			LONDON FORCES	40
5	• 757116E-02			
			INTERCEPTION	62
			DIFFUSION	2
			SEDIMENTATION	16
			LONDON FORCES	18
10	• 247844E-01			
			INTERCEPTION	70
			SEDIMENTATION	20
			LONDON FORCES	9
25	• 124406			
			INTERCEPTION	71
			SEDIMENTATION	25
			LONDON FORCES	3
50	• 412309			
			INTERCEPTION	67
			SEDIMENTATION	30
			LONDON FORCES	1

VELOCITY=		M/S		
DROP SIZE (MICRON)	TOTAL EFFICIENCY		SIGNIFICANT MECHANISM	%RELATIVE CONTRIBUTION
• 1	• 1E-03			
	• 88728E-02		DIFFUSION	97
			LONDON FORCES	1
1	• 333375E-02			
			INTERCEPTION	4
			DIFFUSION	56
			SEDIMENTATION	15
			LONDON FORCES	24
5	• 189697E-01			
			INTERCEPTION	17
			DIFFUSION	3
			SEDIMENTATION	66
			LONDON FORCES	12
10	• 669071E-01			
			INTERCEPTION	17
			SEDIMENTATION	75
			LONDON FORCES	5
25	• 3854			
			INTERCEPTION	15
			SEDIMENTATION	82
			LONDON FORCES	1
50	1.47221			
			INTERCEPTION	13

VELOCITY=	.5E-02	M/S		
DROP SIZE (MICRON)	TOTAL EFFICIENCY	SIGNIFICANT MECHANISM	%RELATIVE CONTRIBUTION	
.1	.902696E-03	DIFFUSION LONDON FORCES	90 8	
1	.848545E-03	INTERCEPTION DIFFUSION SEDIMENTATION LONDON FORCES	35 20 1 42	
5	.827297E-02	INTERCEPTION SEDIMENTATION LONDON FORCES	83 3 12	
10	.280434E-01	INTERCEPTION SEDIMENTATION LONDON FORCES	90 3 6	
25	.13841	INTERCEPTION SEDIMENTATION LONDON FORCES	93 4 2	
50	.436568	INTERCEPTION SEDIMENTATION LONDON FORCES	93 5 1	

VELOCITY=	.1E-01	M/S		
DROP SIZE (MICRON)	TOTAL EFFICIENCY	SIGNIFICANT MECHANISM	%RELATIVE CONTRIBUTION	
.1	.632274E-03	DIFFUSION LONDON FORCES	88 11	
1	.830012E-03	INTERCEPTION DIFFUSION LONDON FORCES	44 14 40	
5	.972962E-02	INTERCEPTION SEDIMENTATION LONDON FORCES	88 1 10	
10	.335846E-01	INTERCEPTION SEDIMENTATION LONDON FORCES	93 1 4	
25	.166561	INTERCEPTION SEDIMENTATION LONDON FORCES	96 1 1	
50	.522979	INTERCEPTION SEDIMENTATION	96 2	

VELOCITY=	.25E-01	M/S		
DROP SIZE (MICRON)	TOTAL EFFICIENCY	SIGNIFICANT MECHANISM	%RELATIVE CONTRIBUTION	
.1	.418569E-03		INTERCEPTION	1
			DIFFUSION	82
			LONDON FORCES	16
1	.94468E-03		INTERCEPTION	58
			DIFFUSION	7
			LONDON FORCES	33
5	.137069E-01		INTERCEPTION	92
			LONDON FORCES	6
10	.482869E-01		INTERCEPTION	96
			LONDON FORCES	3
25	.241387		INTERCEPTION	98
			LONDON FORCES	1
50	.757321		INTERCEPTION	98

VELOCITY=	.1	M/S		
DROP SIZE (MICRON)	TOTAL EFFICIENCY	SIGNIFICANT MECHANISM	%RELATIVE CONTRIBUTION	
.1	.332417E-03		INTERCEPTION	6
			DIFFUSION	63
			LONDON FORCES	30
1	.251449E-02		INTERCEPTION	79
			DIFFUSION	1
			LONDON FORCES	18
5	.474883E-01		INTERCEPTION	97
			LONDON FORCES	2
10	.171294		INTERCEPTION	98
			LONDON FORCES	1
25	.865654		INTERCEPTION	99
50	2.7213		INTERCEPTION	99

VELOCITY=	.5E-01	M/S		
DROP SIZE (MICRON)	TOTAL EFFICIENCY	SIGNIFICANT MECHANISM	%RELATIVE CONTRIBUTION	
.1	.334307E-03	INTERCEPTION	2	
		DIFFUSION	75	
		LONDON FORCES	22	
1	.125866E-02	INTERCEPTION	68	
		DIFFUSION	4	
		LONDON FORCES	27	
5	.209523E-01	INTERCEPTION	95	
		LONDON FORCES	4	
10	.747426E-01	INTERCEPTION	97	
		LONDON FORCES	2	
25	.375747	INTERCEPTION	99	
50	1.17981	INTERCEPTION	99	

VELOCITY= .5E-02 (M/S)

BED DEPTH	FRACTION NOT CAPTURED
1	.513025
2	.397916
3	.331231
4	.286467
5	.253794
10	.165959
20	.995533E-01
30	.698821E-01
40	.525194E-01
50	.410098E-01
60	.328109E-01
70	.266909E-01
80	.219732E-01
90	.182496E-01
100	.152585E-01
110	.12823E-01
120	.108193E-01

APPENDIX M.

FORMULATION OF EQUATION TO DESCRIBE CRITICAL TRAJECTORY
OF A DROP APPROACHING A CYLINDRICAL COLLECTOR.

In Section 10.2 it was shown that London-Van der Waal's forces may aid capture of drops having a diameter less than about 10 μm . Spielman and Goren⁶⁸ derived an equation to describe the critical trajectory of a drop approaching a cylindrical fibre and their equation was used as a basis for a more detailed investigation into the role of London forces. Their result, which is based on a force balance for a drop in the vicinity of the collector, is,

$$-\sin \Theta_p \frac{dH}{d\Theta_p} = \frac{f_1(H)}{f_3(H)} \left[f_2(H) \cos \Theta_p + \frac{N_{Adc} f_{ret}}{(H + 2)^2 H^2} \right] \quad \text{M.1}$$

M.1. London Attractive Force.

The term of equation M.1 containing the Adhesion number is based on the original expression of Hamaker (see equation 5.6) for the London force acting between a sphere and a plane surface. Consideration of the interactions for a sphere/plane system is acceptable when the Interception number is very small. However, when the drop diameters are dimensionally close to the collector diameter, it follows that equations describing the London attraction between a sphere and cylinder will provide a better description that is also consistent with the flow model based on flow over an array of parallel cylinders. Rosenfeld⁷⁵ derived an expression for the London force between a sphere and cylinder, thus,

$$F_{Ad} = \frac{-2Qa_p^3}{3(2a_p+h)^2 h^2} \left[\frac{a_c}{a_c + a_p + h} \right]^{0.5} \left[1 - \frac{0.2h(2a_p+h)}{a_c(a_c+a_p+h)} \right] \quad \text{M.2}$$

where a_p and a_c are the drop and collector radius

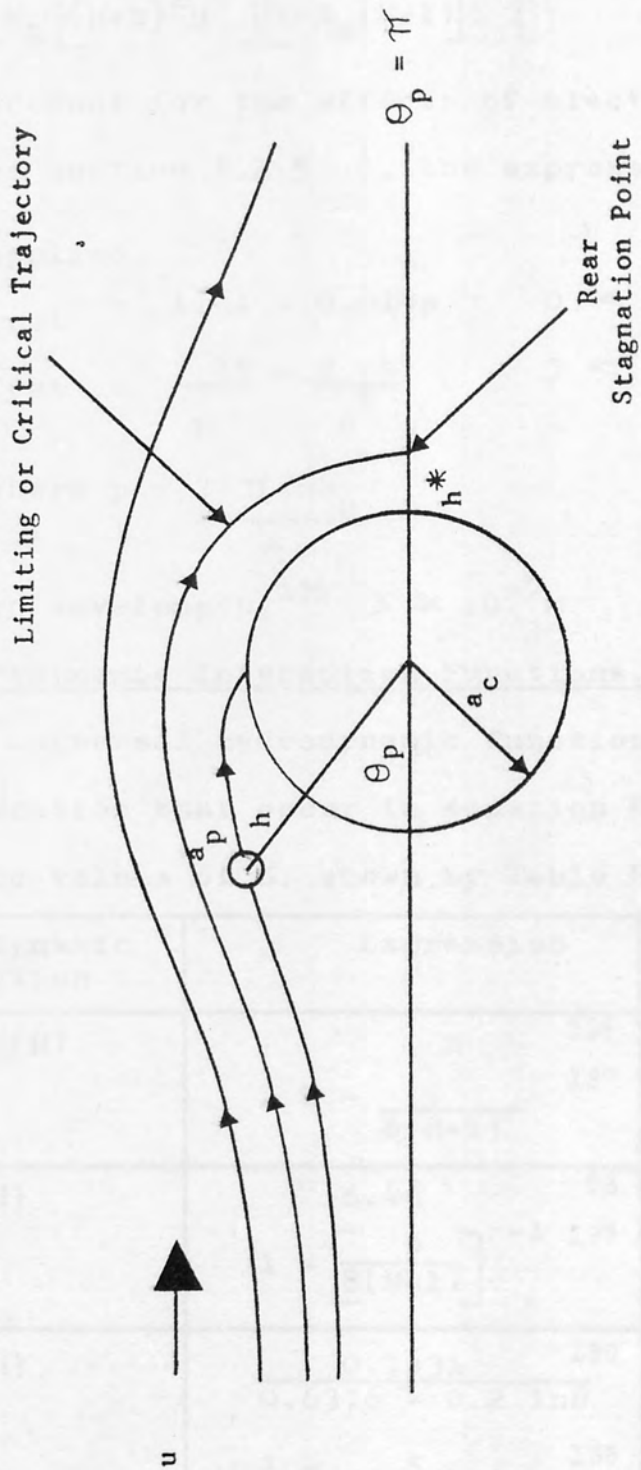


Fig M.1 Qualitatively Depicted Trajectories for Cylindrical Collector.

respectively and h is the separation between drop and collector as shown by Fig M.1 which also presents the co-ordinate system. Defining the dimensionless separation, $H = \frac{h}{a_p}$, equation M.2 may be rearranged to,

$$F_{Ad} = \frac{2Q}{3a_c N_R} \left[\frac{1}{(H+2)^2 H^2} \right] \left[\frac{1}{1+N_R(H+1)} \right]^{0.5} \left[1 - \frac{0.2 N_R^2 H(H+2)}{1+N_R(H+1)} \right] \quad M.3$$

To account for the effects of electromagnetic retardation (see Section 5.2.5), the expression given by Kruyt¹³⁵ was applied,

$$\left. \begin{aligned} f_{ret} &= 1.01 - 0.014p & 0 < p < 3 \\ f_{ret} &= \frac{2.45}{p} - \frac{2.04}{p^2} & 3 < p < \infty \end{aligned} \right\} \quad M.4$$

$$\text{where } p = \frac{2 \pi Ha_p}{\lambda}$$

The London wavelength,¹³⁵ $\lambda \simeq 10^{-7} \text{m}$.

M.2. Hydrodynamic Interaction Functions.

The universal hydrodynamic functions of the dimensionless separation that occur in equation M.1 are known for asymptotic values of H ; shown by Table M.1.

Hydrodynamic function	Expression	Constraint
$f_1(H)$	H ¹³⁶	$H \rightarrow 0$
	$1 - \frac{9}{8(H+1)}$ ¹³⁷	$H \rightarrow \infty$
$f_2(H)$	6.46 ⁶⁸	$H \rightarrow 0$
	$1 - \left[\frac{9}{8(H+1)} \right]^{-1}$ ¹³⁷	$H \rightarrow \infty$
$f_3(H)$	$\frac{0.7431}{0.6376 - 0.2 \ln H}$ ¹³⁸	$H < 0.003$
	$1 - \frac{5}{16(H+1)^3}$ ¹³⁸	$H > 0.003$

Table M.1. Asymptotic Behaviour of Universal Hydrodynamic Functions.

The variation of these functions is shown graphically in

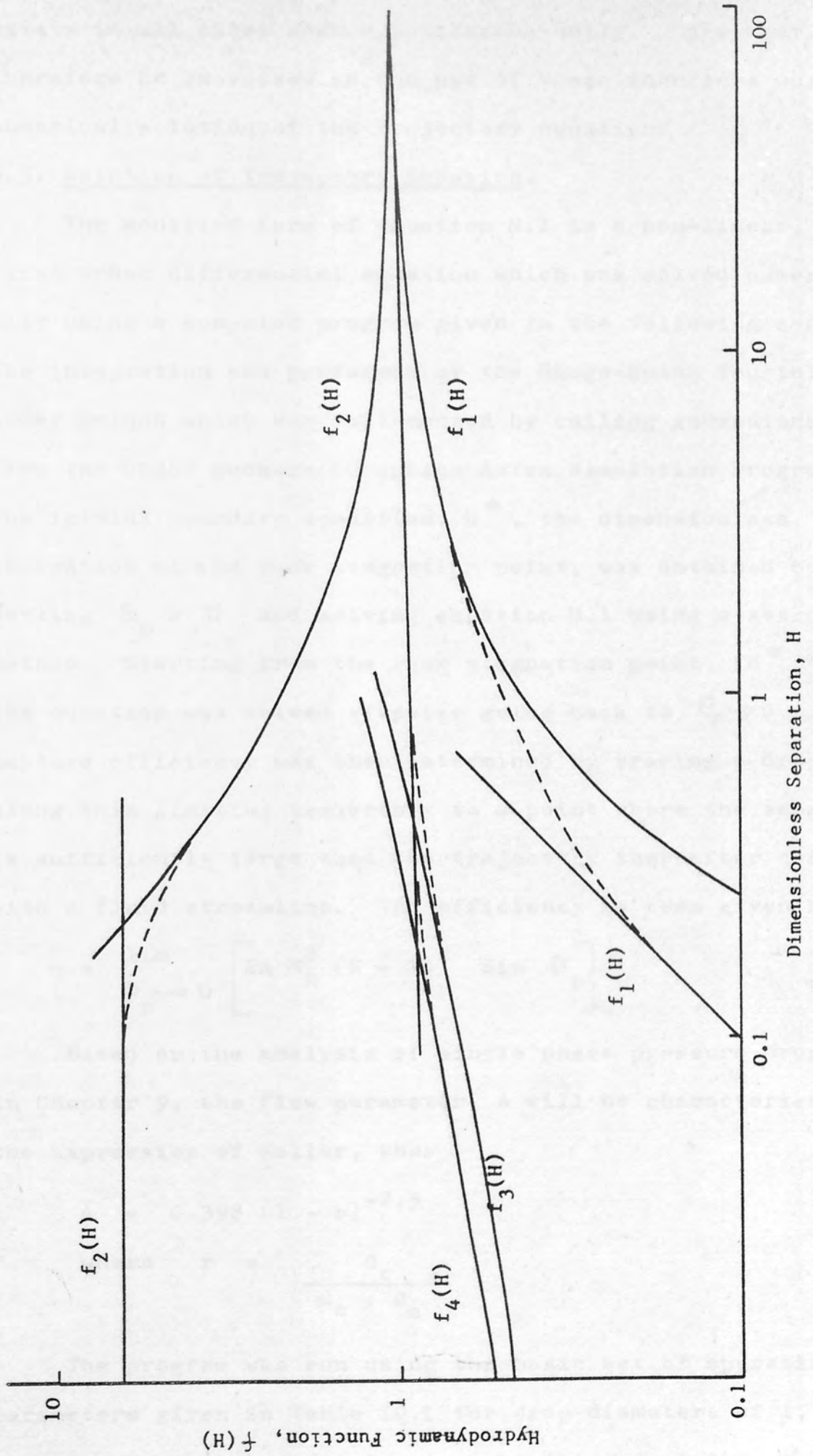


Fig. M.2 Behaviour of Hydrodynamic Functions with Separation illustrating discontinuities.

Fig M.2 from which it may be observed that a discontinuity exists in all cases when H approaches unity. Care must therefore be exercised in the use of these functions during numerical solution of the trajectory equation.

M.3. Solution of Trajectory Equation.

The modified form of equation M.1 is a non-linear, first order differential equation which was solved numerically using a computer program given in the following section. The integration was performed by the Runge-Kutta fourth order method which was implemented by calling subroutines from the GRASP package (Graphics Aston Simulation Program). The initial boundary condition, H^* , the dimensionless separation at the rear stagnation point, was obtained by setting $\Theta_p = \pi$ and solving equation M.1 using a search method. Starting from the rear stagnation point, (H^*, π) , the equation was solved stepwise going back to $\Theta_p \rightarrow 0$. The capture efficiency was then determined by tracing a drop along this limiting trajectory to a point where the separation is sufficiently large that the trajectory thereafter coincides with a fluid streamline. The efficiency is then given by⁶⁸,

$$\eta = \lim_{\Theta_p \rightarrow 0} \left[2A N_R^2 (H + 1)^2 \sin \Theta_p \right] \quad M.5$$

Based on the analysis of single phase pressure drop data in Chapter 9, the flow parameter, A will be characterised by the expression of Keller, thus

$$A = 0.398 (1 - r)^{-2.5} \quad M.6$$

$$\text{where } r = \frac{d_c}{d_c + d_a}$$

The program was run using the basic set of operating parameters given in Table 10.1 for drop diameters of 1, 5 and 10 μm and the output is given at the end of this appendix.

Using the graph plotting subroutines of the package, the associated critical trajectories are plotted below the numerical output. The discontinuity observed when H approaches unity is due to use of the asymptotic forms of the universal hydrodynamic functions.


```

9 FEM COMPUTATION OF CAPTURE EFFICIENCY FROM EQUATION
10 FEM DESCRIBING CRITICAL DROP TRAJECTORY
11 DIM S(300),A(1000),E(1000)
20 FEM INPUT GENERAL SYSTEM PROPERTIES
25 READ B1,G,L1,M3,F3,R2
30 FEM INPUT MESH PROPERTIES
35 READ A3,D4,N3,V5
50 FEM INPUT SUPERFICIAL VELOCITY
55 READ U2
60 FEM INPUT HAMAKER CONSTANT
65 READ Q
70 FEM INPUT A.S.P. PARAMETERS
80 READ P,E,M,I
270 FEM EVALUATION OF INTERSTITIAL VELOCITY
280 U1=U2/V5
281 PRINT "SUPERFICIAL VELOCITY=",U2,"M/S."
290 FEM CALCULATION OF FLOW PARAMETER AFTER KELLER
300 A4=1/((2*(2-LOG(2*R3*A3*U1/M3)))
310 FEM INPUT DROP RADIUS
320 READ A1
330 A8=2*A1: PRINT "DROP DIAMETER=",A8,"M.": PRINT
340 IF A8>D4 THEN PRINT "DROP DIAMETER> APERTURE": GOTO 905
350 D3=2*A3
360 R5=D3/(D3+D4)
370 H1=1.257*(1-R5)+2.5
380 A4=1/(2*H1)
390 FEM EVALUATION OF INTERCEPTION,ADHESION & GRAVITY GROUPS
400 R=A1/A3
410 N1=G*A3+2/(9*3.14159*M3*A4*U1*A1+4)
420 N2=(2*(R2-R3)*G*A3+2)/(9*M3*A4*U1)
430 PRINT "INTERCEPTION NUMBER=",R
440 PRINT "ADHESION NUMBER=",N1
450 PRINT "GRAVITY NUMBER=",N2: PRINT
460 FEM DETERMINATION OF SEPARATION AT REAR STAGNATION POINT
470 I=1:C=1:H2=1:J=1
480 H=.1E-05
490 GOSUB 2000
500 IF ABS((W2-F2)/W2)<C THEN 372
510 IF W2>F2 THEN 374
520 J=2:H=(1-H2)*H
530 GOTO 377
540 IF C<.1E-01 THEN 410
550 H2=.5*H2:C=.1*C:I=1: GOTO 369
560 IF J>1 THEN 376
570 H=H*10: GOTO 367
580 H=(1+H2)*H
590 I=I+1
600 IF I<20 THEN 367
610 H2=H2*.5: GOTO 367
620 PRINT "SEPARATION AT REAR STAGNATION POINT=",H: PRINT
630 REM INITIALISATION SECTION
640 PRINT " THETA DH/DI H"
650 PRINT
660 D7=1
670 CALL (1,P1)
680 T=22*(1-P1)/7
690 D5=0:K=1:S(K)=1:J=1
700 FEM DERIVATIVE SECTION
710 CALL (2,P,E,Z1,Z2)
720 A(J)=T:E(J)=H:J=J+1
730 IF T>.1 THEN 586
740 FEM COMPARISON OF DROP TRAJECTORY WITH FLUID STREAMLINE
750 K=K+1:S(K)=(H+1)+2*SIN(T):C=ABS((S(K)-S(K-1))/S(K-1))
760 IF C>.1E-01 THEN 586
770 IF D7>1 THEN 586
780 PRINT : PRINT "TRAJECTORY/STREAMLINE COINCIDENCE"

```

```

578 REM CALCULATION OF CAPTURE EFFICIENCY
579 E1=2*A4*F+2*S(K)
580 PRINT : PRINT "CYLINDER CAPTURE EFFICIENCY=",E1: PRINT
581 D7=2
586 REM TESTING THE VALLES OF PRINT & TERMINAL FLAGS
587 IF Z2=1 THEN 600
590 PRINT 1,D5,H
600 IF Z1=2 THEN PRINT "TERMINAL VALLE OF THETA ATTAINED": GOTO 85
610 REM INTEGRATION SECTION
615 IF T<.3 THEN M=.5E-02
620 CALL (3,F1,M,I)
630 CALL (4,H,-D5)
640 T=22*(1-P1)/7
650 GOSUB 2000
660 Y4=F2*COS(T)
670 D5=- (F1/F3*(Y4+W2))/SIN(T)
675 IF P1>.5 THEN D5=-D5
676 IF H>1000 THEN 850
680 GOTO 530
850 GOTO 880
870 REM PLOTTING OF DFOF CRITICAL TRAJECTORIES
880 CALL (5)
890 CALL (6,-45,80,-25,0)
900 CALL (7,2,-45,0,F)
910 CALL (7,1,35,0,F)
920 A3=INT(A3*.1E07):A1=A1*.1E07
930 CALL (7,2,-A3,0,F)
940 X=-A3:S1=1:S2=.25
950 Y=S1*SQR(A3+2-X+2)
960 CALL (7,1,X,Y,F)
970 IF X=A3 THEN S1=-S1:S2=-S2
980 X=X+S2
990 IF X<-A3 THEN 1005
1000 GOTO 950
1005 CALL (8,7)
1010 FOR I=1,J
1020 Z=A3+A1*(1+B(I))
1030 Y=Z*SIN(A(I)):X=Z*COS(A(I))
1035 X=-X
1040 CALL (7,1,X,Y,F)
1050 NEXT I
1060 CALL (8,7)
1070 CALL (9)
1130 STOP : END
1500 DATA 1,9.81,.1E-06,.1E-02,1000,866.9
1510 DATA .1525E-04,.53E-04,.3E05,.6995
1530 DATA .5E-02
1540 DATA .55E-20
1550 DATA .1,1,.1E-01,4
1560 DATA .25E-05
1999 REM COMPUTATION OF UNIVERSAL HYDRODYNAMIC FUNCTIONS
2000 IF H>.5 THEN F1=1-(9/(8*(H+1))):F2=1/F1: GOTO 2020
2010 F1=H:F2=6.46
2020 IF H>.1 GOTO 2030
2025 F6=.6376-.2*LOG(H):F3=.7431/F6:F4=.8436/F6
2026 GOTO 2040
2030 F3=1-(5/(16*(H+1)+3)):F4=F3
2040 P2=44/7*A3*R*H/L1
2050 IF P2>3 THEN F5=2.45/F2-2.04/F2+2: GOTO 2070
2060 F5=1.01-.14*P2
2070 X1=1/((H+2)+2*H+2)
2080 X2=SQR(1/(1+F*(H+1)))
2090 X3=1-(.2*F+2*H*(H+2))/(1+R*(H+1))
2100 W2=N1*X1*X2*X3*F5
2110 RETURN

```

SUPERFICIAL VELOCITY= .5E-02 M/S.
DROP DIAMETER= .1E-05 M.

INTERCEPTION NUMBER= .327869E-01
ADHESION NUMBER= .817123E-01
GRAVITY NUMBER= -.761783E-02

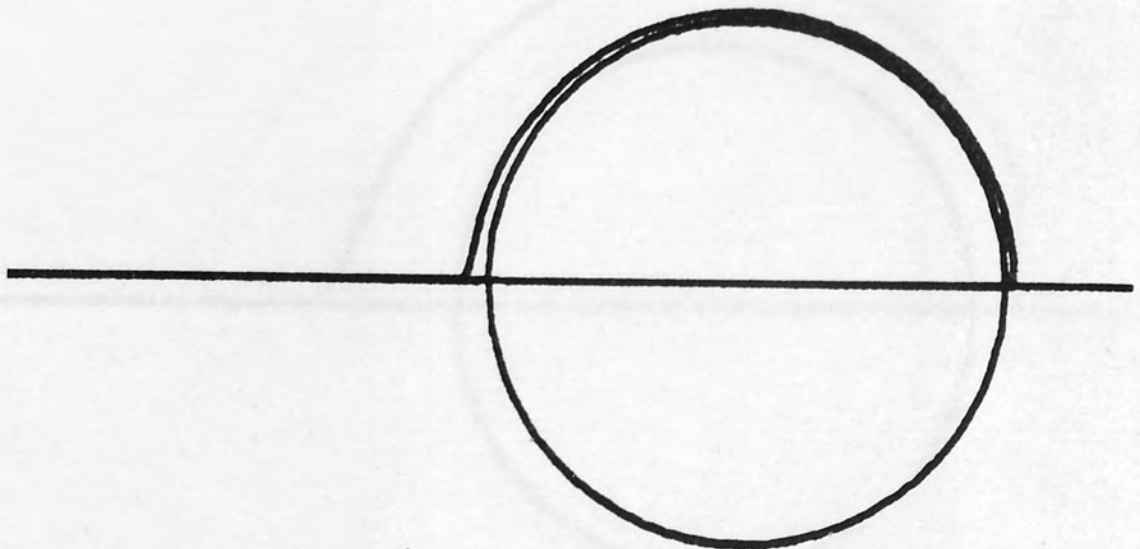
SEPARATION AT REAR STAGNATION POINT= .48526E-01

THETA	DH/D1	H
3.14286	0	.48526E-01
2.82857	.126118E-01	.497941E-01
2.51429	-.373687E-01	.521105E-01
2.2	-.674708E-01	.572677E-01
1.88572	-.118488	.663315E-01
1.57143	-.220764	.826662E-01
1.25714	-.420732	.113627
.942859	-1.11063	.183376
.628574	-4.04336	.40223
.314288	-3.28551	.706423

TRAJECTORY/STREAMLINE COINCIDENCE

CYLINDER CAPTURE EFFICIENCY= .111041E-02

.299726E-05 - .333638E 00 845.422



SUPERFICIAL VELOCITY= .5E-02 M/S.
LEOF DIAMETER= .5E-05 M.

INTERCEPTION NUMBER= .163934
ADHESION NUMBER= .13074E-03
GRAVITY NUMBER= -.761783E-02

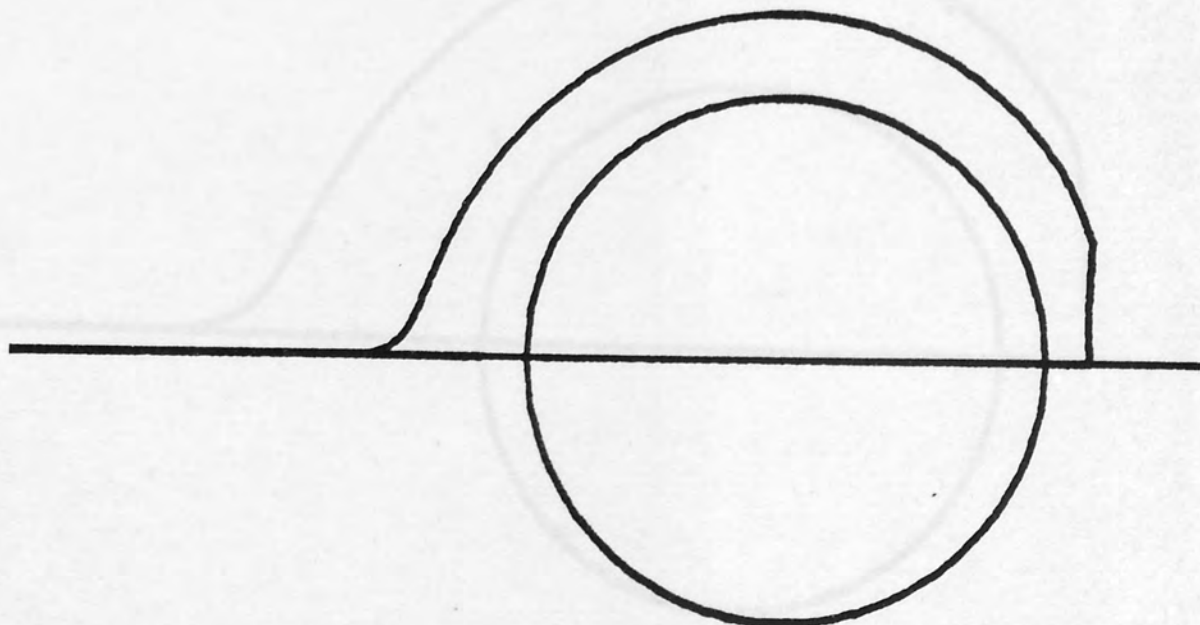
SEPARATION AT REAR STAGNATION POINT= .212868E-02

THETA	DH/L1	H
3.14286	0	.212868E-02
2.82857	8.98418	.398726
2.51429	1.47026	.715918
2.2	.767422	.823938
1.88572	.341885	.878217
1.57143	.664824E-03	.895047
1.25714	.339545	.911725
.942859	.757121	.965473
.628574	1.42579	1.07115
.314288	3.15937	1.28234

TRAJECTORY/STREAMLINE COINCIDENCE

CYLINDER CAPTURE EFFICIENCY= .263106E-01

.299726E-05 .333638E 06 842.191



SUPERFICIAL VELOCITY= .5E-02 M/S.
LEOP DIAMETER= .1E-04 M.

INTERCEPTION NUMBER= .327869
ADHESION NUMBER= .817121E-05
GRAVITY NUMBER= -.761783E-02

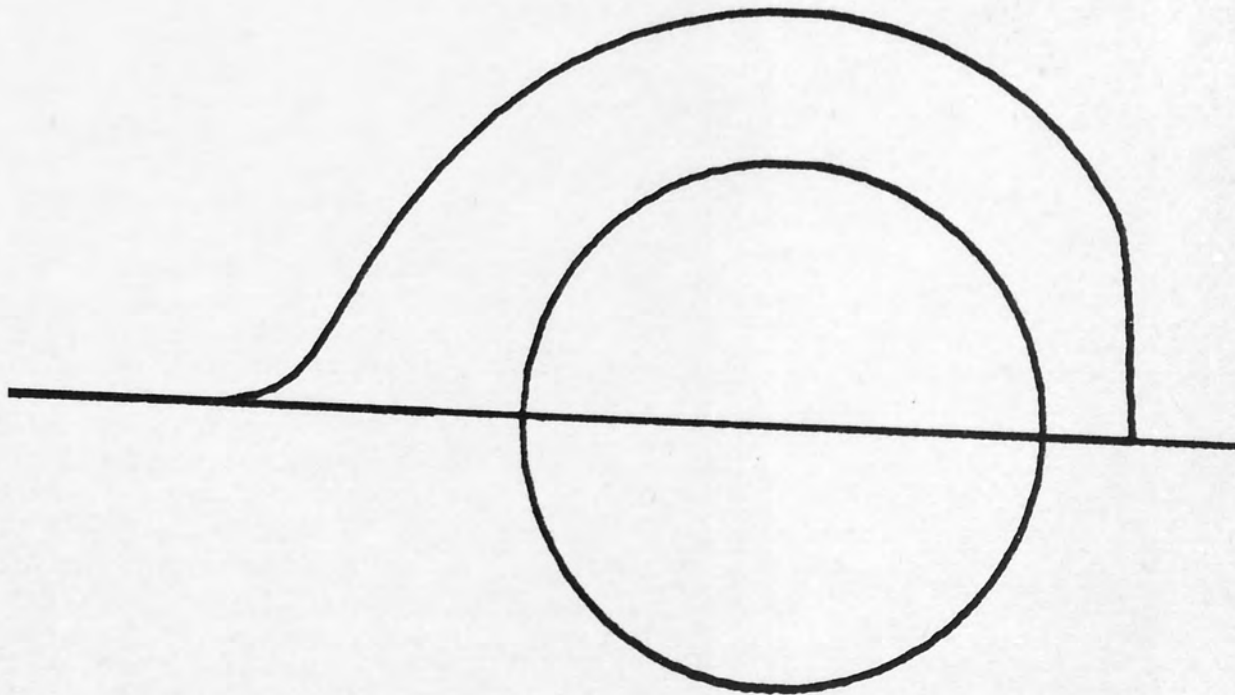
SEPARATION AT REAR STAGNATION POINT= .520666E-03

THEIA	LH/L1	H
3.14286	0	.520666E-03
2.82857	3.36507	.132306
2.51429	1.50926	.536792
2.2	.782611	.64729
1.88572	.347781	.702572
1.57143	.675944E-03	.719686
1.25714	.344939	.736635
.942859	.767706	.791184
.628574	1.44151	.898176
.314288	3.18199	1.11125

TRAJECTORY/STREAMLINE COINCIDENCE

CYLINDER CAPTURE EFFICIENCY= .933547E-01

.299726E-05 .333638E 06 842.902




```
10 REM      CALCULATION OF DROP CAPTURE RATE BY DIRECT INTERCEPTION
20 REM      AND LONDON FORCES FOR A BED WITH ZERO INITIAL SATURATION
40 DIM D(50),N(50),F(120)
50 READ D1,D3,A2,C1,E5,R4,R3
55 READ G1,M3,Q
56 FOR K=1,5
60 READ U1
70 DEF FNW(D)=113.525-3.44166*D+.221507E-01*D^2
80 A1=1-(D1/(D1+D3))^2
90 R2=.5*D3/(D1+D3)
100 REM     HYDRODYNAMIC FUNCTION AFTER KELLER
110 H=1.257*(1-2*R2)^2.5
114 PRINT TAB(15);"VELOCITY=",U1,"(M/S)": PRINT
120 I=1
130 D2=2
140 F1=U1*A2*C1
150 W1=FNW(D2)
160 D2=D2+2
170 W2=FNW(D2)
180 F2=.1E-01*(W1-W2)*F1
190 D(I)=(D2-1)*.1E-05
200 N(I)=6*F2/(3.14159*D(I)+3)
210 I=I+1:W1=W2
220 IF D2=54 THEN 235
230 GOTO 160
235 C=10
236 PRINT : PRINT TAB(15);"BED DEPTH FRACTION NOT CAPTURED"
240 FOR L=1,120
250 F3=0
260 FOR J=1,I-1
270 R=D(J)/D3
280 E3=2*(1+R)*LOG(1+R)-(1+R)+1/(1+R)
285 V=U1/E5
290 G=(D(J)+2*(R4-R3)*G1)/(18*M3*V)
300 N1=8*H*Q/(9*3.14159*R+2*M3*V*D(J)+2)
310 E1=E3/(2*H)-G
320 E1=E1+R+2/H*(3*3.14159*N1/4)+.33333
330 N(J)=N(J)*(1-E1*A1)
335 IF N(J)<1 THEN N(J)=0
340 F3=F3+N(J)*3.14159*D(J)+3/6
350 NEXT J
360 F4=F3/F1
365 IF L<6 THEN PRINT TAB(15);L,F4
375 IF L=C THEN PRINT TAB(15);L,F4:C=C+10
370 NEXT L
380 PRINT : PRINT
385 NEXT K
390 STOP
500 DATA .53E-04,.305E-04,.962112E-03,.5E-03,.6995,866.9,1000
510 DATA 9.81,.1E-02,.55E-20
520 DATA .5E-02
530 DATA .15E-01
540 DATA .25E-01
550 DATA .35E-01
560 DATA .7E-01
600 END
```

APPENDIX O.

EVALUATION OF SPECIFIC SURFACE FOR COALESCER

CONTAINING DROPS OF DISPERSED PHASE.

Specific surface,

$$a = \frac{(\text{surface area of drops} + \text{surface area of fibres})}{(\text{volume of drops} + \text{volume of fibres})}.$$

let number of drops per unit volume of bed = N_p

$$\text{then surface area} = N_p \pi d_p^2$$

$$\text{volume} = N_p \pi \frac{d_p^3}{6}$$

let number of fibres per unit volume of bed = N_c

$$\text{then surface area} = N_c \pi d_c l$$

$$\text{volume} = N_c \pi \frac{d_c^2}{4} l$$

if void fraction of fibres = e_1

$$\text{then } N_c = \frac{4(1-e_1)}{\pi d_c^2 l}$$

if void fraction of drops = e_p

$$N_p = \frac{6(1-e_p)}{\pi d_p^3}$$

also two phase effective voidage, $e_2 = (e_p + e_1) - 1$

then $e_p = (e_2 - e_1) + 1$

but saturation, $S = 1 - \frac{e_2}{e_1}$

$$\text{or } e_2 = e_1 (1-S)$$

$$\therefore a = \frac{\left\{ \frac{6}{d_p} (e_1 - e_2) + \frac{4}{d_c} (1 - e_2) \right\}}{(1 - e_2)}$$

$$a = \frac{\left\{ \frac{6}{d_p} e_1 S + \frac{4}{d_c} [1 - e_1(1-S)] \right\}}{[1 - e_1(1-S)]}$$

Since the first term of the integrand in the same equation is a constant, this must be considered separately. Substituting $x = 1-y$, the first term becomes,

$$\frac{6}{d_p} e_1 S \int_0^1 \frac{dy}{(1-y)(1-y^2)}$$

$$= \frac{6}{d_p} e_1 S \int_0^1 \frac{dy}{(1-y)^2(1+y)}$$

$$= \frac{6}{d_p} e_1 S \left[\frac{1}{1-y} + \frac{1}{1+y} - \frac{1}{1-y^2} \right]_0^1$$

$$= \frac{6}{d_p} e_1 S \left[\frac{1}{1-1} + \frac{1}{1+1} - \frac{1}{1-1} \right]$$

It is now possible to integrate the individual items on the right hand side,

$$\int_0^1 \frac{dy}{(1-y)(1-y^2)} = \int_0^1 \frac{dy}{(1-y)^2(1+y)}$$

$$= \left[\frac{1}{1-y} + \frac{1}{1+y} - \frac{1}{1-y^2} \right]_0^1$$

$$= \left[\frac{1}{1-1} + \frac{1}{1+1} - \frac{1}{1-1} \right]$$

APPENDIX P.

INTEGRATION OF PRESSURE DROP EQUATION.

In Section 10.5 solution of the equation describing the two phase pressure drop ratio was reduced to the problem of integration of the following polynomial expression,

$$\int_{L_I}^L f(S)dl = \frac{1}{k} \int_{ge}^{g+h} \frac{(av^4 + bv^3 + cv^2 + dv + f)}{(v-h)(1-v)^3} dv \quad P.i$$

Since the first term of the numerator is the same degree in v as the denominator, this must be considered separately. Substituting $z = (1-v)$, the first term becomes,

$$\begin{aligned} \frac{av^4}{(v-h)(1-v)^3} &= \frac{a(1-z)^4}{z^3(1-h-z)} \\ &= \frac{a}{(1-h-z)} \left[z - 4 + \frac{6}{z} - \frac{4}{z^2} + \frac{1}{z^3} \right] \end{aligned}$$

then re-substituting $v = (1 - z)$

$$\begin{aligned} \frac{av^4}{(v-h)(1-v)^3} &= a \left[\frac{-3}{(v-h)} - \frac{v}{(v-h)} + \frac{6}{(v-h)(1-v)} \right. \\ &\quad \left. - \frac{4}{(v-h)(1-v)^2} + \frac{1}{(v-h)(1-v)^3} \right] \end{aligned}$$

It is now possible to integrate the individual items on the right hand side,

$$\begin{aligned} \int_{ge}^{g+h} \frac{av^4}{(v-h)(1-v)^3} dv &= a \left\{ - \left[3 \log_e(v-h) \right] - \right. \\ &\quad \left[-h + v + h \log_e(v-h) \right] + \left[\frac{6}{(1-h)} \log_e \frac{(v-h)}{(1-v)} \right] - \\ &\quad \left. \frac{4}{(1-h)} \left[\frac{1}{(1-v)} - \frac{1}{(1-h)} \log_e \frac{(v-h)}{(1-v)} \right] + \right. \end{aligned}$$

$$\frac{1}{2(1-h)} \left[\frac{-1}{(1-v)^2} - \frac{2}{(1-h)} \left\{ \frac{1}{(1-v)} - \frac{1}{(1-h)} \log_e \frac{(v-h)}{(1-v)} \right\} \right]$$

Further algebraic manipulation of the constituent terms reduces the above equation to a relatively simplified form,

$$\int_{ge}^{g+h} -k(L-L_I) \frac{av^4}{(v-h)(1-v)^3} dv = a \left[\frac{(6h^2-8h+3)}{(1-h)^3} \log_e \frac{(v-h)}{(1-v)} + \frac{h(9-8v)-(11-10v)}{2(1-h)^2(1-v)^2} - (h+3) \log_e (v-h) - (v-h) \right] \quad P.2.$$

Having evaluated the first term of the numerator, the remainder may now be resolved into partial fractions,

$$i.e. \frac{bv^3+cv^2+dv+f}{(v-h)(1-v)^3} = \frac{A}{(1-v)} + \frac{B}{(1-v)^2} + \frac{C}{(1-v)^3} + \frac{D}{(v-h)} \quad P.3.$$

$$\begin{aligned} \text{or } bv^3+cv^2+dv+f &= A(1-v)^2(v-h)+B(1-v)(v-h)+C(v-h)+D(1-v)^3 \\ &= A [v^3-(h+2)v^2+(2h+1)v-h] \\ &\quad +B [-v^2+(h+1)v-h] +C [v-h] \\ &\quad +D [-v^3+3v^2-3v+1] \end{aligned}$$

equating coefficients,

$$v^3 : \quad b = A - D$$

$$v^2 : \quad c = (h + 2) A - B + 3D$$

$$v : \quad d = (2h + 1) A + (h+1)B + C - 3D$$

$$v^0 : \quad f = - hA - hB - hC + D$$

Solution of these four simultaneous equations gives,

$$A = \frac{(3b+c)h^2 + (d-3b)h + (b+f)}{(1-h)^3} \quad P.4.$$

$$B = \frac{h(3b+2c+d) - 2b - c + f}{(1-h)^2} \quad P.5.$$

$$C = \frac{b+c+d+f}{(1-h)} \quad P.6.$$

$$D = \frac{(3b+c)h^2 + (d-3b)h + (b+f)}{(1-h)^3} - b \quad P.7.$$

Since the above expressions are independant of v, it is possible to integrate equation P.3 in its simple form,

$$\int_{ge}^{g+h} \frac{bv^3 + cv^2 + dv + f}{(v-h)(1-v)^3} dv = \left[-A \log_e(1-v) + \frac{B}{(1-v)} + \frac{C}{2(1-v)^2} + D \log_e(v-h) \right]_{ge}^{g+h} \quad \text{P.8.}$$

where the constants A,B,C and D are given by equations, P.4, P5, P6 and P.7 respectively.

Equation P.1 may now be equated to the sum of the components of the integral from equations P.2 and P.8, thus,

$$\int_{L_I}^L f(S) dl = \frac{1}{k} \left\{ \frac{a(6h^2 - 8h + 3)}{(1-h)^3} \log_e \frac{(v-h)}{(1-v)} + a \left[\frac{h(9-8v) - (11-10v)}{2(1-h)^2(1-v)^2} \right] + [D - a(h+3)] \log_e(v-h) - (v-h) - A \log_e(1-v) + \frac{B}{(1-v)} + \frac{C}{2(1-v)^2} \right\}_{ge}^{g+h} \quad \text{P.9.}$$

```

430 PRINT "CONSTANT SATURATION, 0.25"
440 INPUT L1
450 PRINT "INLET SATURATION, 0.1"
460 INPUT S1
470 PRINT "OUTLET SATURATION, 0.1"
480 INPUT S2
490 PRINT "SUPERFICIAL VELOCITY, 1"
500 INPUT U
510 READ D1, D2, D3
520 L1=L1+D1
530 L2=L2+D2
540 L3=L3+D3
550 PRINT "PRINT "LAYER FACTOR" "
560 PRINT "PRINT "NUMBER OF "
570 PRINT "LAYERS "
580 L1=L1+D1
590 L2=L2+D2
600 L3=L3+D3
610 L4=L4+D4
620 L5=L5+D5
630 L6=L6+D6
640 L7=L7+D7
650 L8=L8+D8
660 L9=L9+D9
670 L10=L10+D10
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6790 L622=L622+D622
6800 L623=L623+D623
6810 L624=L624+D624
6820 L625=L625+D625
6830 L626=L626+D626
6840 L627=L627+D627
6850 L628=L628+D628
6860 L629=L629+D629
6870 L630=L630+D630
6880 L631=L631+D631
6890 L632=L632+D632
6900 L633=L633+D633
6910 L634=L634+D634
6920 L635=L635+D635
6930 L636=L636+D636
6940 L637=L637+D637
6950 L638=L638+D638
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6970 L640=L640+D640
6980 L641=L641+D641
6990 L642=L642+D642
7000 L643=L643+D643
7010 L644=L644+D644
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7250 L668=L668+D668
7260 L669=L669+D669
7270 L670=L670+D670
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7470 L690=L690+D690
7480 L691=L691+D691
7490 L692=L692+D692
7500 L693=L693+D693
7510 L694=L694+D694
7520 L695=L695+D6
```

```

420  REM          PREDICTION OF TWO PHASE PRESSURE DROP RATIO
440  PRINT "SLOPE OF SATURATION PLOT, B"
450  INPUT B
460  PRINT "CONSTANT SATURATION LENGTH, L1"
465  INPUT L1
466  PRINT "INLET SATURATION, S1"
470  INPUT S9
480  PRINT "OUTLET SATURATION, SE"
490  INPUT S5
491  PRINT "SUPERFICIAL VELOCITY, U"
492  INPUT U
500  READ D1, D3, E1
503  L1=L1*2*D3
504  B=B*2*D3
505  K=1/(B/(S9-S5)-L1)
507  PRINT : PRINT "DECAY FACTOR=", K
508  PRINT : PRINT "NUMBER OF          PRESSURE"
509  PRINT "  LAYERS          DROP RATIO"
510  X1=4/D3
520  X2=6/D1
530  X3=1-E1
540  X4=X2*E1+2
550  X5=E1*(X1+X2*X3)
560  X6=X1*X3
570  Y1=X4+2
580  Y2=2*X4*X5
590  Y3=2*X4*X6+X5+2
600  Y4=2*X5*X6
610  Y5=X6+2
620  Y6=S9-S5
630  Y7=S5
640  Z1=((3*Y2+Y3)*Y7+2+(Y4-3*Y2)*Y7+Y2+Y5)/(1-Y7)+3
650  Z2=((3*Y2+2*Y3+Y4)*Y7-2*Y2-Y3+Y5)/(1-Y7)+2
660  Z3=(Y2+Y3+Y4+Y5)/(1-Y7)
670  Z4=Z1-Y2
680  W1=Y1*(6*Y7+2-8*Y7+3)/(1-Y7)+3
690  W2=Z4-Y1*(Y7+3)
695  M=1
699  C=10
700  FOR J=1, 120
710  I=1
720  L=J*2*D3
730  V=Y6+Y7
740  W3=Y1*(Y7*(9-8*V)-(11-10*V))/(2*(1-Y7)+2*(1-V)+2)
745  I1=W1*LOG((V-Y7)/(1-V))+W2*LOG(V-Y7)+W3-(V-Y7)
747  I1=I1-Z1*LOG(1-V)+Z2/(1-V)+Z3/(2*(1-V)+2)
760  IF I=2 THEN 800
770  I2=I1
775  W4=EXP(-K*(L-L1))
776  IF W4>.1E-06 THEN 789
777  W4=.1E-06
778  IF M>1 THEN 789
779  PRINT : PRINT "EXPONENTIAL DECAYED AFTER"
780  PRINT J, "LAYERS": PRINT : M=2
789  V=Y6*W4+Y7
790  I=I+1: GOTO 740
800  P2=(I2-I1)/K
810  P1=L*(Y1*S9+4+Y2*S9+3+Y3*S9+2+Y4*S9+Y5)/(1-S9)+3
820  P=D3+2*(P1+P2)/(16*X3+2*L)
825  P=P*1.724
830  IF J=C THEN PRINT J, P: C=C+10
840  NEXT J
900  STOP : END
1000  DATA .25E-04, .305E-04, .6995

```

Q.2 PROGRAM OUTPUT FOR
 CONSTANT INLET SATURATION
 VALUE ($S_I = 0.55$)

NAME OF LABORATORY PROJECT
 DATE OF LABORATORY WORK
 NAME OF STUDENT
 COURSE NUMBER
 NAME OF INSTRUCTOR
 NAME OF INSTITUTION
 ADDRESS
 CITY
 STATE
 ZIP CODE
 PHONE NUMBER
 FAX NUMBER
 E-MAIL ADDRESS
 TITLE
 OBJECTIVE
 THEORY
 PROCEDURE
 RESULTS AND DISCUSSION
 CONCLUSION
 REFERENCES
 APPENDICES
 GRADE

SLOPE OF SATURATION PLOT, B
 !1.1412
 CONSTANT SATURATION LENGTH, L1
 !!
 INLET SATURATION, SI
 !.55
 OUTLET SATURATION, SE
 !.226
 SUPERFICIAL VELOCITY, U
 !.005

DECAY FACTOR= 6499.6

NUMBER OF LAYERS	PRESSURE DROP RATIO
10	42.943
20	28.6378
30	23.8936
40	21.5573

EXponential DECAYED AFTER LAYERS

50	17.7491
60	14.791
70	12.678
80	11.0932
90	9.86063
100	8.87457
110	8.06779
120	7.39547

SLOPE OF SATURATION PLOT, B
 !1.1789
 CONSTANT SATURATION LENGTH, L1
 !!
 INLET SATURATION, SI
 !.55
 OUTLET SATURATION, SE
 !.175
 SUPERFICIAL VELOCITY, U
 !.015

DECAY FACTOR= 7647.15

NUMBER OF LAYERS	PRESSURE DROP RATIO
10	36.1991
20	22.6341
30	18.1178

EXponential DECAYED AFTER LAYERS

40	14.9717
50	11.9774
60	9.98115
70	8.55527
80	7.48586
90	6.6541
100	5.98869
110	5.44426
120	4.99058

SLOPE OF SATURATION PLOT, B
 !1.2071
 CONSTANT SATURATION LENGTH, L1
 !!
 INLET SATURATION, SI
 !.55
 OUTLET SATURATION, SE
 !.138
 SUPERFICIAL VELOCITY, U
 !.025

DECAY FACTOR= 8494.65

NUMBER OF LAYERS	PRESSURE DROP RATIO
10	32.9566
20	19.7172
30	15.3254

EXponential DECAYED AFTER LAYERS

40	11.9292
50	9.54334
60	7.95278
70	6.81667
80	5.96459
90	5.30185
100	4.77167
110	4.33788
120	3.97639

SLOPE OF SATURATION PLOT, B
 !1.8660
 CONSTANT SATURATION LENGTH, L1
 !1
 INLET SATURATION, SI
 !.55
 OUTLET SATURATION, SE
 !.092
 SUPERFICIAL VELOCITY, U
 !.07

DECAY FACTOR= 5332.52

NUMBER OF LAYERS	PRESSURE DROP RATIO
10	28.7656
20	16.3743
30	12.3269
40	10.3058
50	9.07069

EXPONENTIAL LAYERS	DECAYED AFTER LAYERS
51	
60	7.62142
70	6.53264
80	5.71607
90	5.08095
100	4.57285
110	4.15714
120	3.81071

SLOPE OF SATURATION PLOT, B
 !1.4445
 CONSTANT SATURATION LENGTH, L1
 !1
 INLET SATURATION, SI
 !.55
 OUTLET SATURATION, SE
 !.114
 SUPERFICIAL VELOCITY, U
 !.035

DECAY FACTOR= 7087.3

NUMBER OF LAYERS	PRESSURE DROP RATIO
10	30.9148
20	18.0268
30	13.7549

EXPONENTIAL LAYERS	DECAYED AFTER LAYERS
39	
40	11.3877
50	9.11015
60	7.59179
70	6.50725
80	5.69384
90	5.0612
100	4.55508
110	4.14098
120	3.7959

Q.3. PROGRAM OUTPUT FOR
FITTED VALUES OF INLET
SATURATION

SLOPE OF SATURATION PLOT, B
 !1.1412
 CONSTANT SATURATION LENGTH, L1
 !!
 INLET SATURATION, SI
 !.615
 OUTLET SATURATION, SE
 !.226
 SUPERFICIAL VELOCITY, U
 !.005

DE CAY FACTOR= 8477.87

NUMBER OF LAYERS	PRESSURE DROP RATIO
10	74.471
20	44.4235
30	34.4139

EXponential DECAYED AFTER LAYERS

40	26.7751
50	21.4201
60	17.85
70	15.3
80	13.3875
90	11.9
100	10.71
110	9.73639
120	8.92502

SLOPE OF SATURATION PLOT, B
 !1.1789
 CONSTANT SATURATION LENGTH, L1
 !!
 INLET SATURATION, SI
 !.485
 OUTLET SATURATION, SE
 !.175
 SUPERFICIAL VELOCITY, U
 !.015

DE CAY FACTOR= 5848.74

NUMBER OF LAYERS	PRESSURE DROP RATIO
10	21.2777
20	15.135
30	13.1258
40	12.1515

EXponential DECAYED AFTER LAYERS

50	10.8434
60	9.0362
70	7.74531
80	6.77715
90	6.02413
100	5.42172
110	4.92883
120	4.5181

SLOPE OF SATURATION PLOT, B
 !1.2071
 CONSTANT SATURATION LENGTH, L1
 !!
 INLET SATURATION, SI
 !.485
 OUTLET SATURATION, SE
 !.138
 SUPERFICIAL VELOCITY, U
 !.025

DE CAY FACTOR= 6613.79

NUMBER OF LAYERS	PRESSURE DROP RATIO
10	18.2279
20	12.3262
30	10.3832
40	9.35209

EXponential DECAYED AFTER LAYERS

50	7.70521
60	6.42101
70	5.50372
80	4.81575
90	4.28067
100	3.8526
110	3.50237
120	3.2105

SLOPE OF SATURATION PLOT, B
 !1.8660
 CONSTANT SATURATION LENGTH, L1
 !!
 INLET SATURATION, SI
 !.415
 OUTLET SATURATION, SE
 !.092
 SUPERFICIAL VELOCITY, U
 !.07

DECAY FACTOR= 3431.68

NUMBER OF LAYERS	PRESSURE DROP RATIO
10	6.06266
20	4.7741
30	4.56703
40	4.48294
50	4.43434
60	4.40152
70	4.36857

EXponential DECAYED AFTER LAYERS	78
80	4.27632
90	3.80117
100	3.42106
110	3.11005
120	2.85088

SLOPE OF SATURATION PLOT, B
 !1.4445
 CONSTANT SATURATION LENGTH, L1
 !!
 INLET SATURATION, SI
 !.475
 OUTLET SATURATION, SE
 !.114
 SUPERFICIAL VELOCITY, U
 !.035

DECAY FACTOR= 5461.96

NUMBER OF LAYERS	PRESSURE DROP RATIO
10	14.6054
20	9.81421
30	8.27781
40	7.51191

EXponential DECAYED AFTER LAYERS	50
50	7.04636
60	5.87197
70	5.03311
80	4.40397
90	3.91464
100	3.52318
110	3.20289
120	2.93598

Symbols have the following meanings except where specifically indicated in the text:

- a - specific surface area (m^2/m^3)
- a' - surface area per unit volume of screen (m^2/m^3)
- a_c - collector radius (m)
- a_p - drop radius (m)
- A - flow parameter (-)
- B - mesh thickness (m)
- C - fractional concentration of dispersion (-)
- d_a - aperture diameter (m)
- d_c - collector diameter (m)
- d_{ce} - effective collector diameter (m)
- d_p - drop diameter (m)
- d_{pc} - critical drop diameter for release (m)
- d_{pe} - exit drop diameter (m)
- \mathcal{D} - molecular diffusion coefficient (m^2/s)
- D - diameter of coalescing bed (m)
- e - voidage fraction (-)
- f - friction factor (-)
- f(H) - universal hydrodynamic function (-)
- f_{ret} - electromagnetic retardation correction factor (-)
- F_{Ad} - adhesion force (N)
- F_{DL} - double layer force (N)
- F* - dimensionless drag force (-)
- g - acceleration due to gravity = 9.81 (m/s^2)
- h - drop/collector separation (m)
- H - dimensionless separation between drop and collector = $\frac{h}{a_p}$ (-)
- H* - dimensionless separation at rear stagnation point

- I - mass capture rate (m^3/s)
- k - saturation profile parameter characterising rate of decay (-)
- k' - Boltzman's constant = $1.38048 \times 10^{-23} (J/K)$
- K - Kozeny constant (-)
- ℓ - distance into coalescing bed from inlet face (m)
- ℓ_{max} - film resolution (lines/mm)
- L - bed depth (m)
- L_I - length of bed near inlet face having constant saturation value (m)
- L_q - length of queue (-)
- m_o - hologram recording magnification (-)
- m_l - hologram reconstruction magnification (-)
- M - number of service channels in queueing system (-)
- n - number of units in queueing system (-)
- N_{Ad} - adhesion number = $\frac{4Q}{9\pi N_R^2} \frac{1}{\mu_c u d_p^2}$ (-)
- N_c - mesh count (m^{-1})
- N_{DL} - double layer group = $\frac{K d_p}{2} (-)$
- N_{E2} - electrokinetic group = $\frac{2\zeta_p \zeta_c}{(\zeta_p^2 + \zeta_c^2)} (-)$
- N_G - gravity number = $\frac{d_p^2 (\rho_d - \rho_c) g}{18 \mu_c u} (-)$
- N_L - number of mesh layers (-)
- N_{Pe} - Peclet number $\frac{d_c u}{D} (-)$
- N_R - interception number = $\frac{d_p}{d_c} (-)$
- N_{Rd} - direct interception number = $\frac{d_p}{d_a} (-)$
- N_{Re} - Reynolds number = $\frac{d_c u \rho_c}{\mu_c} (-)$
- N_{Re''} - modified Reynolds number = $\frac{d_c u \rho_c}{\mu_c (1-e_1)} (-)$
- N_{Sh} - Sherwood number = $\frac{I}{\pi d_c D C} (-)$
- N_{Stk} - Stokes number = $\frac{d_p^2 \rho_c u}{9 \mu_c d_c} (-)$
- P_n - probability that n units present in queueing system (-)

ΔP	- pressure drop (N/m^2)
Q	- Hamaker constant (J)
r	- mesh geometric parameter = $\frac{d_c}{d_c + d_a}$ (-)
r_{22}	- intermolecular distance (m)
S	- local saturation of dispersed phase (-)
\bar{S}	- average saturation in bed (-)
S_I	- saturation near inlet face of bed (-)
S_E	- saturation near exit face of bed (-)
t	- time (s)
T	- tortuosity factor (-)
u	- superficial velocity (m/s)
u_I	- interstitial velocity (m/s)
u_o	- object distance from sample to photographic plate (m)
v_o	- distance from spatial filter to photographic plate (m)

Greek Letters

α	- 'customer impatience' factor (-)
β	- ratio of number of collisions to number of coalescence events (-)
γ	- interfacial tension (N/m)
ϵ	- dielectric constant of continuous phase
σ	- surface tension (N/m)
η	- drop capture efficiency (-)
η_c	- coalescence efficiency (-)
μ	- viscosity (Ns/m^2)
μ	- service rate in queueing system (s^{-1})
ρ	- density (kg/m^3)
ρ	- utilisation factor in queueing system = $\frac{\lambda}{M\mu}$ (-)
λ	- arrival rate in queueing system (s^{-1})

- λ_c - theoretical filter coefficient (m^{-1})
- λ_e - experimental filter coefficient (m^{-1})
- λ_o - wavelength of hologram recording illumination
(m)
- λ_1 - wavelength of hologram reconstruction
illumination (m)
- θ_e - equilibrium contact angle (rad.)
- θ - coalescence time (s)
- θ_p - polar co-ordinate for drop/collector system
(rad.)

Subscripts

- 1 - single phase flow
- 2 - two phase flow
- c - continuous phase
- d - dispersed phase
- D - diffusion
- DI - direct interception
- G - gravity (sedimentation)
- I - interception
- II - inertial impaction
- L - London forces
- T - total

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